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NASA CR-135140



QUIET CLEAN SHORT-HAUL EXPERIMENTAL ENGINE (QCSEE)

Whirl Test of Cam/Harmonic Pitch Change Actuation System

by

Aircraft Systems Department, Propulsion Project Group

Hamilton Standard
Division of United Technologies Corporation

Under Subcontract to General Electric Company (P.O. 200-4XX-14G38570)

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Prepared For

National Aeronautics and Space Administration

NASA Lewis Research Center Contract NAS3-18021



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1.0 SUMMARY

The Hamilton Standard blade pitch change actuation system for the Quiet Clean Short Haul Experimental Engine (QCSEE) was tested on a whirl rig, at Hamilton Standard, between November 10, 1975 and February 16, 1976.

The objectives of the test were to verify that the unit satisfied the design requirements and was structurally adequate for use in an engine test.

The testing included evaluation of the travel limit switch, blade angle position accuracy, performance, frequency response, and endurance.

During tests of the travel limit switch it was found that the no-back output shaft was under-designed. Analysis of the shaft revealed that it could not be adequately strengthened within the space available. In order to reduce the load on the no-back, a snubber was installed in the actuator. When run with the snubber, no further distress was noted on the output shaft.

The testing showed that:

- Blade overtravel after actuation of the travel limit switch at maximum pitch change rate and zero fan speed was within the calculated value of 6.5-7.0 degrees.
- Blade angle can be positioned within 0.25 degrees when moving from open to close at zero fan speed and within 1.5 degrees at high fan speeds.
- There is approximately 1 degree of hysteresis in the system when reversing the direction of blade angle change at zero fan speed.
- The minimum attainable blade angle change was 0.17 degrees toward open and 0.26 degrees toward close.
- The maximum pitch change rate attained during a blade angle change was 135 degrees/second.
- Frequency response without the snubber indicated reasonable correlation with predictions at frequencies up to 1 Hertz where the magnitude ratio was lower and the phase shift was higher than at frequencies above 1 Hertz.

- Frequency response with the snubber again indicated reasonable correlation with predictions at frequencies up to 1 Hertz with excitation magnitudes of ± 8 ma, however with ± 4 ma excitation the magnitude ratio was down and there was considerable phase shift.
- The actuation system successfully completed 500 simulated mission endurance cycles at pitch change rates up to 75 degrees/second and 50 cycles at pitch change rates up to 135 degrees/second.

2.0 INTRODUCTION

During the proposal effort for the Quiet Clean Shorthaul Experimental Engine (QCSEE), a design study of ten prospective fan pitch change systems was conducted. The results of this study were reported in HSPC 74A14 QCSEE Variable Pitch Fan System Proposal. On the basis of this effort, four systems were selected for further study.

The detail study of the four systems was conducted under General Electric Purchase Order 200-4XX-14G31376. The results of the study were presented in SP 08A74 QCSEE Variable Pitch Fan Pitch Change System. The study showed that a system incorporating a remotely mounted Beta Regulator, driving a harmonic drive/cam actuator through a flexible drive shaft, was the most attractive. This type of system was designed, manufactured, and tested under NASA Contract NAS 3-18021. The results of the design effort were reported in NASA CR-134852 which also includes a summary of the preliminary design studies done on the other systems.

A second actuator was also developed by the General Electric Company. Details of the design of the General Electric actuator are presented in NASA CR-134873 "QCSEE Ball Spline Pitch Change Mechanism Design Report".

This report describes the whirl rig testing conducted on the system. The object of the test was to determine the operating characteristics of the actuator, verify that it satisfies the design requirements, and assure it's structural adequacy for use in an engine test.

3.0 SYSTEM DESCRIPTION

The variable pitch actuator system is shown in Figure 1. An electrical input command signal from the engine digital control to the electro-hydraulic servo valve (EHV) directs high pressure oil to motors in the beta regulator. This provides rotary mechanical input to the actuator differential gear train through a flexible drive shaft. Rotary motion is then transmitted through a no-back, harmonic drive, rotating cam, and cam follower arms to the blade trunnions. Since there is a fixed mechanical relationship between hydraulic motor rotation and blade angle, two linear variable differential transformers (LVDT) driven by motor output provide redundant blade angle feedback signals to the digital control to close the control loop and null the input signal when the blades reach the commanded position.

The overall gear ratio from the blades to the drive shaft is 1005:1 with most of the ratio (201:1) provided by a harmonic drive. This permits the low-torque power transmission elements between the beta regulator and the harmonic drive to be designed for low weight and improved blade angle accuracy.

The mechanical pitch change power and blade angle feedback functions are provided by the beta regulator module which is remotely mounted in a readily accessible area of the engine cowling. A simplified schematic of the beta regulator is shown in Figure 2. A blade angle change command from the engine control to the EHV mounted in the accessory section, causes movement of the servo valve to direct supply oil to either the open or close pitch ports of the hydraulic motor. The motor output drives the flexible shaft to change blade angle and drives two LVDT's through a worm gear and screw to provide an electrically redundant blade position feedback. Electrical limit switches are provided to cut off the command signal to the EHV if a blade angle is commanded beyond the maximum operating range. Pressure relief valves across the motor ports limit motor pressure to 3000 psi during rapid accelerations of the acutator system.

The rotary output of the beta regulator is transmitted to the actuator differential gear train through a flexible drive shaft passing through the engine reduction gearing. The shaft core is encased in a flexible teflon lined casing supported in a rigid conduit mounted on the engine reduction gear support. Continuous engine lubrication oil flow is directed through the casing from the beta regulator to lubricate the core and the actuator components.

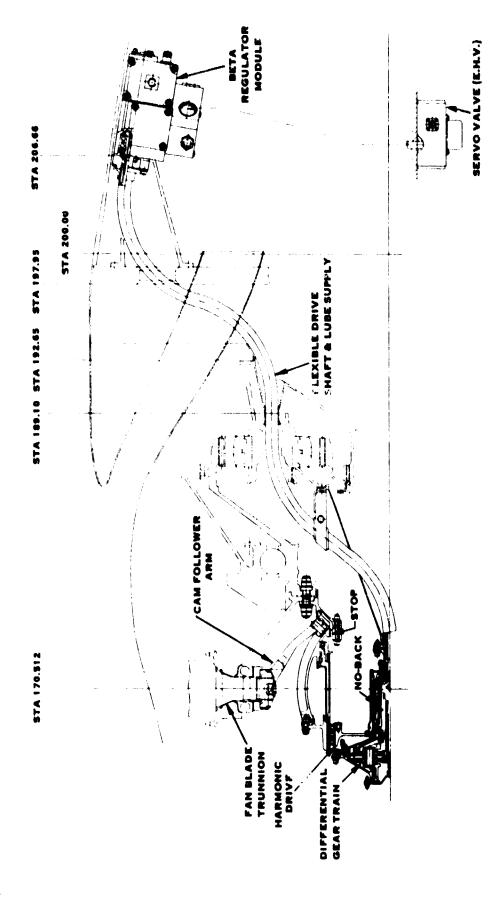


FIGURE 1. QCSEE VARIABLE PITCH ACTUATOR SYSTEM

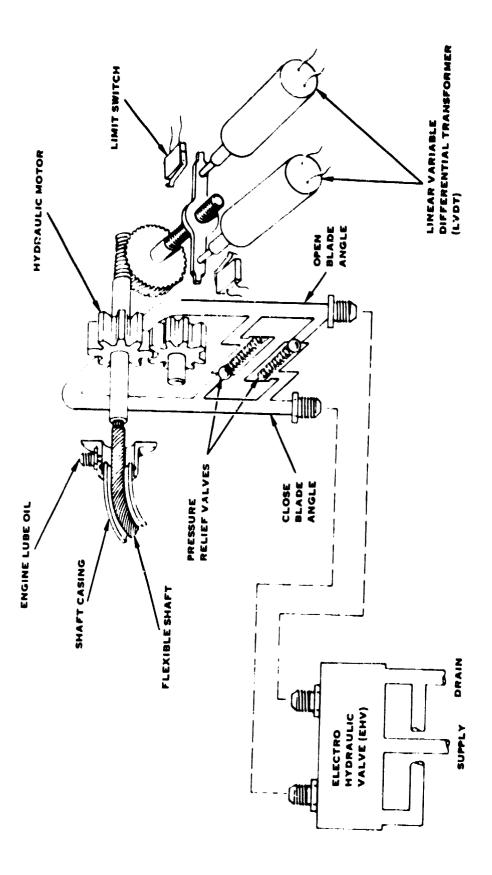


FIGURE 2. BETA REGULATOR SCHEMATIC

A planetary differential gear train is utilized to cross the rotating boundary of the fan. The differential gearing is a conventional 5:1 ratio phase difference type with a grounded sun gear, an input sun gear driven by the flexible shaft, three pairs of planet gears on a bearing supported cage, a reference speed ring gear fixed to the fan disk and an output ring gear driving the noback input. With no pitch change input, the output ring gear relates at fan speed. Rotation of the input sun gear during pitch change causes the output ring gear to either advance or recede with respect to the fan speed. This change in output is the input to the no-back. The gears and bearings are lubricated by oil directed outward centrifugally from the sun gear shafts.

A bi-directional spring clutch or "no-back" is provided between the differential gearing and the harmonic drive to maintain the set blade angle position in the absence of a pitch change command. This device consists of a self-energizing steel spring which is in contact with the inner surface of a fixed housing, the input and output shafts and the necessary couplings and bearings. When holding a fixed blade angle, the blade loads are transmitted to ground (housing) through the spring. When the input acts against opposing blade loads (raising the load), the spring slides in the housing and does not react to any blade loads. When the input acts against aiding blade loads (lowering the load), the input releases the spring at the commanded pitch ..te and the blade load energy is dissipated in frictional heat between the spring and housing. Due to the short duty cycle and the thermal mass of the parts, the total heat rise is low in the no-back. Lubrication oil flows continually through the no-back and is supplied centrifugally from the sun gear shafts.

No-back housing torque is reacted by a disk-type torque limiter brake. The no-back is a high-gain locking device capable of locking more than a million in-lbs of torque at high friction coefficients, and the torque limiter limits maximum no-back torque to ground to acceptable structural limits during rapid pitch change decelerations. The brake disks are lubricated by oil supplied centrifugally from the differential gearing.

The harmonic drive provides the primary gear reduction for the mechanical actuator and increases the input torque to the level required to change pitch. Four basic elements are incorporated in this high-ratio (201:1) mechanical transmission rated at 50,000 in-lbs output. They are: a three-lobed harmonic-shaped wave generator input plug which provides the harmonic the harmonic lobe shape to the flexible spline, a triplex split inner race ball bearing set for high radial stiffness, a flexible spline (flex spline) to convert from

the harmonic lobe shape to a grounded circular shape with minimum frictional losses and a stiff circular output spline which drives the blade pitch cam.

The thin-race ball bearings are pressed on the three-lobed wave generator plug and assume the three-lobe harmonic shape. Spline teeth on the outside diameter of the flex spline mesh with spline teeth on the inside diameter of the circular spline at the three lobe locations. Circular splines on the other end of the flex spline ground it to the fan disk. Due to a 3-tooth difference in number of teeth between the circular spline and flex spline (603-600=3), one revolution of the wave generator input rotates the circular spline output 3/603 or 1/201 of a revolution.

Lubrication oil for the harmonic bearings and splines is supplied centrifugally from the differential gearing and no-back.

The cam and follower arms convert output rotation of the harmonic drive to fan blade angle change. Titanium follower arms, splined and clamped to the blade trunnions, engage individual cam slots in the spherical cam surface through cam rollers to synchronize the blades and sum the blade torques. The radial axis defined by the roller and cam track centerlines always intersects the fan axis of rotation at the same point similar to the apex point of a bevel goar mesh.

Cam support is provided by a preloaded duplex bearing set mounted on a support ring attached to the fan disk mounting flange for accurate balance control. Lubrication oil from the harmonic drive lubes the bearing set and is returned centrifugally to the engine scavenge area. A single dynamic oil seal with centrifugal venting procludes a dynamic pressure head.

Fixed mechanical stop lugs between the cam and cam support ring restrict the blades to 7° overtravel at each end of the maximum operating range.

4.0 RIG DESCRIPTION

Figure 3 is a drawing showing the arrangement of the whirl rig and the actuator.

The whirl rig used for the test was a modification of an existing rig. The rig utilizes a 186.4 kilowatts (250 horsepower) electric motor to drive the fan through an eddy current clutch and a speed increasing gearbox. The connecting shaft between the gearbox and the actuator has a flange which dimensionally duplicates the actuator/disk mounting surface in the engine. The flange also provides a path for the removal of the lubrication fluid.

A disk together with trunnions, stub blades (counterweights), and other retention hardware was provided for the test by General Electric. The stub blades are designed to apply the same centrifugal load to the blade retention and the actuator as the actual blade, and to approximate the twisting moment of the actual blade. Figure 4 is a curve of twisting moment versus blade angle for the actual and the stub blade. Stub blades were used for the test as they do not produce thrust and therefore can be driven by a relatively small motor.

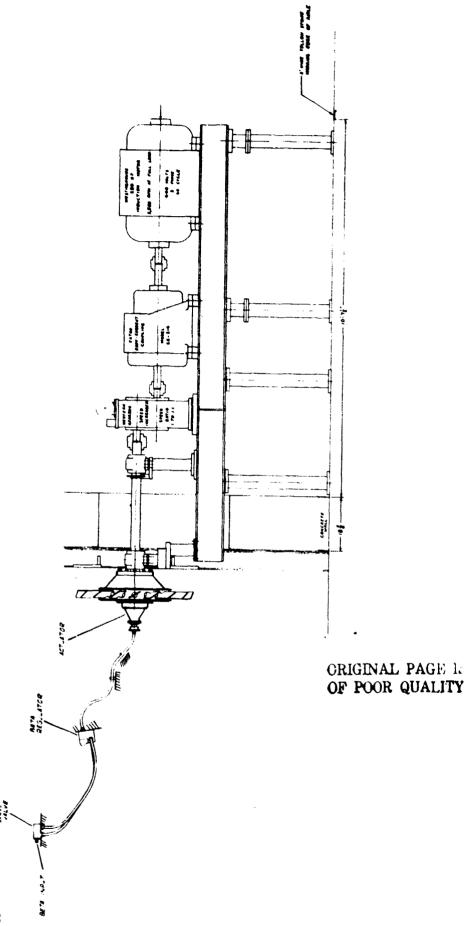
Initial attempts to run the rig revealed the need to provide a shroud to enclose the stub blades and their retention to reduce the windage losses and consequently the power required to drive the assembly and the noise level in the vicinity of the test rig.

In the engine, the Beta Regulator is mounted behind the actuator and the flex shaft is routed through the engine gearbox to the actuator. To duplicate this arrangement on the whirl rig would require a hollow connecting shaft between the speed increaser gear box and the actuator, a hollow shaft through the gear box, and a quill shaft between the flex cable and the pitch change input at the actuator. This approach was investigated and discarded as it added another spring rate to the pitch change input system (the quill shaft), as well as being expensive and time consuming to obtain a hollow shaft gearbox.

Instead, a set of test hardware which provided a front input to the actuator for the flex cable was designed and manufactured. This hardware does not affect the spring rate of the input system. Figure 5 is an assembly drawing of the front input hardware.

In an engine installation, the lubricating and pitch change fluids are supplied by an engine driven pump. For the whirl test, a Viking pump rated at 18.9 liters/min (5 gpm) at 68.9 newtons/cm² (100 psi) was used to supply the





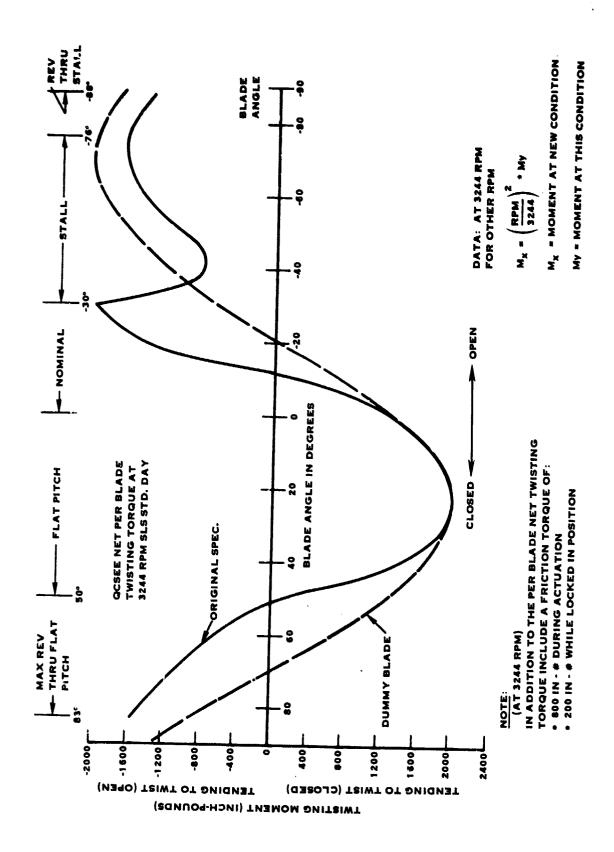


FIGURE 4. MAX NET TWISTING MOMENT

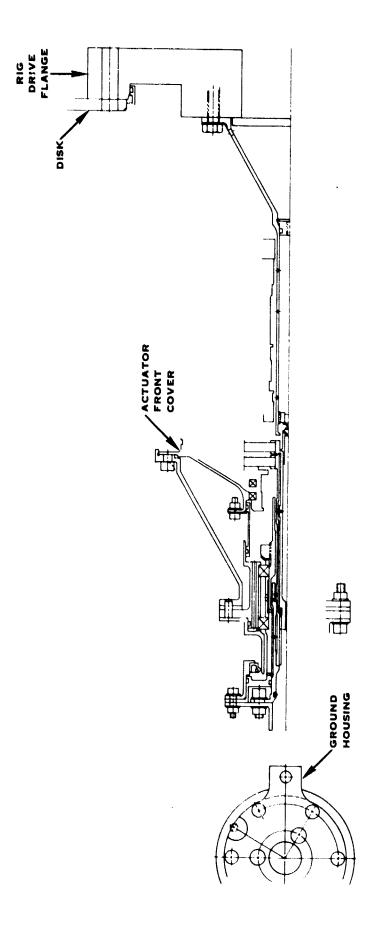


FIGURE 5. ASSEMBLY, FRONT INPUT HARDWARE

4.0 (Continued)

lubricating fluid. Initially, it was planned to use one Denison pump rated at 132.5 liters/min (35 gpm) at 3447.4 newtons/cm² (5000 psi) to supply change fluid for the majority of the testing, and to use a Denison pump rated at 94.6 liters/min in parallel with it for testing requiring high flow. (maximum pitch change rates). Operating the two pumps proved to be quite difficult, so the second pump was replaced by an accumulator.

The instrumentation provided for the test is listed in Table I. The flex shaft speed was measured by machining a six tooth wheel on the feedback shaft in the beta regulator, and installing a magnetic pickup in the beta regulator housing.

The flex shaft torque was measured by strain gaging and calibrating the ground sun gear in the actuator.

Actual blade angle was measured using photo diode sensors to measure the relative position of the actuator cam and the disk. The output of the sensors was read by a phase meter.

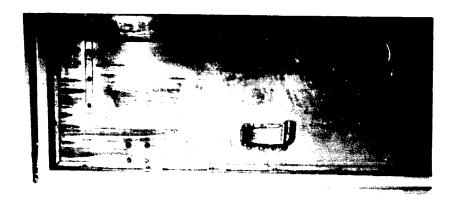
Figure 6 is a photograph of the whirl rig installation.

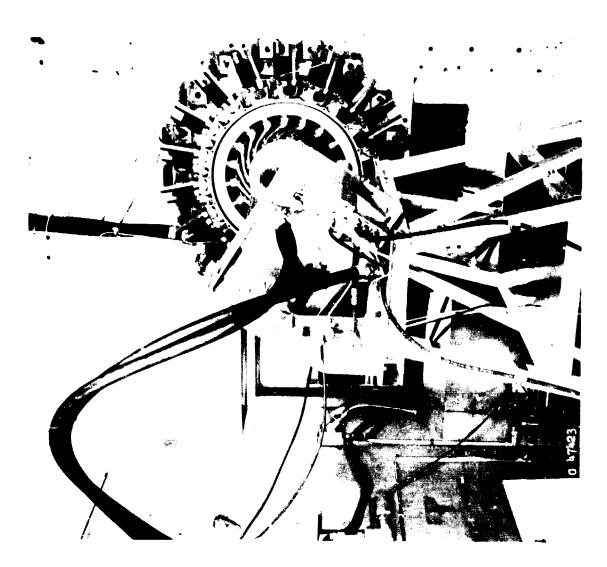
A closed loop variable gain control system, modified from an existing unit, was used to operate the actuator.

The rig was operated in accordance with Operating Procedure 222PT-37. A copy of this procedure is included in Appendix A.

TABLE I INSTRUMENTATION LIST

Measurement	Range	Accuracy
EHV Supply Pressure	0 - 4000 paig	± 40 psi
EHV Current Signal	± 100 ma	\pm 2.25% of reading
Flow to EHV	0 - 45 gpm	± 2.83% of full scale
ΔP Across-Motor	0 - 3500 psig	± 2.25% of full scale
Temperatures	0 - 300°F	± 2°F
Blade Angle Command	+20 to -120°	± 2% of full scale
LVDT Feedback Voltage	± 5 V de	± 2% of full scale
Flex Shaft Speed	0 - 24,000 rpm	± 2.83% of full scale
Flex Shaft Torque	0 - 200 in lb	± 2.83% of full scale
Lube Oil Flow	0 - 1 qt/min	± 2% of reading
Lube Oil Pressure	0 - 100 psig	± 1 psi
Fan Speed	0 - 3700 rpm	± 2% of full scale
Vibration - Horizontal	0 - 30 mils	± 5% of reading
Vibration - Vertical	0 - 30 mils	± 5% of reading
Fan Blade Angle	+20 to -120°	± 0.2°





5.0 TEST PROCEDURE

The test was conducted essentially in accordance with Plan of Test 222PT-31 Rev. A. A copy of this plan is included in Appendix B. Variations from the Plan of Test are described below at the appropriate point in the test procedure description.

The test points in the plan are defined in terms of fan blade angle to be compatible with the specification. However, the stub blades supplied by General Electric were offset 25° relative to the blade angle in order to properly locate the peak twisting moment loads on the actuation system and a portion of the test data was recorded in terms of counterweight angles. The data is marked blade angle or counterweight angle, as appropriate, and the relationship between the two is that blade angle plus 25° equals counterweight angle. For the test, the counterweights were reindexed 2 spline teeth (18.5 degrees) from the design condition. This was done at the direction of General Electric to obtain more reverse blade angle.

The lubrication flow check was conducted with the flex shaft disconnected from the actuator. The flow from the shaft was measured at varying inlet pressures. Inlet pressure was measured at the pump.

The test was later repeated measuring the pressure at the inlet to the regulator and measuring the flow at the inlet to the flex shaft.

The LVDT null and calibration test was accomplished by changing blade angle with the manual pitch change input and mer auring the counterweight angle. The blades were moved from "close" to "open" only.

The travel limit switch tests were accomplished by setting the controller to give a step command of approximately 30-40 degrees of blade angle which would actuate the travel limit switch. The pitch change rate was varied by regulating servo valve supply pressure and flow. The amount of overtravel was determined by physical measurement of the angle at which the counterweights stopped, compared with the angle at which the switch actuated. The testing was conducted at 0 fan rpm.

During early testing it was found that operation at the maximum pitch change rate overstressed the no-back output shaft. In order to continue testing, the maximum pitch change rate was limited to 75 degrees/second.

Initial running revealed a rig resonance at approximately 2500 fan rpm. To avoid this resonance, all testing specified at 2500 rpm was conducted at 2700 rpm.

The position accuracy test was conducted by setting blade angle with the controller and measuring actual blade angle. Because of the instrumentation accuracies in this test (controller setting, actual blade angle readout) an alternate static positioning accuracy test was also run. In this test, the blade angle feedback (LVDT's) were positioned with the manual input and the actual blade angle change was measured by a dial indicator and converted to angle change.

The test to determine the pressure and flow required to start and sustain actuator motion was conducted by commanding the desired change with no pressure to the servo valve, and then increasing the pressure to the valve until the blade angle change was complete.

The test to determine the minimum blade angle change around 0° was conducted by slowly changing the controller input until a change in LVDT feedback voltage was observed. The actual counterweight position was measured to determine that the blades had moved.

Frequency response testing was conducted with a two channel transfer function analyzer with automatic gain and sweep control and direct plot output. For the static testing, blade angle was measured by a proximity pickup set up on one of the stub blades.

The endurance test was conducted in accordance with Table II. The forward thrust test points were set manually, the modulating thrust points were run using an oscillator input to the controller, and the reverse/unreverse transients were step inputs to the controller. Initial cycles were conducted at approximately six per hour to preclude overheating the no-back. Since experience showed no evidence of heating on the no-back spring, the remaining cycles were conducted at a rate of ten to twelve per hour.

Following testing in accordance with Plan of Test 222PT-31 Rev. A, the actuator was modified by the addition of a snubber. Testing in accordance with Plan of Test 222PT-38 was then accomplished. A copy of this plan is included in Appendix B.

TABLE II QCSEE ACTUATOR

Speed	Blade	DC Command	Feedback	Step Co	mmand
(rpm)	Angle	Pot Setting	Gain	Set	Cond
2700 (#3)	12°	309	325	9.81	ON
700	-3°	205	†	†	†
408 (#1)	-3°	205			
408	0	230			
068 (#2)	0	230			
068	+7°	269*			
†	+5°	†			
	+7°				
1	+9°				
068	+7°	269*			
068	+7°	272			ON
068	-100°	272			OFF
408 (#1)	-100°	272	{		OFF
700 (#3)	-100°	272			OFF
700	+ 7 °	272	¥	•	ON
700	·r12°	301	325	9.81	ON

*Plug in Function Gen. - Amp - Min Freq - Range 0.1 Dial 1.9

Don't Forget to go to 2700 Prior to Unreverse.

≈ 10 min/cycle

These tests were conducted in the same manner as the tests of Plan of Test 222PT-31 Rev. A with the exception of the frequency response test. This test was conducted with a single channel transfer function analyzer. This unit did not have automatic gain and sweep control, or direct plot capability. The following test points were run rather than those listed in the plan of test.

Frequency (cps)	Servo Valve Current (ma)		
0.5	±4 ±8		
1.0	± 4 ± 8		
2.0	± 4 ± 8 ± 12		
3.0	± 4 ± 8 ± 16		

6.0 TEST RESULTS

Structural

During the test the following structural problems were noted with the hard-ware.

During the initial tests of the travel limit switches, after six actuations at pitch change rates up to 82 degrees/second, it was found that the no-back output shaft had fractured at the webs between the three windows. Figure 7 is a photograph of the shaft. The fracture of the shaft was attributed to the shaft being under-designed in this area for the expected load of 271.1 newton meters (2400 inch pounds). The shaft was repaired by electron beam welding, and "beefed" up in the area where it had fractured. In addition a new shaft was fabricated incorporating additional strengthening. The welded shaft (763494-1/222X575) was approximately twice as strong and the new shaft (763494-1 Chg. B) was four times as strong as the design which fractured. Figure 8 is a comparison of the three configurations.

Testing was resumed with the welded output shaft. Performance tests, including some running at high pitch change rates, and a total of 46 flight cycles were completed. The unit was disassembled for a routine inspection, and the welded output shaft was found to be fractured in two of the three webs.

Analysis of this incident revealed that the original design of the shaft did not take into account the torque generated by inertia of the no-back drum and brake hardware which must be accelerated up to speed when the actuator starts slowing down.

At pitch change rates of 135 degrees per second, this inertia raises the load on the shaft to 536.6 newton meters (4750 inch pounds). Further analysis revealed that even with the inertia reduced as much as feasible in the existing design, the output shaft could not be strengthened sufficiently within the space available. Consequently, testing with the new shaft was limited to a pitch change rate of 75 degrees per second.

After completion of 500 flight cycles, the new shaft (763494-1 Chg. B) was found to be cracked in the web area. Investigation of this incident including a static torque test of the system, revealed that the spring rate of the system was 5648.7 newton meters per radian (50,000 inch pounds per radian) rather than the calculated 2824.4 newton meters per radian (25000 inch pounds per radian). The higher spring rate raises the expected 271.1 newton meter load on the shaft at 75°/second pitch change rate, to 451.9 newton meters (4000 inch pounds).

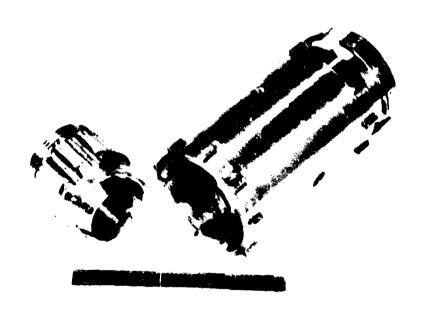
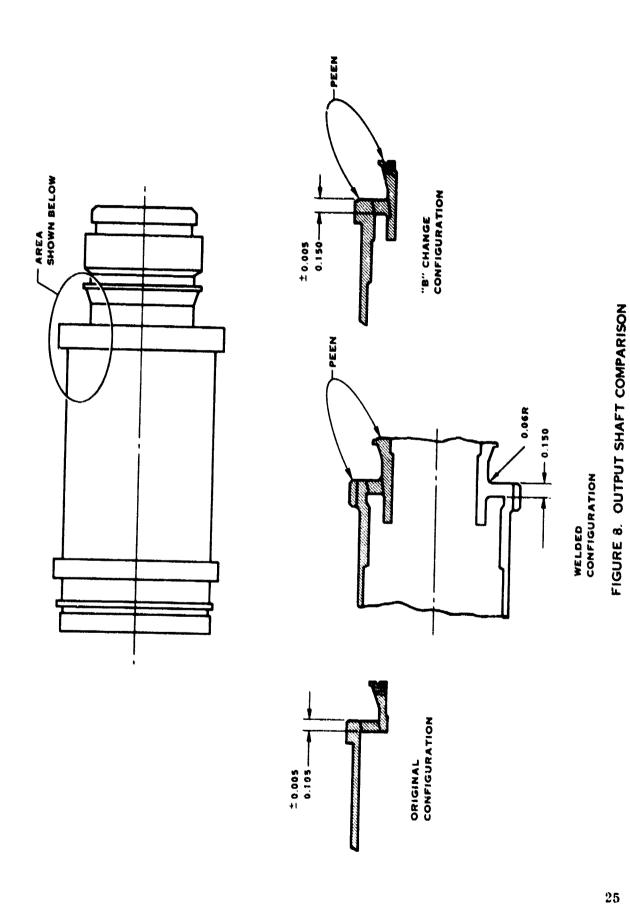


FIGURE 7. 763494-1 OUTPUT SHAFT

ORIGINAL PAGE 15 OF POOR QUALITY



The difference between the calculated and actual system spring rates is the result of the coefficient of friction assumed between the no-back spring and the no-back drum. In the calculations, a lower than actual coefficient was used. This results in more engaged spring coils to absorb a given torque resulting in a greater spring deflection, and therefore a lower no-back spring rate.

To restore the actuators maximum pitch change rate capability, a snubber was designed and fabricated to reduce the spring rate of the system to 677.8 newton meters per radian (6000 inch pounds per radian), which reduced the output shaft load to 271.1 newton meters at the maximum pitch change rate of 135°/second.

Incorporation of the snubber results in the following life predictions for the output shaft and the snubber:

Maximum Flex Shaft Speed (rpm)	17,500	21,000
Snubber Fatigue		
Life (cycles)		
at room temperature	> 1,000	750
at 93.3°C (200°F)	1,000	500
Output Shaft Fatigue		
Life (cycles)		
at room temperature	5 x 10 ⁴	1.5×10^4
at 93.3°C (200°F)	2.1×10^{5}	1 x 10 ⁵

At the same time that the snubber was installed in the actuator, the no-back spring was replaced with a new part. This was done as a precautionary measure as the original spring had been subjected to higher than design loads during operation without a snubber.

Prior to, and following the testing of Plan of Test 222PT-38, the spring rate of the snubber was measured. The initial measurement revealed the actual spring rate to agree closely with the calculated rate (684.4 newton meters per radian [6067 inch pounds per radian] vs. 698.2 newton meters per radian [6180 inch pounds per radian]). The after testing check revealed no change in spring rate or other deterioration of the snubber.

During the initial spring rate check of the snubber, its deflection versus load was calibrated. A means of indicating relative motion during actuator operation was added to the snubber, and the deflection checked at various pitch change rates up to maximum. These measurements indicated that the snubber limited the load on the output shaft to the design value of 293.7 newton meters (2600 in lbs).

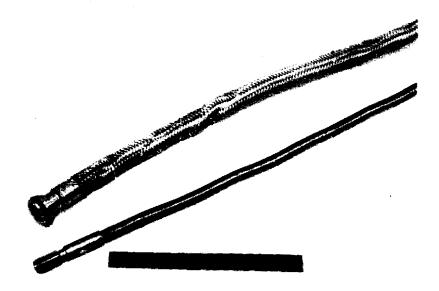
It was observed during the test that the wear pattern on the cam tracks was high in the track on one side, and low on the other. Inspection of the cam revealed that the sides of the tracks were not parallel to a radius through the center of the track. Subsequent running over the full range of speeds and loads has shown that this small abnormality in the pattern does not represent a significant problem and no corrective action was taken.

As a result of an assembly error, the rear support housing was fractured in the 'O' seal groove area that mates to the engine. This fracture extended about 3 inches circumferentially at the aft edge of the seal groove. The housing was repaired by removing the damaged area, electron beam welding a new ring onto the back of the housing, and remachining.

During the lubrication flow check it was found that oil leaked through the outer casing of the flex shaft at the junction with the end fittings. This was found to be a result of a drawing error which did not call for the teflon lining in the casing to be swaged between the casing and the end fittings. The second flex cable leaked slightly at the junction of one end fitting and the casing. This was attributed to a poor joint at the swage of the casing to the end fitting. For rig running, the conduit was sealed at the actuator and the drain hole plugged to contain the leakage.

At the completion of testing, the flex shaft core was deformed, apparently as a result of having been over-torqued. The distortion of the core during the over-torque also deformed the casing. Reference Figure 9. The over-torque was attributed to a shutdown during the travel limit switch tests from a pitch change rate of 130 degrees per second with a controller time constant of 10 milliseconds. The controller was modified to a time constant of 25 milliseconds, (system designed for 20-30) and subsequent shaft torques were measured to be within acceptable limits.

During the inspection following the completion of testing per 222PT-31 Rev. A, pitting and scoring were noted on the bevel gears and their pinion. The data received with the bevel gear set from the manufacturer had shown the final grind pattern to be marginal.



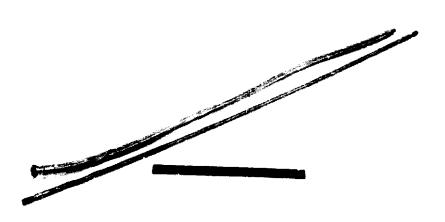


FIGURE 9. 763402-1 FLEXIBLE SHAFT

During the course of the test, the gear pattern had been examined several times and always appeared to be high on the tooth. Consequently, the shimming of the pinion and gears was changed several times to bring them closer into mesh. As the parts are brought closer into mesh the backlash is reduced. The lubrication system for the beta regulator was designed to provide a mist to lubricate the gear mesh. As the backlash was reduced, this method of lubrication apparently became marginal. The gear set was replaced, and the regulator reworked to provide an oil jet which sprays directly into the gear mesh. The mesh was shimmed to insure sufficient backlash rather than attempting to center the pattern on the tooth.

During the disassembly of one of the hydraulic motors for inspection, the bearing housing was damaged by a faulty disassembly procedure. This motor was replaced by a new motor. Following the completion of the flight cycles conducted with the snubber, an attempt was made to conduct the frequency response test. With an input to the servo valve of ± 4 ma, no response was obtained from the actuator. The hydraulic motors were disassembled for inspection, and the new motor was found to exhibit heavy wear on the housing bore at the drive gear face. Dimensional inspection revealed no reason for this wear. Another motor was installed in the regulator, and the actuator then responsed to current inputs to the servo valve of ± 4 ma.

All other hardware in the actuator and the regulator was in good condition following the tests. The hardware was subjected to magnaflux, zyglo, and visual inspection following the completion of testing per Plan of Test 222PT-31 Rev. A, and visual inspection following testing per Plan of Test 222PT-38.

Performance

The lubrication flow check revealed that it took 63.8 newtons per square centimeter (92.5 pounds per square inch) at the lubrication pump to obtain a flow of 0.80 liters per minute (0.85 quarts per minute) out of the flex shaft. Figure 10 is a plot of the data taken during this test. A second check was made to determine the pressure required at the inlet to the regulator to obtain the required flow. This test showed that it took 42 newtons per square centimeter (61 pounds per square inch) to achieve the desired flow. The design value was 48 newtons per square centimeter (70 pounds per square inch). Figure 11 is a plot of the data taken during this test.

The results of the LVDT calibration tests showed the output voltages from the two LVDT's to be in close agreement. Table III is a summary of the data taken during the test where the blades were moved from "close" to "open", and Figures 12 and 13 are plots of output voltage versus blade angle.

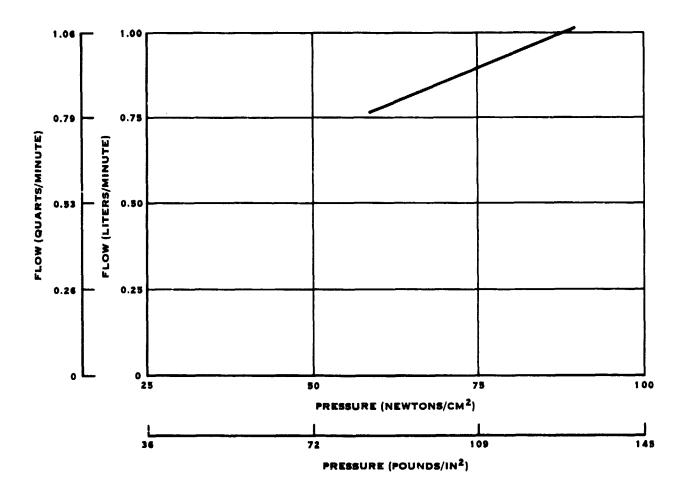


FIGURE 10. LUBRICATION FLOW CHECK FLOW VS. PRESSURE AT PUMP 11-10-75

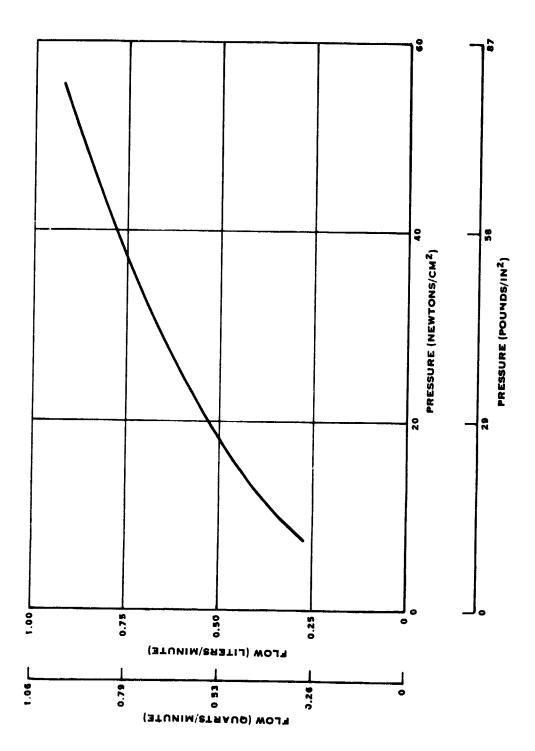


FIGURE 11. LUBRICATION FLOW CHECK - FLOW VS. PRESSURE AT REGULATOR 1-14-76

TABLE III LVDT CALIBRATION

DATA FROM 11-10-75

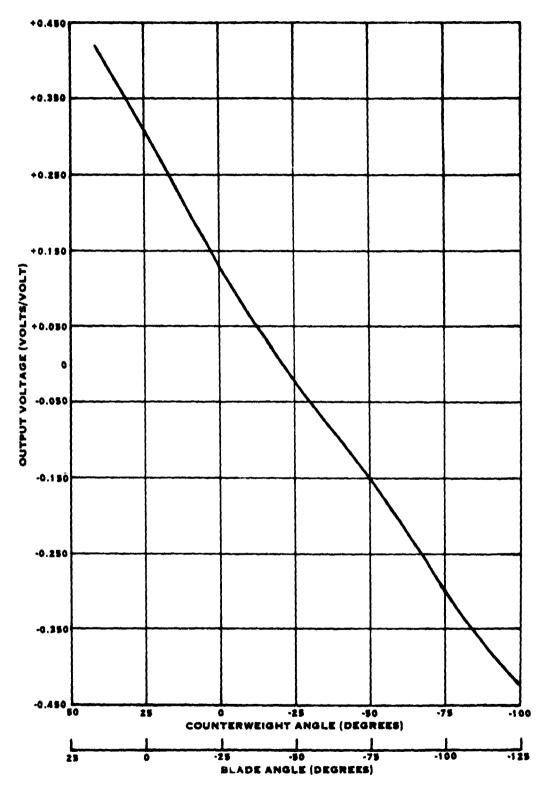
Counterweight	Blade	Excitation	-	Voltage tal		Voltage s/Volt
Angle	Angle	Voltage	LVDT1	LVDT2	LVDT1	LVDT2
*42.2	17.2	5.666	2.373	2.371	0.419	0.418
35.0	10.0	5.665	2.107	2.105	0.372	0.372
30.0	5.0	5.667	1.907	1.905	0.337	0.336
25.0	0	5.664	1.715	1.711	0.303	0.302
20.0	- 5.0	5.666	1.513	1.509	0.267	0.266
15.0	-10.0	5.667	1.307	1.305	0.231	0.230
10.0	-15.0	5.665	1.094	1.091	0.193	0.193
5.0	-20.0	5.666	0.893	0.890	0.158	0.157
0	-25.0	5.665	0.706	0.704	0. 125	0.124
- 5.0	-30.0	5.665	0.518	0.517	0.091	0.091
-10.0	-35.0	5.666	0.345	0.342	0.061	0.060
-15.0	-40.0	5.665	0.184	0.183	0.032	0.032
-20.0	-45.0	5.664	0.034	0.033	0.006	0.006
-25.0	-50.0	5.665	-0.121	-0.121	-0.021	-0.021
-30.0	-55.0	5.665	-0.265	-0.265	-0.047	-0.047
-35.0	-60.0	5.665	-0.425	-0.425	-0.075	-0.075
-40.0	-65.0	5.665	-0.575	-0.574	-0.102	-0.101
-45.0	-70.0	5.665	-0.725	-0.725	-0. 128	-0.128
-50.0	-75.0	5.665	-0.875	-0. 873	-0.154	-0.154
-55.0	-80.0	5.665	-1.027	-1.027	-0.181	-0.181
-60.0	-85.0	5.665	-1.190	-1.187	-0.210	-0.210
-65.0	-90.0	5,665	-1.353	-1.351	-0.239	-0.238

TABLE III (Continued) LVDT CALIBRATION

DATA FROM 11-10-75

Counterweight	Blade	Excitation	Output To	Voltage tal	•	Voltage s/Volt
Angle	Angle	Voltage	LVDT1	LVDT2	LVDT1	LVDT2
-70.0	-95.0	5.665	-1.520	-1.514	-0.268	-0.267
-75.0	-100.0	5.665	-1.686	-1.683	-0.298	-0.297
-80.0	-105.0	5.665	-1.850	-1.847	-0.327	-0.326
-85.0	-110.0	5.665	-2.007	-2.003	-0.354	-0.354
-90.0	-115.0	5.665	-2.153	-2.150	-0.380	-0.380
-95.0	-120.0	5.665	-2.290	-2.284	-0.404	-0.403
*-98. 7	-123.7	5.665	-2.382	-2.378	-0.420	-0.420

^{*}Mechanical Stop



FIGIJRE 12. OUTPUT VOLTAGE VS. BLADE ANGLE LVDT #1 11-10-75 DATA TAKEN FROM CLOSE TO OPEN

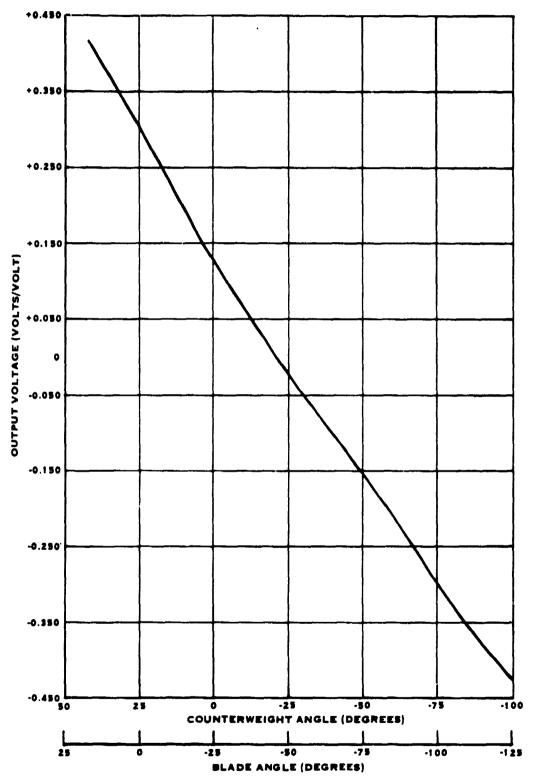


FIGURE 13. OUTPUT VOLTAGE VS. BLADE ANGLE LVDT #2 11-10-75
DATA TAKEN FROM CLOSE TO OPEN

6.0 (Continued)

The data available from the travel limit switch tests indicates that the blade overtravel after a switch actuation at maximum pitch change rate was within the calculated value (6.5°-7.0°), when the data is adjusted for the higher pitch change rate and the effect of the motor back pressure is taken into account.

Table IV is a summary of the data taken during the tests. The data shows that the overtravel and the flex shaft torque are greater at the open end of travel than at the close end. The reason for this has not been determined. Overtravel and flex shaft torque are a function of the pitch change rate, the servo valve time constant, and the drag torque of the torque limiter brake. The higher the pitch change rate, the greater the overtravel and the higher the shaft torque. The greater the valve time constant, the greater the overtravel and the lower the shaft torque. The higher the drag of the torque limiter, the less the overtravel and the lower the shaft torque. Figure 14 is a curve of calculated overtravels and torques versus valve time constants.

In setting the torque limiter brake drag, variations amounting to 1 n-m (10 in. lb) of flex shaft torque from one direction to the other occurs. This does not account for the differences noted in the test. It is felt that the difference is a result of different dynamics in the no-back operation from one direction to the other. Sanborn traces of a travel limit switch stop at both the open and close ends of travel are included in Appendix C.

The rotating blade angle position accuracy test showed a maximum deviation of 1.5° between the sct (LVDT) blade angle and the angle determined by the photo diode system. Tables V, VI, and VII are summaries of the data. The deviation noted is within that expected based on estimates of the mechanical and hydraulic hysteresis, the accuracy of the LVDT calibration, and the accuracy of the photo diode system. The three points which show the photo diode sensor angle to be closer to open than the LVDT angle are considered to be bad data points and should be ignored.

The static blade angle position accuracy test showed that the correlation between the set and resultant blade angles were good. In the close direction over a 4° range from 2° open to 2° close the resultant angle deviated from the set angle by a total of 0.25°. This deviation is within the accuracy of the LVDT calibration.

The test also showed that there is slightly less than 1 degree of hysteresis in the system when reversing the direction of blade angle change. This is somewhat greater than the expected value of 0.79 degrees. The expected value consists of a mechanical error of 0.48 degrees, a feedback backlash

HSER 7002

TABLE IV TRAVEL LIMIT SWITCH

DATA FROM 2-13-75

Step Blade Angle Command

Time Constant = 25 milliseconds

Fan Speed = 0 rpm

Blade Angle Travel Prior to Stop - 30 degrees

Actuator with Snubber

Switch	Setting	Beta	Beta	Pitch Change	Peak	Shaft
Open	Close	Stop	Overtravel	Rate	Tore	que
(degrees)	(degrees)	(degree.s)	(degrees)	(degrees/sec)	(in-lbs)	(N-m)
-115.1		-119.0	3.9	101.2	260	29.4
		-120.5	5.4	106.5	300	33.9
		-121.1	6.0	116.2	340	38.4
	+10.2	+12.3	2.1	103.2	230	26.0
		+15.5	5.3	123.8	250	28.2
		+15.6	5.4	123.8	250	28.2

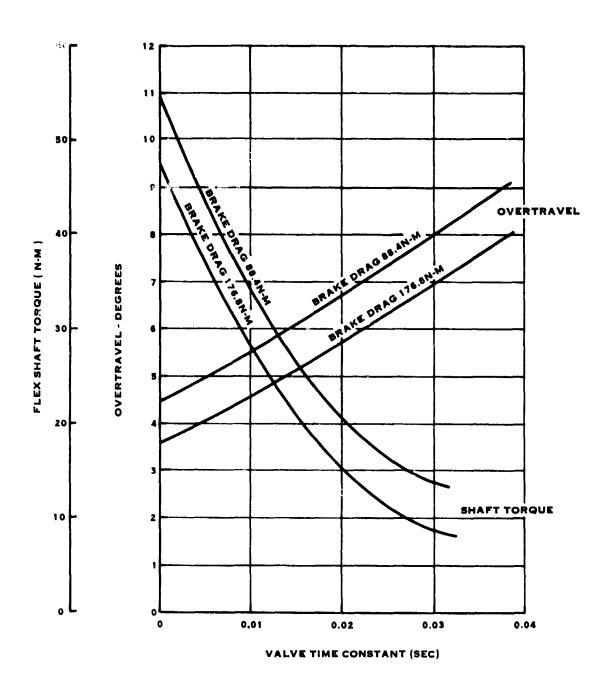


FIGURE 14. CURVE - OVERTRAVEL & SHAFT TORQUE VS VALVE TIME CONSTANT

TABLE V

ANGLE POSITION ACCURACY (ROTATING)

DATA FROM 12-30-75

	Photo Diode Blade Angle (degrees)	Fan Speed (rpm)
	+ 13.0	2700
	- 2.5	3068
	+ 0.8	3068
	+ 8.0	3068
	+ 4.0	3068
	+ 7.5	3068
	+ 9.5	3068
	+ 7.5	3068
00	-100.5	3068
12.0	+ 13.0	3068

TABLE VI BLADE ANGLE POSITION ACCURACY (ROTATING)

DATA FROM 12-30-75

LVDT Blade Angle (degrees)	Photo Diode Blade Angle (degrees)	Fan Speed (rpm)
+12.0	+13.0	2700
- 3.0	- 2.2	2700
0	+ 1.0	2700
+ 7.0	+ 8.5	2700
+ 5.0	+ 6.0	2700
+ 7.0	+ 8.0	2700
+ 9.0	+10.0	2700
+ 7.0	+ 8.0	2700
- 100	-99.5	2700
+12.0	+13.5	2700

TABLE VII

BLADE ANGLE POSITION ACCURACY

(ROTATING)

DATA FROM 12-31-75

LVDT Blade Angle (degrees)	Photo Diode Blade Angle (degrees)	Fan Speed (rpm)
+12.0	+13.0	3068
+ 9.0	+10.5	3068
+ 7.0	+ 8.0	3068
+ 5.0	+ 6.0	3068
0	+ 0.5	3068
- 3.0	- 2.0	3068
- 100	-99. 5	3068
- 3.0	- 2.5	3068
0	- 1.0	3068
+ 5.0	+ 5.5	3068
+ 7.0	+ 8.0	3068
+ 9.0	+10.0	3068
+12.0	+13.0	3068

6.0 (Continued)

of 0.07 degrees, a backlash of the trunnion roller to the cam track of 0.14 degrees, and a calibration inaccuracy of 0.10 degrees. The reason for the difference between the measured and expected value is not known, but could be the result of larger than designed mechanical errors or backlash.

Figure 15 is a sketch of the test setup, Table VIII is a summary of the data taken during the test, and Figure 16 is a plot of the data. It should be noted that this is the maximum hysteresis and would only occur at full reverse during normal operation. During forward thrust operation, the actuator is always loaded towards close.

The results of testing to determine the pressure required to start and sustain blade motion are listed in Table IX. In those cases where more than one set of data is given for an excursion, the blades moved a few degrees, stopped, and then moved again as pressure was raised. The data given as "To Sustain" should have been taken when the blades were moving at a constant rate. As most of the excursions were very short, this was not the case. Even in the excursions to and from reverse, the motor flow could not be restricted enough for a constant pitch change rate to be established. The data given for sustaining was taken at a point where the pressure was constant, or almost constant, for a short period of time. The pressures required to start and sustain motion was well within the 3000 psi available for all cases tested. Sanborn traces of this test are included in Appendix C.

Actuator efficiency was determined at the input to the differential gear train where the actual torque on the ground sun gear was measured. Efficiencies were calculated based on data obtained during transient operation of the pitch change actuator. Since these were not steady state pitch change rates, the possibility of errors in the absolute numbers are greater and the results should therefore be used in a manner which reflects this possibility. The efficiencies determined were higher than anticipated, and were in the 70% range. The following tabulation summaries the measured efficiencies.

Blade Angle (deg)	Direction of Motion	Fan Speed (rpm)	Efficiency (%)
-110	Open	3000	76
-100	Close	3000	73
~ 66	Close	2500	71
- 50	Close	2350	7 5
+ 10	Open	2500	73

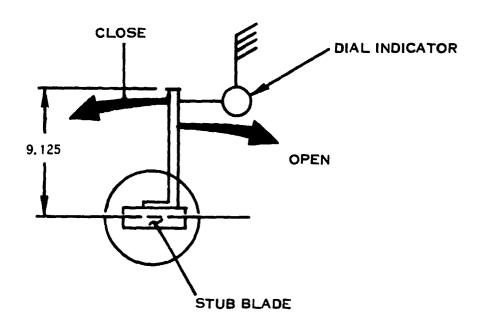


FIGURE 15. TEST SETUP BLADE ANGLE POSITIONING ACCURACY (STATIC)

TABLE VIII

BLADE ANGLE POSITION ACCURACY

STATIC

DATA FROM 2-14-76

BASED ON LVDT #1 CALIBRATION 2-5-76

LVDT Reading (volts)	Set Blade Angle (degrees)	Dial Indicator Reading (inches)	Δ Dial Indicator Reading (inches)	Δ Blade Angle (degrees)	Resultant Blade Angle (degrees)
1.670	-2.900	0.758	0	0	-2.000
1.678	-1.787	0.730	0.028	0.176	-1.824
1.686	-1.574	0.690	0.040	0.250	-1.574
1.693	-1.387	0.666	0.024	0.151	-1.423
1.700	-1.200	0.638	0.028	0.176	-1.247
1.708	-0.987	0.612	0.026	0.163	-1.084
1.715	-0.800	0.582	0.030	0.188	-0.896
1.723	-0.587	0.548	0.034	0.213	-0.683
1.730	-0.400	0.522	0.026	0.163	-0.520
1.738	-0.187	0.486	0.036	0.226	-0.294
1.745	0	0.461	0.025	0.157	-0.137
1.753	+0.213	0.426	0.035	0.220	+0.083
1.760	+0.400	0.400	0.026	0.163	+0.246
1.768	+0.613	0.369	0.031	0.195	+0.441
1.776	+0.826	0.335	0.034	0.213	+0.654
1.784	+1.039	0.306	0.029	0.182	+0.836
1.791	+1.226	0.280	0.026	0.163	+0.999
1.798	+1.413	0.253	0.027	0.170	+1.169

TABLE VIII (Continued)

LVDT Reading (volts)	Set Blade Angle (degrees)	Dial Indicator Reading (inches)	Δ Dial Indicator Reading (inches)	Δ Blade Angle (degrees)	Resultant Blade Angle (degrees)
1.805	+1.600	0.225	0.028	0.176	+1.345
1.813	+1.813	0.189	0.036	0.226	+1.571
1.820	+2.000	0.160	0.029	0.182	+1.753
1.813	+1.813	0.160	0	0	+1.753
1.805	+1.600	0.160	0	0	+1.753
1.798	+1.413	0.160	0	0	+1.753
1.791	+1.226	0.160	0	0	+1.753
1.784	+1.039	0.158	0.002	0.013	+1.740
1.776	+0.826	0.183	0.025	0.157	+1.583
1.768	+0.613	0.213	0.030	0.188	+1.395
1.760	+0.400	0.245	0.032	0.201	+1.194
1.753	+0.213	0.276	0.031	0.195	+0.999
1.745	0	0.306	0.030	0.188	+0.811
1.738	-0.187	0.332	0.026	0.163	+0.648
1.730	-0.400	0.365	0.033	0.207	+0.441
1.723	-0.587	0.390	0.025	0.157	+0.284
1.715	-0.800	0.426	0.036	0.226	+0.058
1.708	-0.987	0.457	0.031	0.195	-0.137
1.700	-1.200	0.488	0.031	0.195	-0.332
1.693	-1.387	0.515	0.027	0.170	-0.502
1.686	-1.574	0.540	0.025	0.157	-0.659
1,678	-1.787	0.573	0.033	0.207	-0.866
1.670	-2.000	0.606	0.033	0.207	-1.073

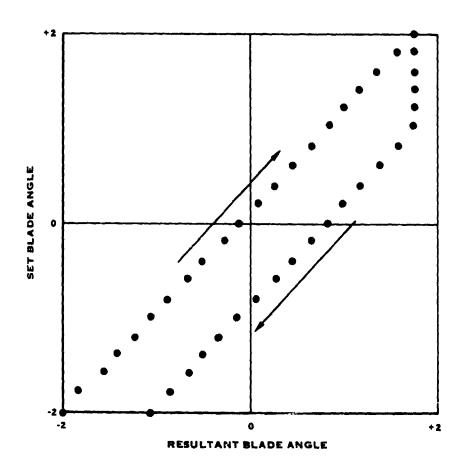


FIGURE 16. BLADE ANGLE POSITION ACCURACY DATA FROM 2-14-76 SET BLADE ANGLE VS. RESULTANT BLADE ANGLE

TABLE IX. PERFORMANCE
Data from 12-31-75

ָרָ קיין			Pressure to	ire to			Torone to	4		Pitch
Angle		S M	Start Motion	Su	Sustain Motion	Start	-	Sustain	ain is:	Change Rate
(degrees)	(rpm)	(bisd)	(newtons/cm ²)	(psig)	(newtons/cm ²)	(dl-ni).	(newton meter)	(in 1b)	(newton meter)	(max) (de grees / sec)
+12 to -3	2700	950	655.0	1000	689. 4	88	9 6			
(+10.5)	2700	1100	758.4	1100	758. 4	3 %	i c	4	20 (3.75
-3 to 0	3408	400	275.8	250	172. 4	3 4	, c	0 4 0	4. 6	22. 5
0 to +7	3068	450	310.2	250	172.4	٠ ٦	; ;	٥ ;	0. 7	11.25
+7 to +5	3068	1600	1103.1	1000	689, 4	' ç	ດ ເ ວັນ	01 9	1.1	18.75
+5 to +7	3068	400	275.8	200	137 9	3 5	o (7	4.7	7.5
+7 to +9	3068	300	206.8		197.0	,	0.5	4	0.5	3.75
+9 to +7	3068	1000	689.4	1000	6.761	4 6	0.5	9	0.7	11.25
	3068	1500	1034.2	1200	897.9	3 8		40	4. 5	5.25
+7 to -100	3068	1600	1103.1	400	275.8	ę <i>g</i>	ဆို (မ	09	8.8	11.25
-100 to +7	2700	1000	689. 4	1000	689.4	3 ~	9 0	4 2	o.	75.0
	2700	1600	1103.1	1000	689. 4	42	. 4.	8	ກ ຕ	22. 5 67 5

6.0 (Continued)

The average pitch change rate for a blade angle excursion from -3 degrees to -100 degrees at 3315 fan rpm with a EHV supply pressure of 2378.6 \pm 34.4 newtons per square centimeter (3450 + 50 psig) was 116 degrees per second. The maximum rate attained was 135 degrees per second. A Sanborn trace of this test is included in Appendix C. The design objective was 135 degrees per second maximum.

The minimum blade angle change which could be achieved around 0 degrees with a fan speed of 3408 rpm and a EHV supply pressure of 2378.6 \pm 34.4 newtons per square centimeter (3450 + 50 psig) was 0.17 degrees in the open direction and 0.26 degrees in the close direction based on LVDT motion. The design objective was 0.5 degrees.

With the blade angle set at -100 degrees, EHV supply pressure at 0, and the fan speed increased from 0 to 3578 rpm, no blade angle change occurred based on LVDT readout indicating that the no back mechanism did not slip.

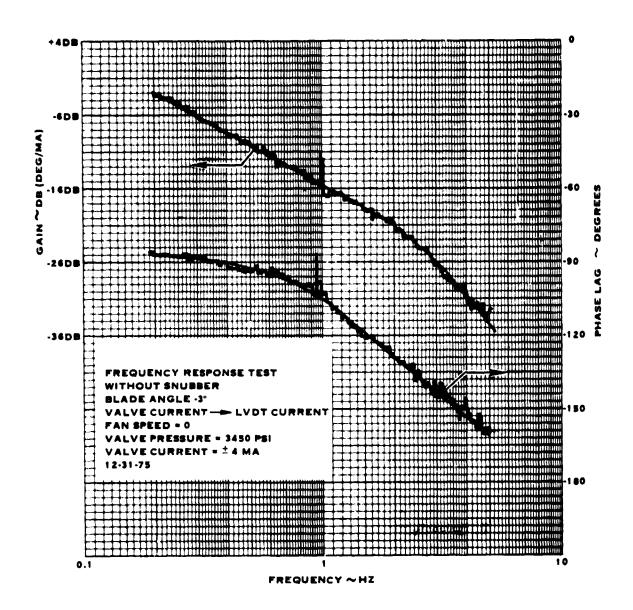
With the blade angle set at 0 degrees, EHV supply pressure at 0, and fan speed increased from 0 to 3408 rpm, no blade angle change occurred based on LVDT readout indicating that the no back mechanism did not slip.

The results of the frequency response testing are presented in Figures 17 through 22. The data in Figures 17 through 20 is for the actuator without a snubber while the data in Figures 21 and 22 is for the actuator with a snubber. In general, the actuator without the snubber indicated reasonable correlation with the predicted performance in the frequency range up to 1 Hertz; magnitude ratio was lower and phase shift was higher than at frequencies above 1 Hertz. The actuator with the snubber indicated reasonable correlation in the frequency range up to 1 Hertz when excitation magnitudes were \pm 8 ma. However for \pm 4 ma excitation the magnitude ratio was down and there was considerable phase shift.

Possible causes of the deviation between the test results and the design intent are high internal leakage in the hydraulic motors, higher than anticipated friction values, and lags due to the snubber.

A total of 500 endurance cycles at pitch change rates up to 75 degrees/second, were conducted on the actuator prior to installation of the snubber, and an additional 50 cycles at pitch change rates up to 135 degrees/second after snubber installation. Sanborn traces of a typical cycle are included in Appendix C.

Copies of the log sheets generated during the test are included in Appendix D, and a test chronology is included in Appendix E.



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FIGURE 17. QCSEE FREQUENCY RESPONSE

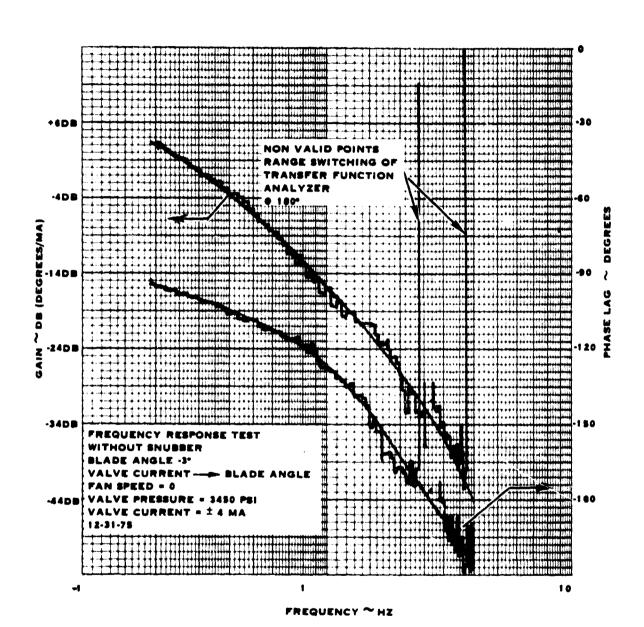


FIGURE 18. QCSEE FREQUENCY RESPONSE

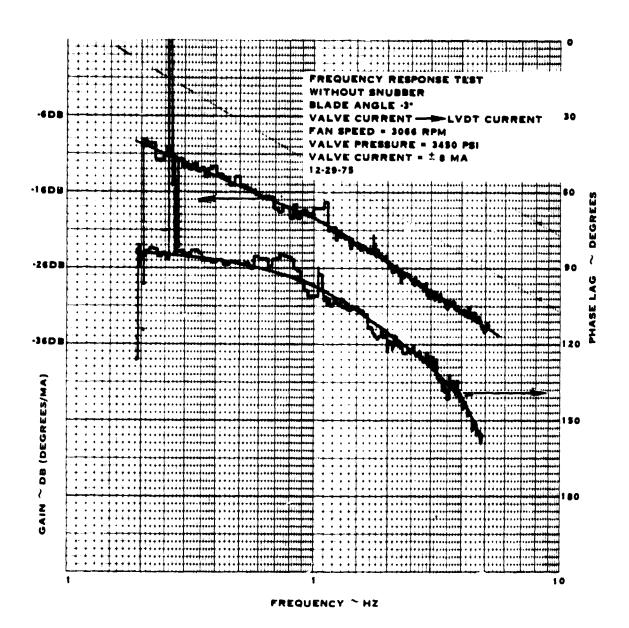


FIGURE 19. QCSEE FREQUENCY RESPONSE

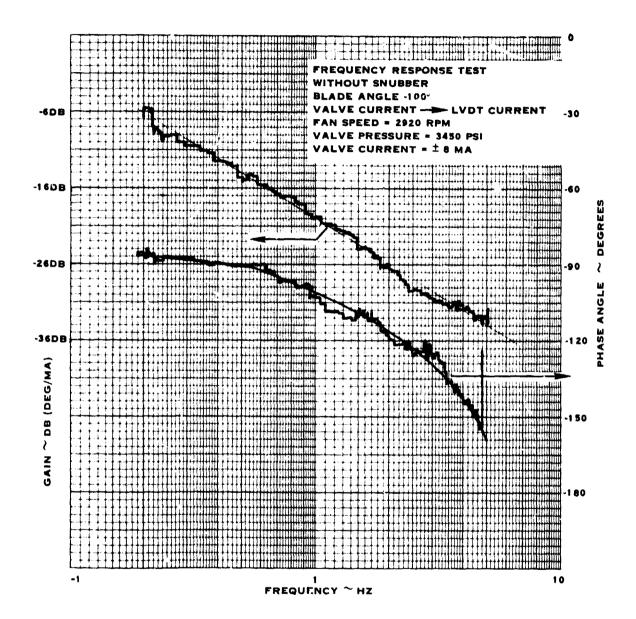


FIGURE 20. QCSEE FREQUENCY RESPONSE

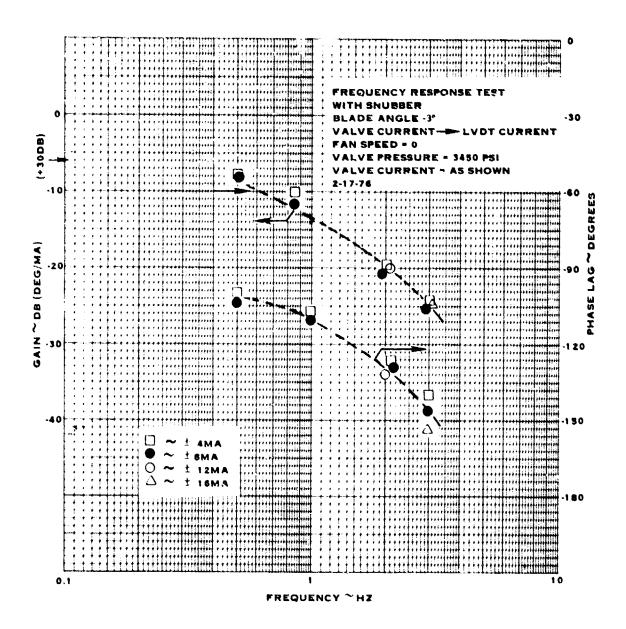


FIGURE 21. QCSEE FREQUENCY RESPONSE

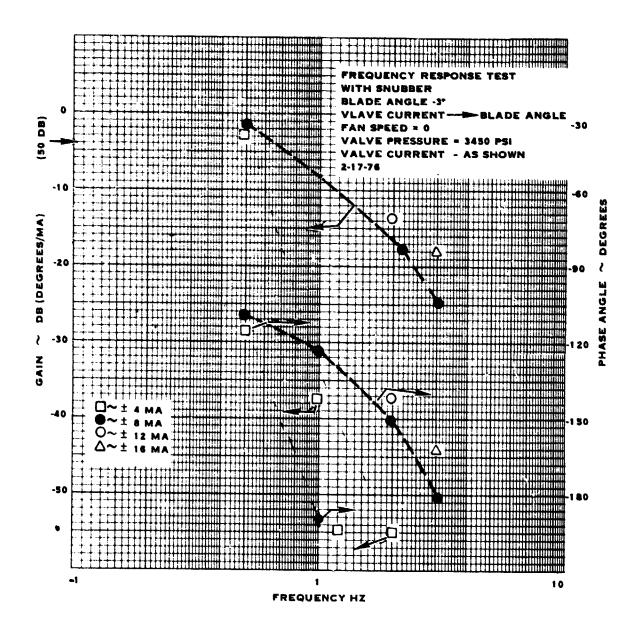


FIGURE 22. GCSEE FREQUENCY RESPONSE

7.0 CONCLUSION

The results of the whirl rig testing shows that the pitch change actuator system, incorporating the snubber, satisfies the design requirements and is structurally adequate for use on the QCSEE being developed for NASA.

The pitch change actuator system was subsequently shipped to General Electric for installation in the engine.

8.0 APPENDICES

APPENDIX A
OPERATING PROCEDURE

1 · B.S. . PINOLOGIA PRING

222PT-37

OPERATING PROCEDURE

FOR

QCSEE ACTUATOR

WHIRL RIG

November 5, 1975

Prepared by: Y) 26 2 2 25

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CONTENTS

- 1.0 GENERAL
- 2.0 RESPONSIBILITIES
- 3.0 RIG OPERATION
 - 3.1 General
 - 3.2 Test Item
 - 3.3 Pre Run Inspection
 - 3.4 Rig Start-up
 - 3.5 Rig Shutdown
 - 3.6 Emergency Shutdown
 - 3.7 After Operation Inspection
 - 3.8 Operating Limits
- 4.0 INSPECTION OF HARDWARE
- 5.0 RIG INSTRUMENTATION
- 6.0 LOG SHEETS
 - 6.1 Frequency
 - 6.2 Content
 - 6.3 Recorded Data

1.0 GENERAL

The purpose of this document is to define a procedure for operating the QCSEE pitch change actuator on the Hamilton Standard whirl test rig (G-7 in the Hydraulics Lab). Copies of this operating procedure will be distributed as follows:

Test Rig - 2 copies
Facilities Group - 1 copy
Engineering - 6 copies (Project and Design)
Lab Supervision - 2 copies
Instrumentation - 2 copies

This document takes precedence over test plans as far as rig operating procedure is concerned. The Test Director will resolve all conflicts.

2.0 RESPONSIBILITIES

All persons associated with the QCSEE actuator test program will be required to conform to the operating principles defined in this document. The actual conduct of the test, in accordance with an approved test plan, will be under the control of a Test Director appointed from the Propeller Project Group. Conduct of the test will be the responsibility of this Test Director or his designated representative.

3.0 RIG OPERATION

- 3.1 <u>General</u> The rig will be operated, at all times, in accordance with these instructions.
- 3.2 <u>Test Item</u> The actuator assembly is defined by drawings 763499 (Beta Regulator) and 763500 (Actuator). Installation and removal of this hardware will be accomplished in accordance with HS 6971 "Assembly and Test of QCSEE Actuator".
- 3.3 <u>Pre-Run Inspection</u> Immediately prior to starting the rig, an inspection must be performed and a "Pre-Run Check List" filled out and signed by both the Rig Operator and the cognizant Engineer conducting the test. A new check list must be filled out if any work is done on instrumentation, rig or test item.

3.4 Rig Start-Up

- 3.4.1 Check to insure that the following has been accomplished:
 - a) Gear box lube system on.
 - b) Clutch water on.
 - c) Actuator control system on.
 - d) Rig speed control set for "manual".
 - e) Drive motor on.

Energize clutch control and bring speed up to 500 rpm. If operation is satisfactory, increase speed to 2000 rpm and record a set of data per paragraph 6.0 before proceeding to test plan.

3.5 <u>Rig Shutdown</u> - Normal rig shutdown will be accomplished, with the rig speed control set at the manual position, by reducing rig speed setting to zero and turning off the clutch and drive motor.

PAGE 64 INTERTIONALLY BLANS

65

- 3.6 Emergency Shutdown If an incident or change in readings occur which the Engineer or Operator judge requires an emergency shutdown, the following procedure will be followed:
 - a) Turn drive motor power off.
 - b) After disc coasts down, turn off gear box lube, actuator lube and actuator hydraulic system.
 - c) Immediately write down all observations noted at the time the incident occurred and mark Sanborn record with time and character of incident.
- 3.7 After Operation Inspection Immediately following rig shutdown, when it is intended that some changes will be made on the test item, or when the rig will be down for more than 30 minutes, a "Post Run Check List" must be filled out by the Engineer and the Operator.
- 3.8 Operating Limits The following are the operating limits which should not be exceeded at any time during operation of this rig:

3600 rpm max. Fan Speed EHV Supply Pressure 3800 psi max. 250°F max. EHV Supply Temperature 22,000 rpm max. Beta Regulator Speed Lube Oil Flow .8 - 1.0 qts/min. 250°F max. Lube Oil Temperature (Actuator) 150°F max. Cell Temperature 2 mils max. Vibration Clutch Water Temp. 180°F max. Shaft Torque 125 in-lbs. max.

4.0 INSPECTION OF HARDWARE

Hardware inspection intervals will be established by the Test Director based on results obtained during the test program.

5.0 RIG INSTRUMENTATION

Instrumentation will be provided on the rig to obtain the data defined in Table I. All measurements noted as recording will be continuously recorded at a paper speed of .1 in/sec.

6.0 LOG SHEETS

A log sheet shall be maintained for all running of the actuator or its components on this rig.

- 6.1 Frequency Entries shall be made on this log sheet in accordance with the following schedule:
 - a) For each start.
 - b) Each new functional test condition being evaluated or at 15 minute intervals.
 - c) For the first endurance cycle of the run and for every fifth endurance cycle thereafter. Readings to be taken at the start and completion of each cycle.
 - d) Prior to shutdown.
 - e) As requested by Test Director.

- 6.2 Content The log sheets will include the following information:
 - a)* Date and time of entry.
 - b) Name of test - G.E. QCSEE Actuator - Functional (or Endurance) Test.
 - c) Rig Speed
 - Blade Angle (from control readout) d)
 - e)* Lube oil flow
 - f)* Lube oil temperature

 - g)* Lube oil pressure h)* EHV Supply Pressure
 - I)* EHV Supply Temperature
 - j) Cell temperature
 - k)* Vibration horizontal and vertical
 - 1) Clutch temperature
 - G.R oil pressure m)
 - n) Test plan paragraph
 - 0)* Cycle number
 - Operator and Engineer's name p)
 - * Record only these for 6.1(c).
- 6.3 Recorded Data Data noted as recording on Table I will be recorded continuously at a record speed of 0.1 in/sec. unless a faster speed is required for the particular test being conducted. All records will be suitably marked with test plan paragraph number or cycle number and date and time for identification and to permit correlation with the rig log sheets.

TABLE 1

Measurement	Range	Type
EHV Supply Pressure	0-4000 psig	Visual
EHV Current Signal	<u>+</u> 100 ma	Recording
Flow to EHV	0-45 gal/min	Recording
4 P Across Motor (1)	0-3500 psig	Recording
Fluid Temperature	0-300°F	Visual
Blade Angle Command	+20° to -120°	Visual and Recording
LVDT Feedback Voltage (1)	<u>+</u> 5 V dc	Recording
Flex Shaft Speed	0-24,000 rpm	Recording
Flex Shaft Torque	0-200 in.lb.	Recording
Lube 011 Flow	0-1 qt/min.	Visual
Lube 011 Prossure	0-100 psig	Visual
Lube 011 Temperature	0-300°F	Visual
Fan Speed	0-3700 rpm	Visual and Recording
Cell Temperature	0-300°F	Visual
Vibration - Horizontal		Visual
Vibration - Vertical		Visual
Fan Blade Angle	+20° to -120°	Visual

POST RUN CHECK LIST

QCSEE ACTUATOR - G-7 RIG

Date:	
Rig Time:	
Flight Cyc	les:

	and the second s		Initi	
	Item	Condition	Operator	Engineer
1	All hardware appears structurally sound.			
2	No evidence of oil leakage.			
3	Arm, disc and cam area clean, dry and free of foreign objects.			
4	All visible bolts, nuts, mounts, etc. appear secure.			
5	All pumps shut off.			
6	Disc and actuator covered up if unit will be shut down for any period of time and is not being worked on.			
7	All data recorded and records properly marked.			

REMARKS -

FRE RUN CHECK LIST

QCSEE ACTUATOR - G-7 RIG

Date:	
Rig Tis	le t
Flight	Cycles:

	a en entre de la companya de la comp	1	Init	ial
	Item	Condition	Operator	Engineer
1_	Disc uncovered.			
2	Instrumentation connected and operating.			
3	Arm, disc and cam area clean, dry and free of foreign objects.			
4	All visible bolts, nuts, mounts, etc. appear secure.			
5	Stop switches correctly set and Beta Regulator indexed to actuator (feedback and actual blade angle agree).			
6	Hydraulic pumps on, bypass closed and correct pressure at servo valve (per test plan).			
7	Actuator lube pump on and set for psi at Beta Regulator.			
8	All personnel and loose material out of cell and cell doors secured.			

REMARKS -

APPENDIX B

TEST PLANS

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Hamilton Standard Over or Course Secret Companies HSER 7002

No. 222PT-31 Rev. A DATE: 10/28/75 PLAN OF TEST	PAGE 1 OF 12
DATE: 10/28/75 PLAN OF TEST	
ITEM: 763500 Actuator (QCSEE)	PREPARED BY: D. Leishman
CONTRACT: GE 200-4XX-14G-38570	APPROVED BY TO THE
TEST PERIOD:	*

- 1. WHAT IS ITEM BEING TESTED?
- 2. WHY IS TEST BEING RUN? WHAT WILL RESULTS SHOW OR BE USED FOR?
- 3. DESCRIBE TEST SETUP INCLUDING INSTRUMENTATION, ATTACH SKETCH OF INSTALLATION,
- 4. ITEMIZE RUNS TO BE MADE GIVING LENGTH OF EACH AND READINGS TO BE TAKEN,
- 5. SPECIAL INSTRUCTIONS: SAFETY PRECAUTIONS FOR OPERATORS AND HANDLING EQUIPMENT, OBSERVATIONS BY SIGHT, FEEL, OR HEARING, LIST POINTS OF OBSERVATION WHICH MIGHT CONTRIBUTE TO ANALYSIS OF (A) PERFORMANCE OF UNITS, (B) INCIPIENT TROUBLE BEFORE IT OCCURS, AND (C) CAUSE OF FAILURE.
- 6. HOW WILL DATA BE USED OR FINALLY PRESENTED? GIVE SAMPLE PLOT. CURVE. OR TABULATION AS IT WILL BE FINALLY PRESENTED.

NUMBER ENTRY AS LISTED ABOVE AND DESCRIBE BELOW

1.0	The item being tested is the pitch change actuator for the QCSEE assembled
	for reverse through stall operation.
2.9	The test is being run to determine the operating characteristics of the
	actuator, verify that it satisfies the design requirements, and assure its
<u> </u>	structural adequacy for use in an engine test.
ļ 	
3.0	The actuator, together with a disc and stub blades supplied by GE will be
	mounted in G-7 Whirl Rig which has been modified to accept the unit.
	Reference Figure 1. A closed loop variable gain control system with a capabal-
	ity to vary gain between 570 and 2700 ma/volt/volt excitation will he utilized
	for the test. The instrumentation to be used during the test is listed in
	Table 1. Figure 2 is a drawing showing the hardware necessary to convert the
	actuator to a front flex shaft input, and Figure 3 is a lubrication schematic
	of the actuator.
4.0	The following tests will be conducted. All blade angles are in GE terms.
	Figure 4 is a plot of blade angle vs. counterweight angle.
	73

TABLE 1

Measurement	Range	Туре
EHV Supply Pressure	0-4000 psig	Visual
EHV Current Signal	<u>+</u> 100 ma	Recording
Flow to EHV	0- 4 5 g a l/min	Recording
△ P Across Motor (1)	0-3500 psig	Recording
Fluid Temperature	0-300°F	Visual
Blade Angle Command	+20° to -120°	Visual and Recording
LVDT Feedback Voltage (1)	<u>+</u> 5 V dc	Recording
Flex Shaft Speed	0-24,000 rpm	Recording
Flex Shaft Torque	0-200 in.lb.	Recording
Lube Oil Flow	0-1 qt/min.	Visual
Lube 0il Pressure	0-100 psig	Visual
Lube Oil Temperature	0-300°F	Visual
Fan Speed	0-3700 rpm	Visual and Recording
Cell Temperature	0-300°F	Visual
Vibration - Horizontal		Visual
Vibration - Vertical		Visual
Fan Blade Angle	+20° to -120°	Visual

- 4.1 Lubrication Flow Check
- 4.1.1 With the flex drive shaft disconnected from the actuator, determine the pressure setting necessary at the beta regulator to obtain a flow of .85 qts/min through the shaft at a shaft speed of 0 rpm.
- 4.2 LVDT Null & Calibration (fan speed = 0 rpm)
- 4.2.1 Using the manual input to the Beta Regulator, determine the LVDT output and blade angle at the mechanical stops of the actuator. Reset the LVDT null so that the output varies an equal amount on either side of null as the blade moves through the full range of travel.
- 4.2.2 Using the manual input to the Beta Regulator, calibrate actual blade angle versus LVDT output voltage every 5° over the full range of travel in both increasing and decreasing pitch directions. During this calibration, set the travel limit switch cams to actuate at -8° and -96° blade angle.
- 4.3 Travel Limit Switch
- 4.3.1 With the travel limit switch cams set for -8° and -96°, and a fan speed of 0 rpm, determine the actual blade angle at which the blades stop when the travel limiting switch is actuated at flex shaft speeds of approximately 6000 rpm, 12,000 rpm and 21,000 rpm. Regulate EHV supply pressure to control rate.
- 4.3.2 With the travel limit switch cams set for -8° and -96°, and a fan speed of 2500 rpm, determine the actual blade angle at which the blades stop when the travel limit switch is actuated at flex shaft speeds of approximately 6000 rpm, 12,000 rpm and 21,000 rpm.
- 4.3.3 Reset the travel limit switch cams for $+12.5^{\circ}$ and -116° or to a modified value if testing does not confirm the calculated values of travel required to stop the system $(6.5 -7^{\circ})$.
- 4.4 Blade Angle Position Accuracy
- 4.4.1 With the EHV inlet pressure set at 2000 +100 psig and a fan speed of 2500 rpm, cycle the actuator in accordance with Table 2. At each condition, record actual vs. LYDT blade angle.

Table 2

Step (Reference Table 5)	Blade Angle	
1	+12°	
3	-3°	
5	0°	
6	+7°	
7	+5°	
8	+7°	
9	+9°	
10	+7°	
12	-100°	
15	+12°	

4.4.2 With the EHV inlet pressure set at 3450 +100 psig, and fan speed as shown, cycle the actuator in accordance with Table 2a. At each condition, record actual vs. LVDT blade angle.

Table 2a

Step (Reference Table 5)	Blade Angle	Fan Speed
•		
1	+12°	2500
3	-3°	3068
5	0°	3068
6	+7°	3068
7	+5°	3068
8	+7°	3068
9	+9°	3068
10	+7°	3068
12	-100°	3068
15	+12°	2500

- 4.5 Performance
- 4.5.1 Determine the pressure required to start actuator motion in both increasing and decreasing pitch directions for the conditions specified in Table 3.

Table 3

Fan Speed	Starting Blade Angle		
2500 rpm	+12° (increasing only)		
3068 rpm	+5°		
3068 rpm	+7°		
3068 rpm	+9°		
3408 rpm	٥°		
3408 rpm	-3°		
2500 rpm	-100° (decreasing only)		

4.5.2 Determine the pressure required to sustain actuator motion for the blade angle excursions specified in Table 4.

Table 4

Fan Speed	Blade Angle Excursion
2500 rpm	+12° to -3°
3408 rpm	-3° to +0°
3068 rpm	0° to +7°
3068 rpm	+7° to +5°
3068 rpm	+5° to +7°
3068 rpm	+7° to +9°
3068 rpm	+9° to +7°
3068 rpm	+7° to -100°
2500 rpm	-100° to +12°

4.5.3 Determine the average pitch change rate for a blade angle excursion from -3° to -100° at a fan speed of 3315 rpm. EHV supply pressure to be set at 3450 \pm 50 psig. (rate based on LVDT feedback voltage)

- 4.5.4 With the blade angle set at 0°, fan speed of 3408 rpm, and EHV supply pressure 3450 ±50 psiq, demonstrate the minimum blade angle change which can be obtained around 0°.
- 4.5.5 With the blade angle set at -100° , EHV supply pressure at 0 psig, increase fan speed to 3578 rpm and record any blade angle change.
- 4.5.6 With the blade angle set at 0°, EHV supply pressure at 0 psig, increase fan speed to 3408 rpm and record any blade angle change.
- 4.6 Frequency Response
- 4.6.1 At a blade angle of -3° and with a fan sneed of 0 rpm, determine Bf/ EHV and LVDT/ EHV. Static frequency response and phase angle shall be determined for EHV current peak to peak amplitudes not to exceed ± 40 ma or that current input which causes a Bf variation of ± 1.9 to ± 2.1 degrees at frequencies of 0.5, 1.0, 1.5, 2.0, 3.0, and 4.0 Hz.
- 4.6.2 Rotating frequency response tests will be conducted to determine & LVDT AEHV.

Two base point conditions will be used:

- 1. Fan speed = 3066 rpm. Bf = 3° open.
- 2. Fan speed = 2920 rpm. Bf = 100° open.

Frequency response and phase angle for base point conditions 1 and 2 shall be determined for EHV current peak to peak amplitudes not to exceed ± 40 ma or that current input which causes a LVDT variation of $\pm .0135$ to $\pm .0165$ volts/volt at frequencies of 0.5, 1.0, 1.5, 2.0, 3.0, and 4.0 Hz.

- 4.7 Endurance Test
- 4.7.1 Prior to and at completion of the endurance test, the unit will be disassembled and the hardware will be examined to identify any potential problem areas.
- 4.7.2 The endurance test will consist of 500 cycles of the actuator in accordance with Table 5. During testing, the EHV supply pressure will be maintained at 3450 ±50 psig. The cycles may be run faster than the normal anticipated engine duty cycle in order to shorten the required test time. The cycle frequency will be established during the functional test program. The limiting factor will be the ability of the unit to dissipate the heat generated in the no-back and the clutch. The blade angle must be allowed to stabilize at each point prior to going on to the next point except during modulating operation. (Steps 7-11 of Table 5)
- 5.0 Special Instructions

Prior to and following operation, the check list shown in Figure 5 must be completed. Note any unusual noises or vibrations, or any changes in noise

TABLE 5

Step	Fan Speed	Blade Angle
1	2500 rpm	+12°
2	2500	-3°
3	3408	-3°
4	3408	0°
5	3068	0°
6	3068	+7°
7	3068	+5°
8	3068	+7°
9	3068	+9°
10	306 8	+7°
11	Repeat steps 7 thru 10) twenty times
12	3068	-100°
13	3408	-100°
14	2500	-100°
15	2500	+12°

5.0 Continued

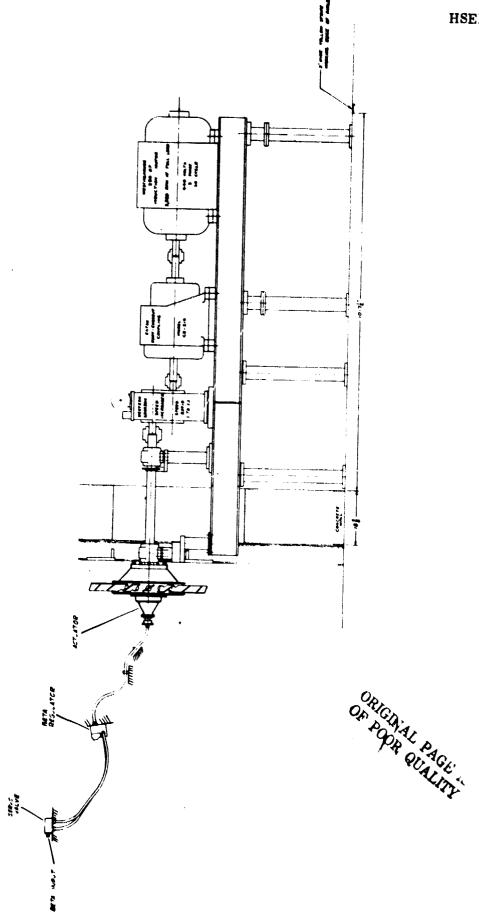
or vibration. During the static and performance testing, a log sheet shall be kept which contains the parameters denoted as visual in Table 1 for each test point. Functional test data denoted as recording shall be taken at each test point at a speed commensurate with the test being run.

During the endurance test, a log sheet shall be kept which contains the parameters denoted as visual in Table 1 for the first and each fifth cycle thereafter. Endurance test data denoted as recording shall be taken continuously at a speed of .1 in/sec.

In order to provide a complete time/rpm history for the fan rotor components, data denoted as recording shall be taken at a speed of .1 in/sec at all times when the fan is rotating except when higher recording speeds are needed for a specific test point.

A component test report will be prepared following the completion of the whirl rig tests. Discussions and data presented in the report will cover all tests conducted during the whirl rig test program. All test data will be available for review and use by GE and NASA representatives.

FIGURE 1. WHIRL RIG ARRANGEMENT



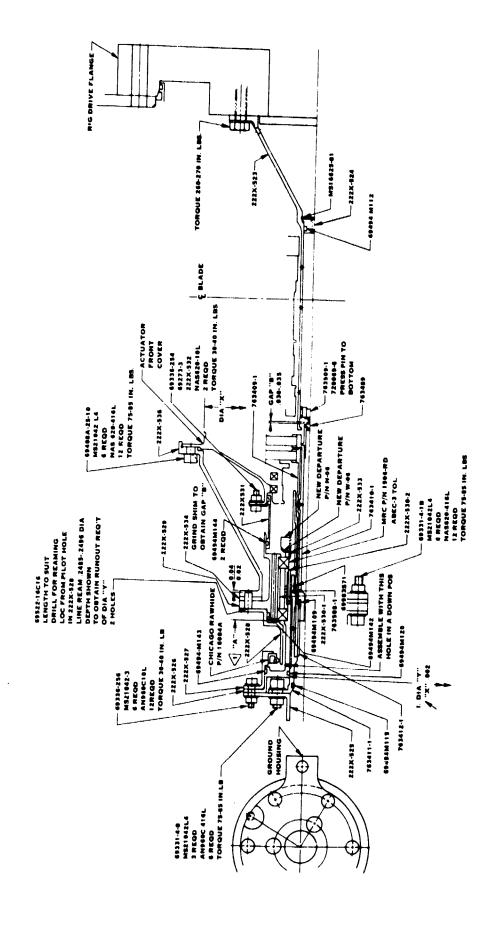


FIGURE 2. ASSEMBLY, FRONT INPUT HARDWARE

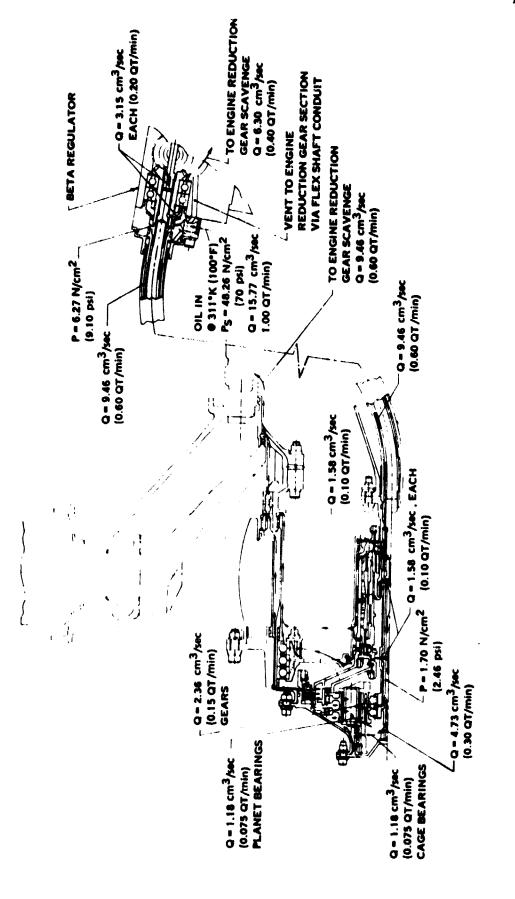


FIGURE 3. LUBRICATION SCHEMATIC

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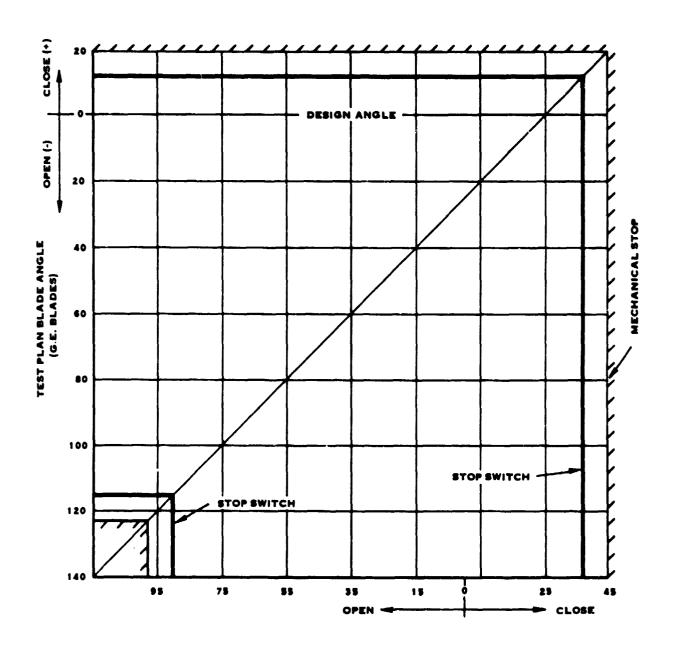


FIGURE 4. QCSEE PITCH CHANGE ACTUATOR TEST PLAN BLADE ANGLE VS. COUNTERWEIGHT ANGLE (TEST PLAN 222 PT-31 REV. A)

QCSEE ACTUATOR WHIRL RIG CHECK LIST

Pre Operation

- 1. Actuator assembled per HS6971
- 2. System is correctly rigged and indexed
- 3. Feedback blade angle agrees with actual angle
- 4. Instrumentation secured
- 5. Cell cleaned
- 6. Hydraulic supply pump on
- 7. Lubrication supply pump on
- 8. Scavenge pump on

Post Operation

- 1. Check for leaks
- 2. Check feedback vs. actual blade angle
- 3. All required data recorded

TEE Dat	e:1-	222PT-: -30-76		BI.	HSER 7002 1 or3 E. Smith
				APPROVED BY:	
		•			
1.	WHAT I	5 17 6 M	OLING TESTED'		
2.	WHY IS	TEST (BEING MUN? WHAT WILL RESULTS SHOW ON BE USED FOR?		Or Park
3.	DESCRI	DE TEST	T SET UP INCLUDING INSTRUMENTATION. ATTACH SEETCH	OF INSTALLATION.	OF POOR OIL
4.	I TEMI 2	E NUNS	TO BE MADE GIVING LENGTH OF EACH AND READINUS TO	BE TAKEN.	QI,
5.	S. SPECIAL INSTRUCTIONS: SAFETY PRECAUTIONS FOR OPERATORS AND HANDLING EQUIPMENT. OBSERVATIONS BY SIGHT, FEEL, UR HEARING, LIST POINTS OF OBSERVATION WHICH MIGHT CONTRIBUTE TO ANALYSIS OF (A) PERFORMANCE OF UNITS, (Q) INC. IP LENT TROUBLE BEFORE IT OCCURS, AND (c) CAUSE OF FAILURE.				
€.			BE USED OR FINALLY PRESENTED? GIVE SAMPLE PLOT, FINALLY PRESENTED.	CURVE, OR TABULATION	
•			NUMBER ENTRY AS LISTED ABOVE AN	D DESCRIBE BELOW	
	1.0 The item being tested is the pitch change actuator for the QCSEE with a soft quill shaft (snubber). (Ref. 763500-1 Rev. D)				
	2.0 The test is being run to evaluate the performance of the actuator system incorporating a soft quill shaft to reduce the impact loading on the no-back output shaft.				
	3.0 The actuator will be installed in G-7 Whirl Rig as defined in 222PT-31 Rev. A. The spindle retaining nut torque will be adjusted to obtain 600-700 in. lbs. of drag torque on each spindle. Retaining nut torque and drag torque will be recorded for each arm bore.			obtain	
	This test will be conducted when the disc is installed on the rig and the spindle torque will be rechecked (without changing retaining nut position) prior to disassembling disc.				

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HAMILTON STANDARD

Test No. 222PT-38

PLAN OF TEST

Date: 1-30-76

Job: 763500 Actuator with Soft Quill Shaft Prepared by: D. E. Smith

Project & Order GE 200-4XX-14G-38570

4.0 The following tests will be conducted.

4.1 System Losses

The torque to move the blades will be measured at the beta regulator manual drive when the installation is complete, after each reassembly and following completion of the 50 flight cycles. Torque will be measured in both directions of motion at $\pm 10^{\circ}$, 0° and $\pm 100^{\circ}$ of blade angle.

4.2 Blade Angle Calibration

The LVDT's will be calibrated against the position of the counterweights, for both directions of motion. Calibration will be in approximately 2° increments between +12 and -10° and 10° increments from -10° to -100°.

4.3 Stop Switches

Stop switches shall be set $6.5^{\circ}-7^{\circ}$ from the mechanical stops. Stop angle and switch setting will be recorded.

4.4 Max Rate Testing (Ref. 4.5.3 of 222PT-31A)

After checking system operation over entire range and adjusting control settings, perform a reverse transient at 3315 fan rpm and 3450 psi EHV supply pressure. Pump stroke should be adjusted for $10,000 \pm 500$ rpm flex shaft speed. Reduce ig speed to 2700 rpm and unreverse at $10,000 \pm 500$ rpm. Repeat test to obtain two (2) cycles and remove and examine no back and snubber. Determine wind up which snubber experienced.

Reassemble no back and install in rig. Repeat test at $17,500 \pm 500$ flex shaft rpm. Remove and examine no back and snubber and record snubber wind up.

4.5 Endurance Test

Conduct fifty (50) cycles of endurance testing in accordance with paragraph 4.7.2 of 222PT-31A. Max flex shaft speed will be adjusted to achieve 17500 \pm 500 rpm during reverse transient portion of run. Following the endurance test, the no back and snubber hardware will be examined.

4.6 Limit Switch Overtravel

The amount of blade travel required to stop, after tripping the travel limit switches, will be determined at both ends of the operating range. Test will be run statically. For each end of travel, the stopping distance will be determined for flex shaft speeds of 10,000, 12,000, 15,000 and 17,500 rpm (\pm 500 rpm). The control time constant will be set for .02 seconds initially. If necessary to increase this time constant, the new value shall be recorded on the log sheet.

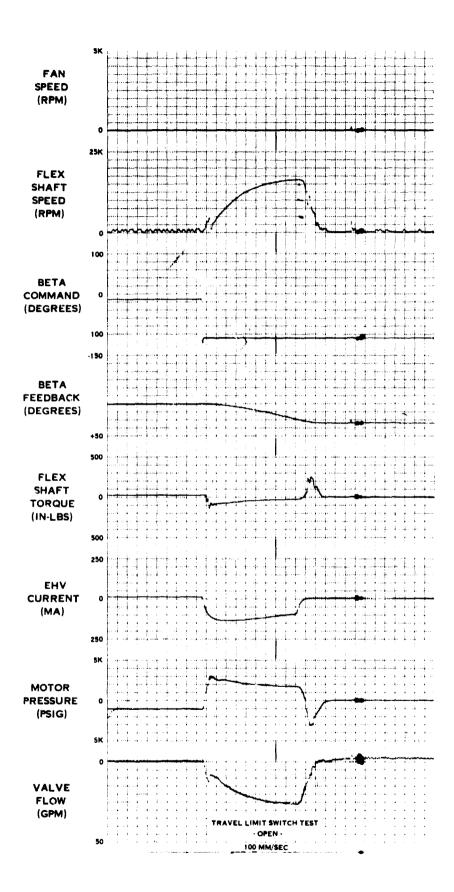
4.7 Frequency Response

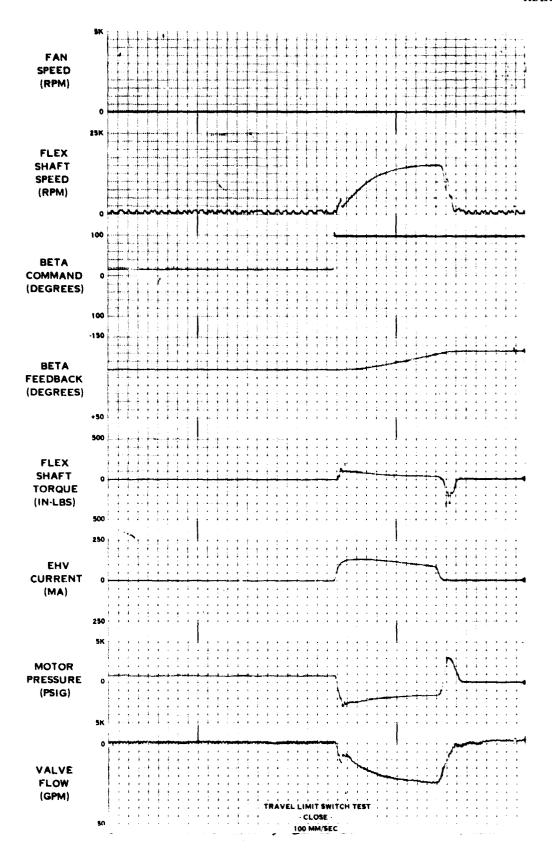
The static frequency response shall be measured as described in 4.6.1 of 222PT-31A.

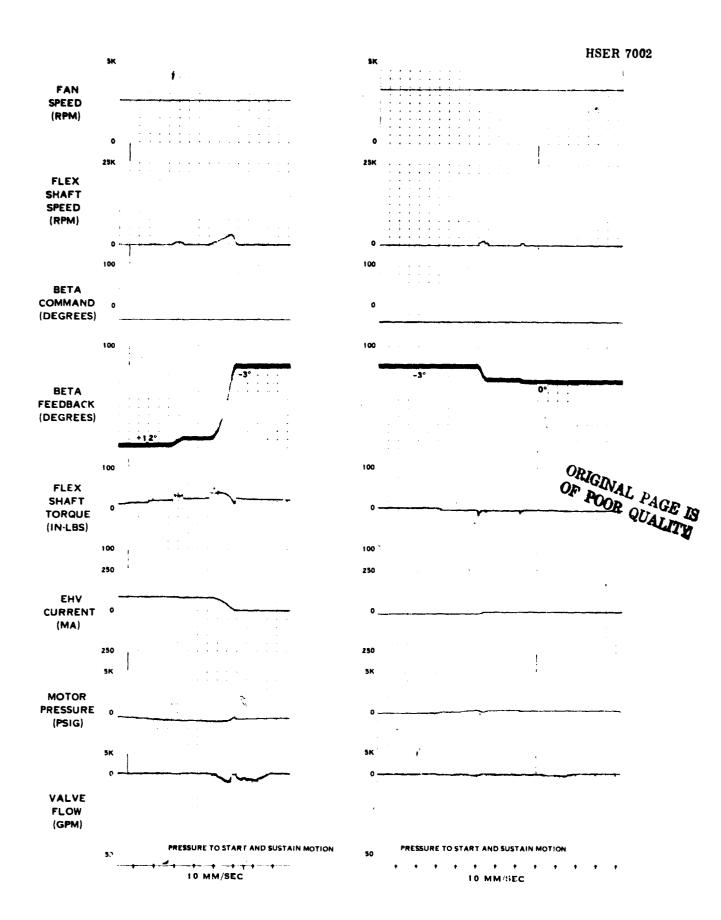
- 5.0 Same as 222PT-31 Rev. A.
- 6.0 Same as 222PT-31 Rev. A.

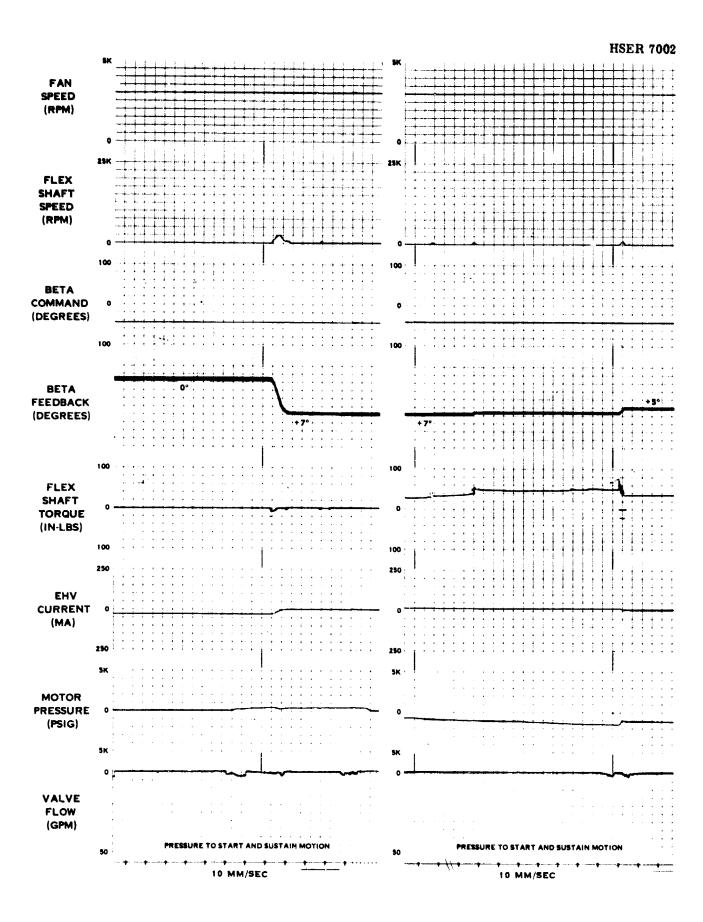
APPENDIX C

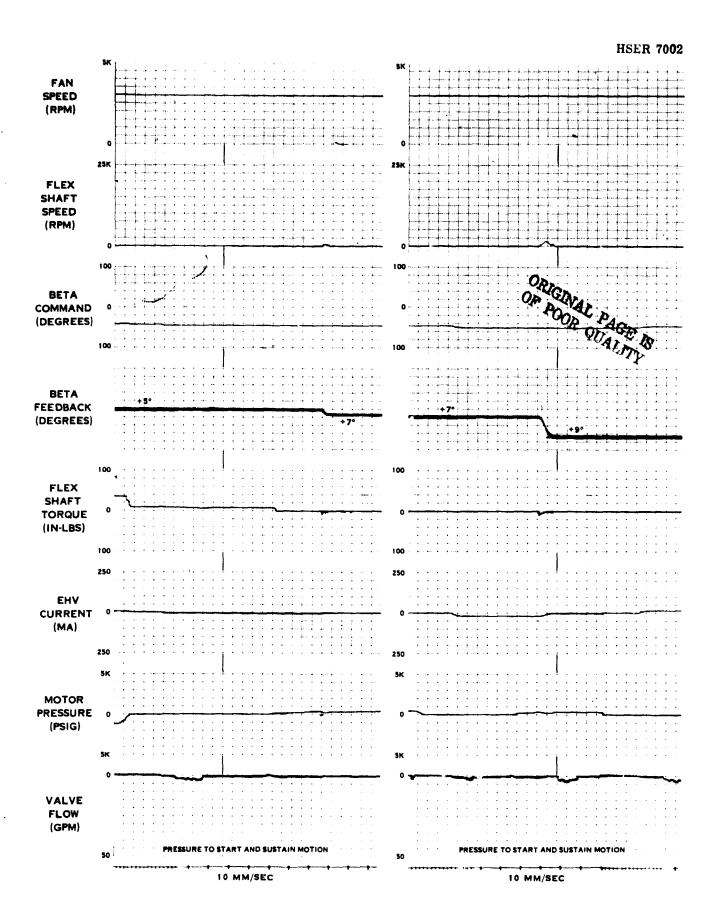
SANBORN RECORDS

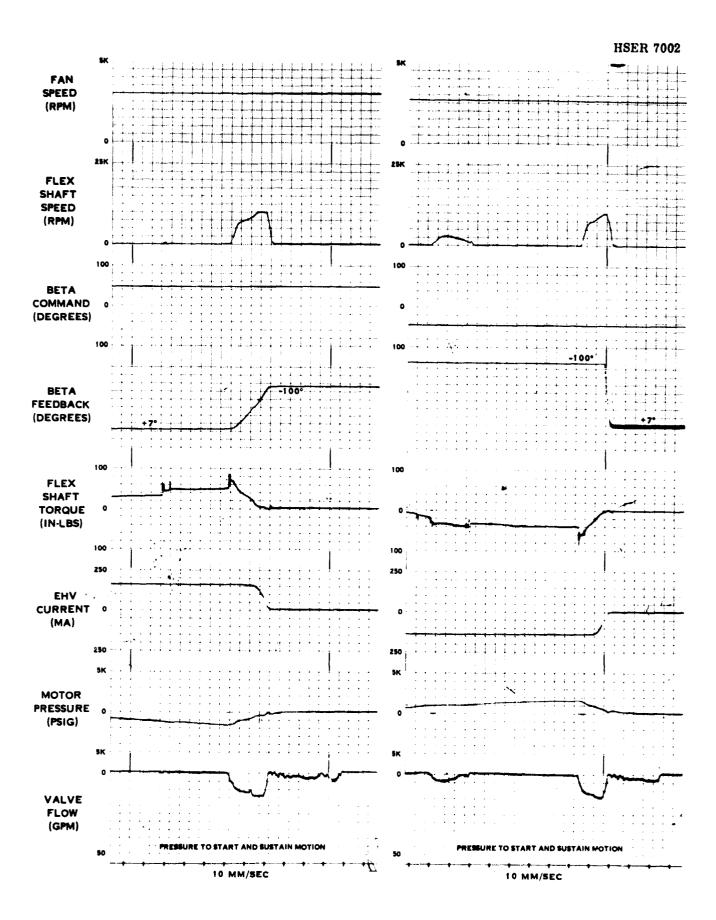


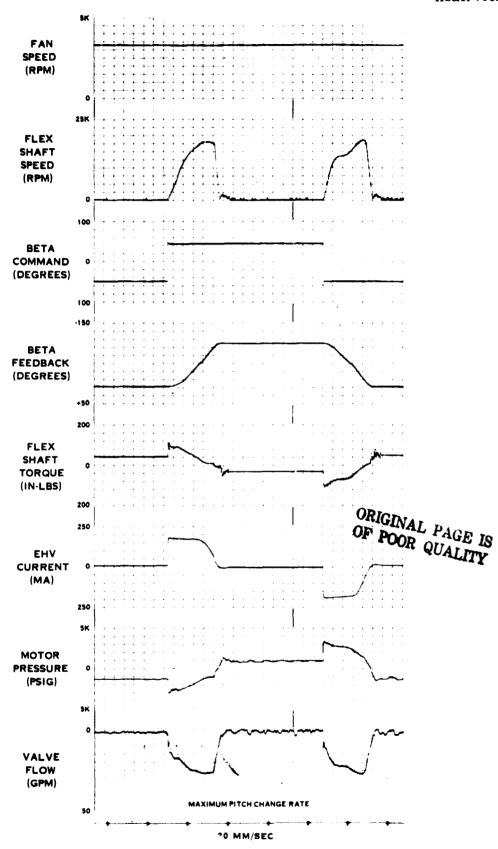


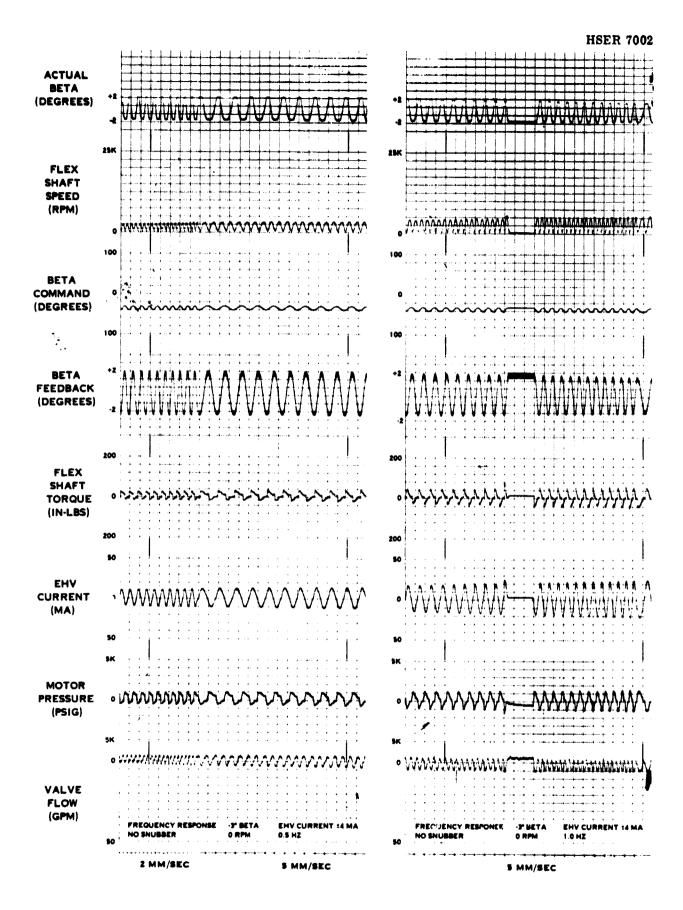


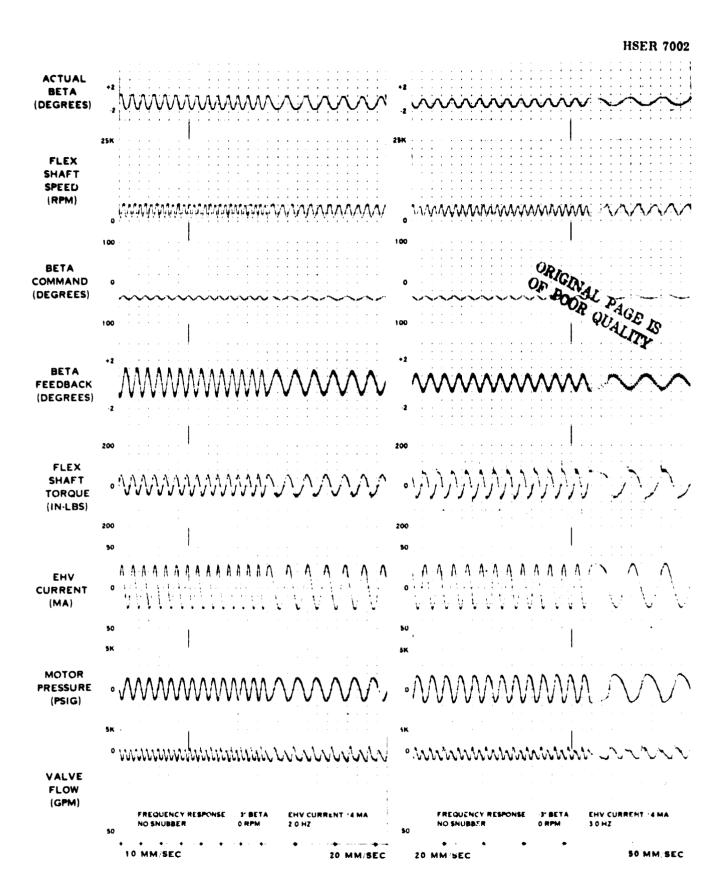




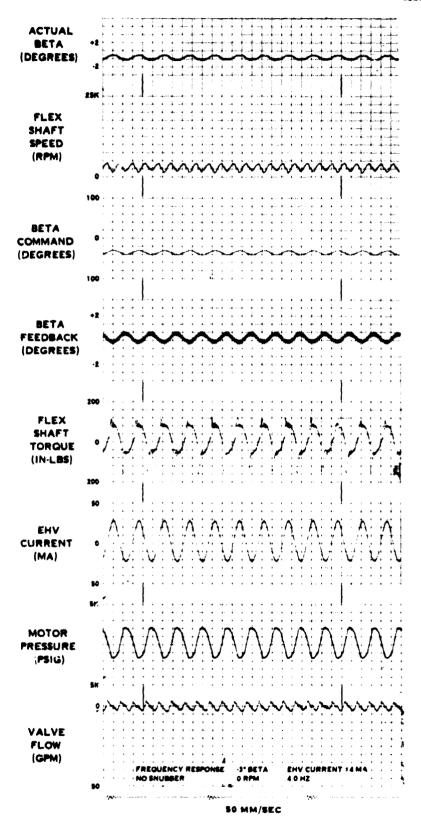


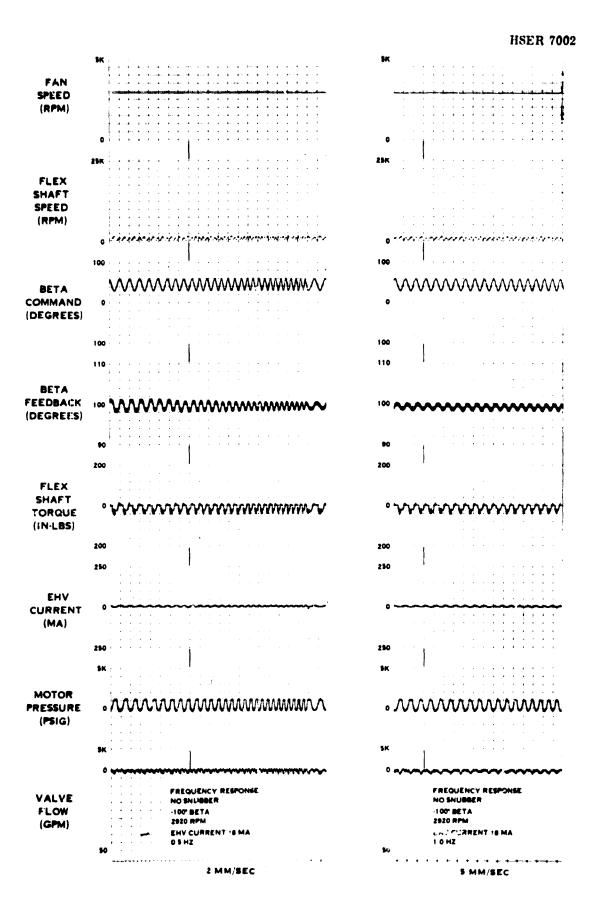


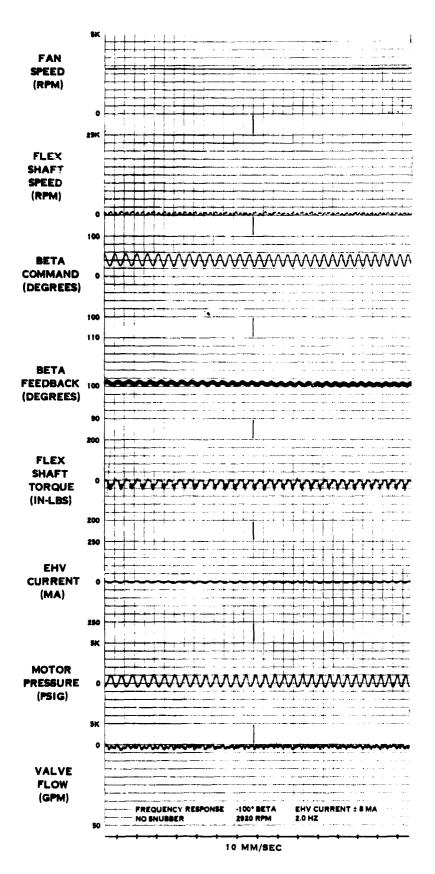




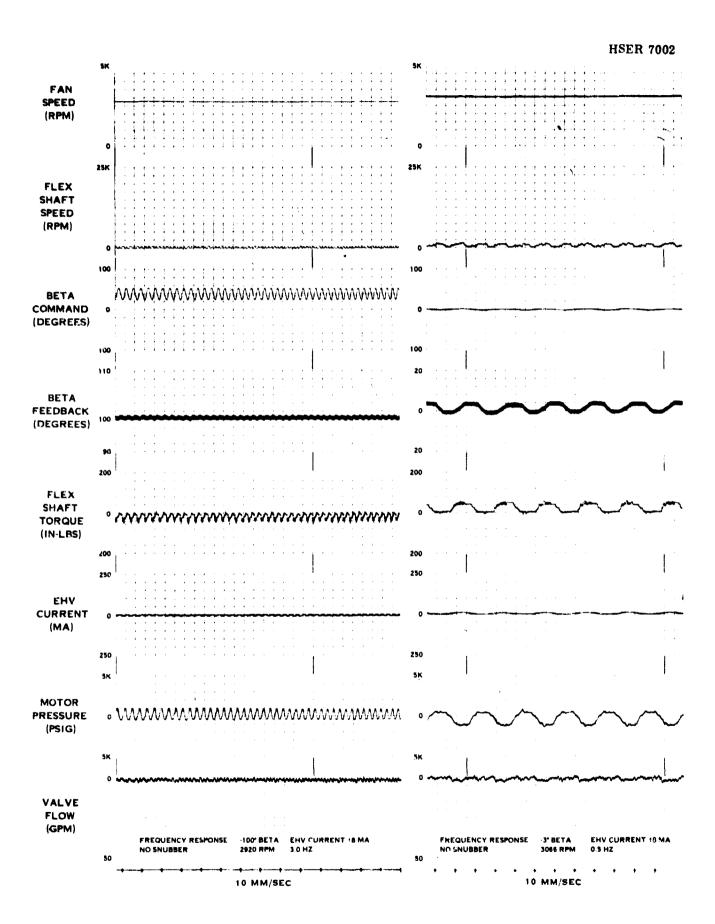
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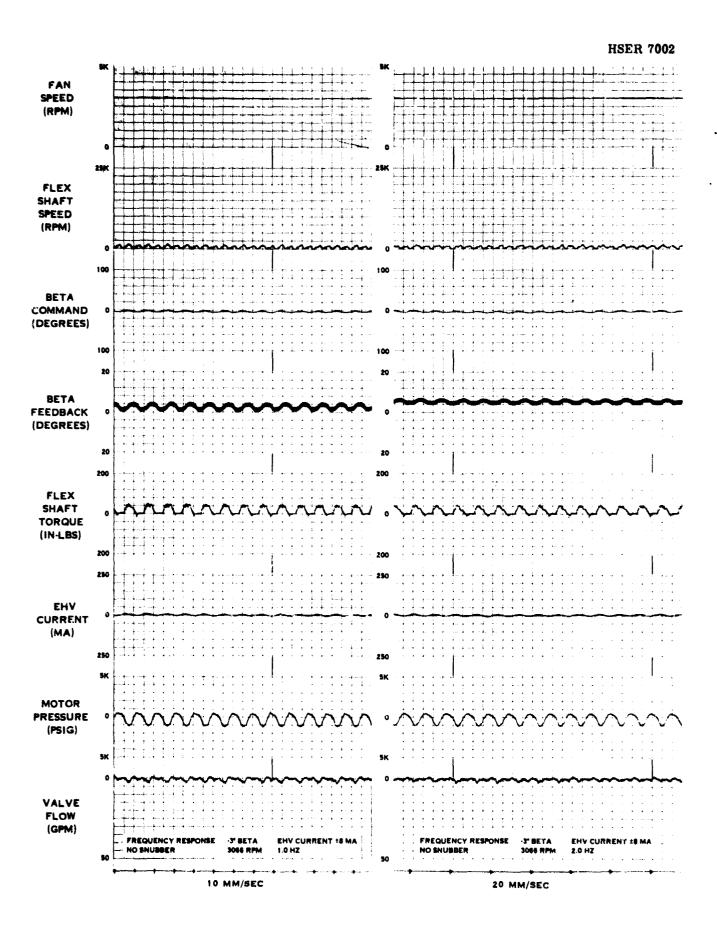


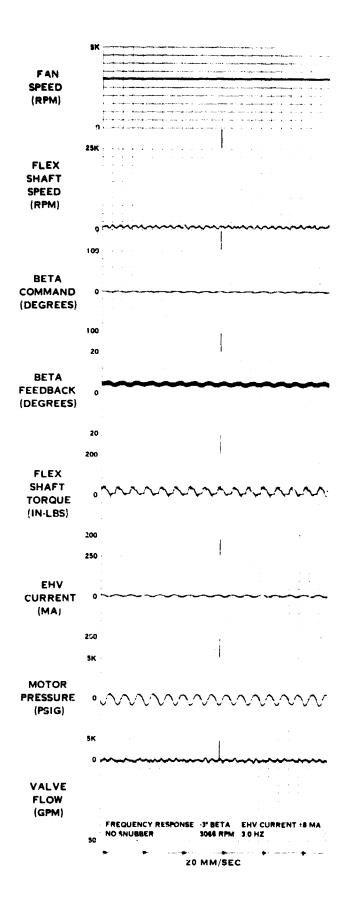


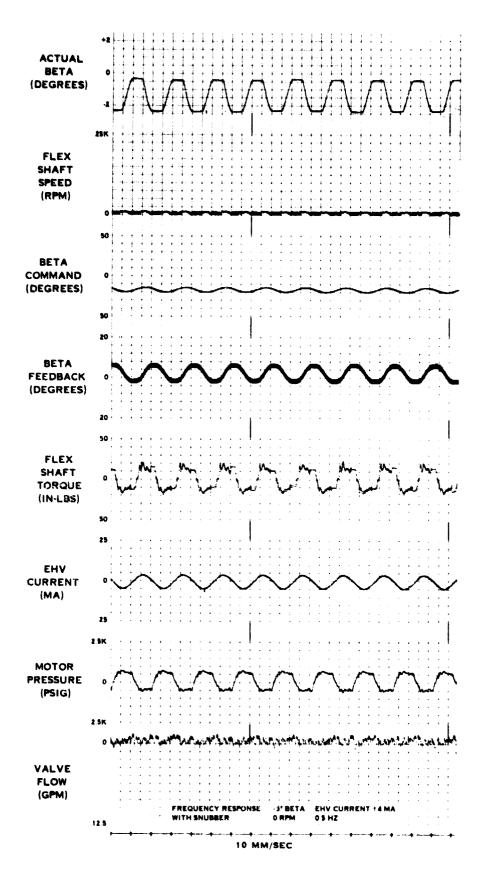


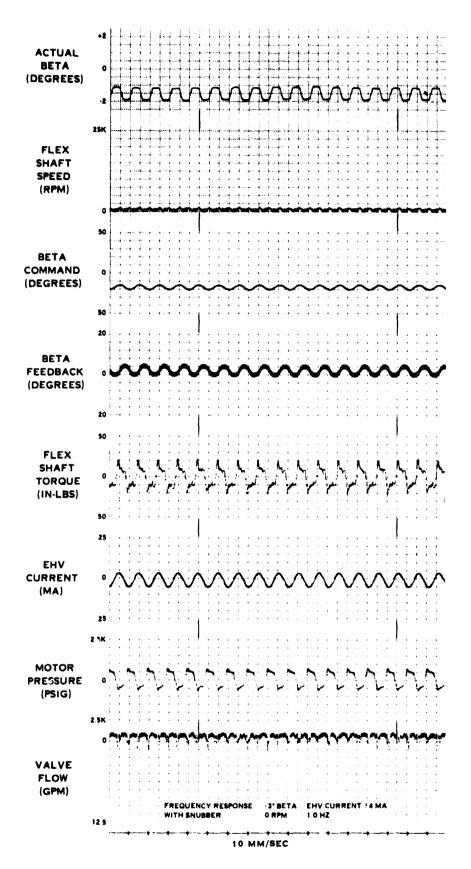
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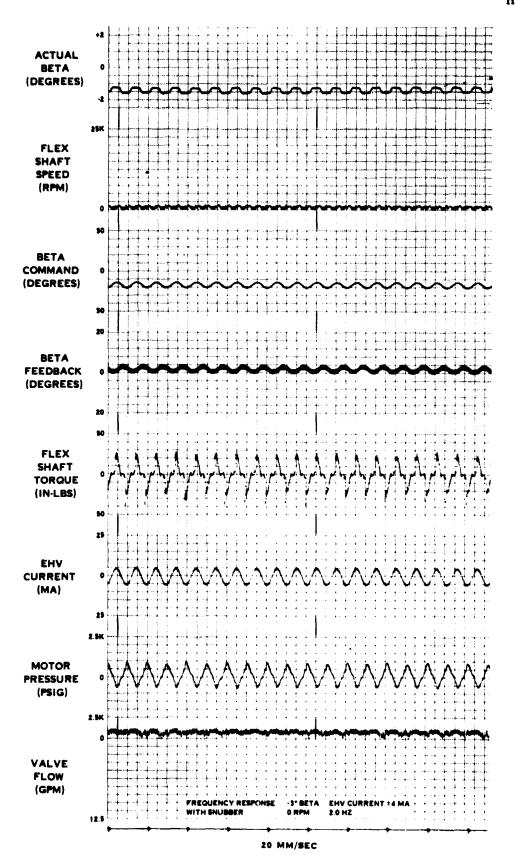


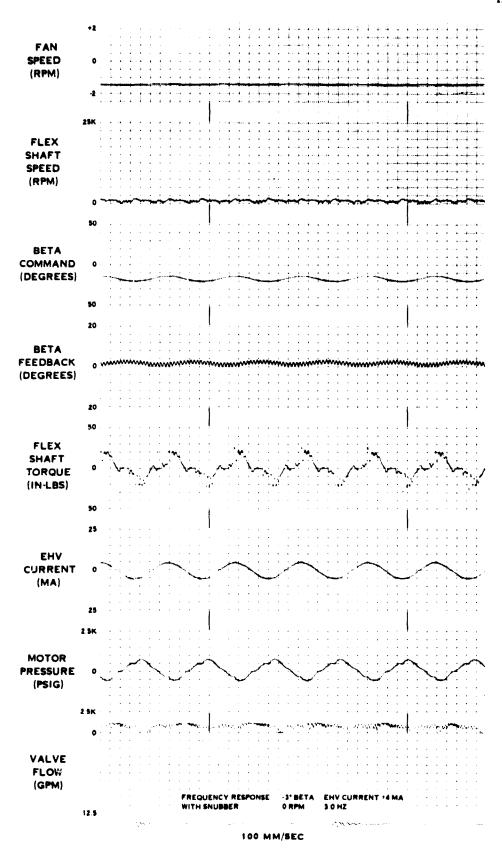


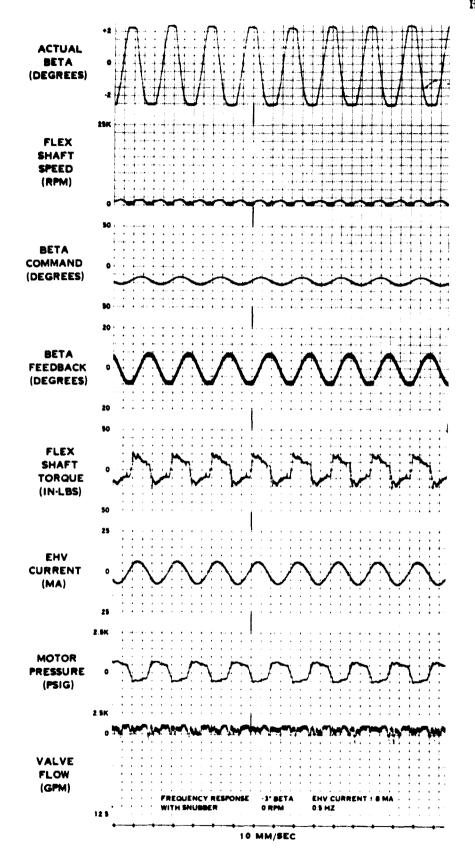


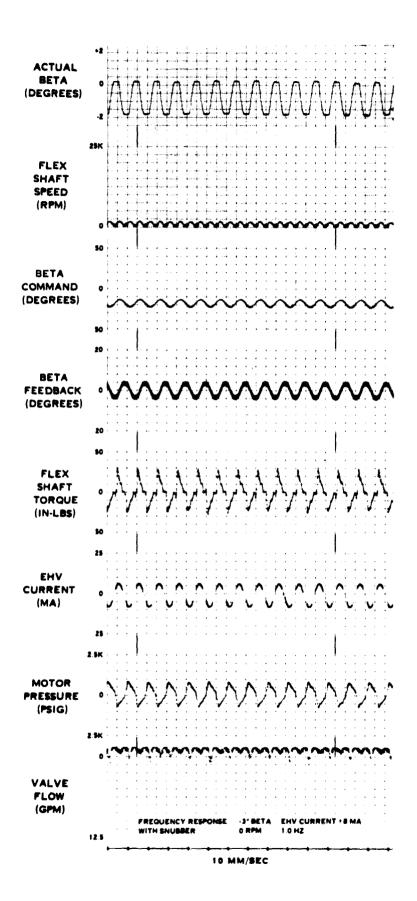


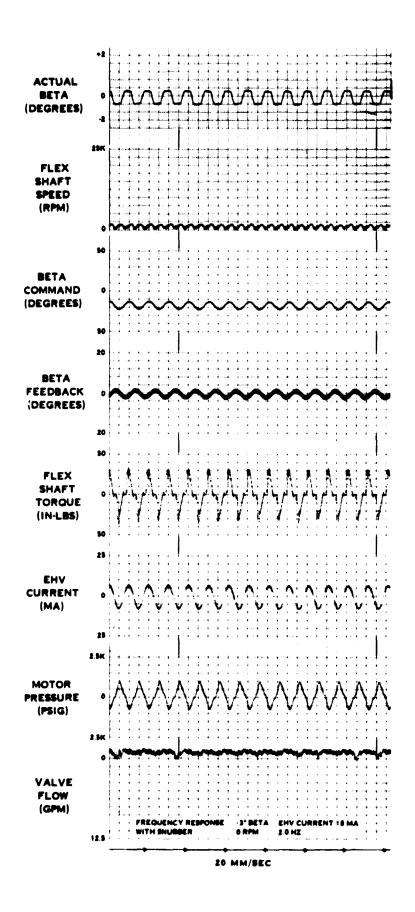


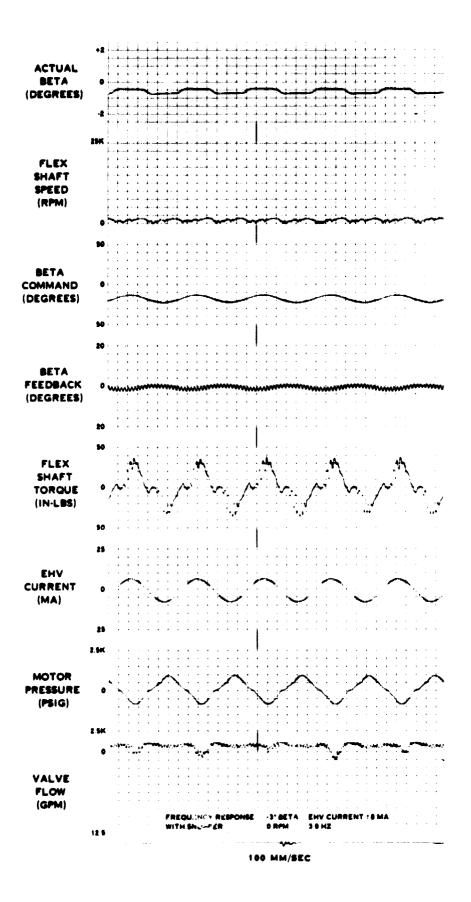


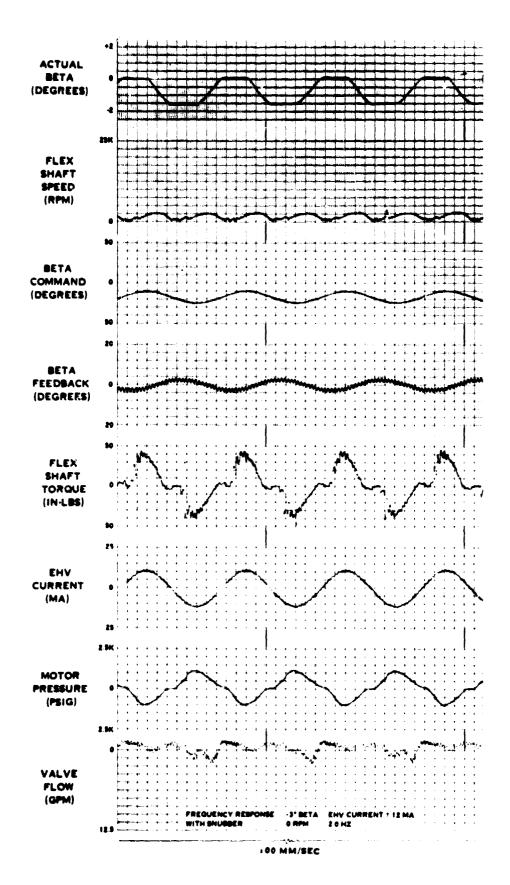


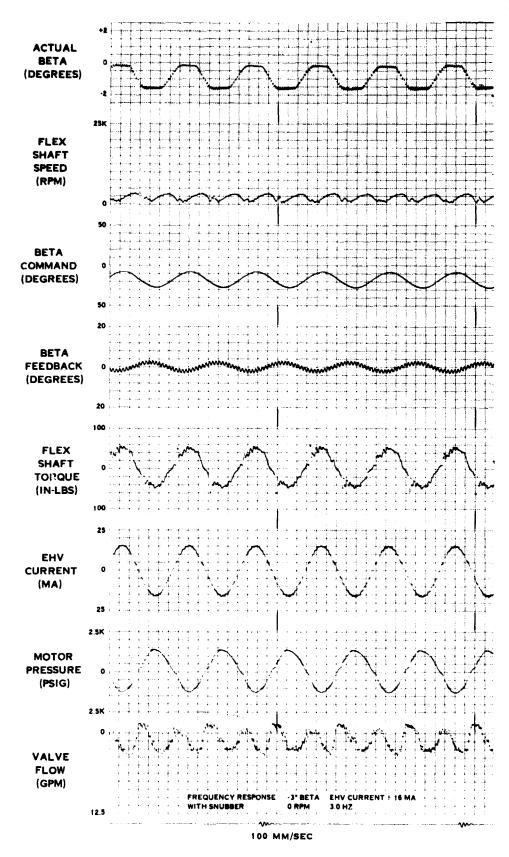


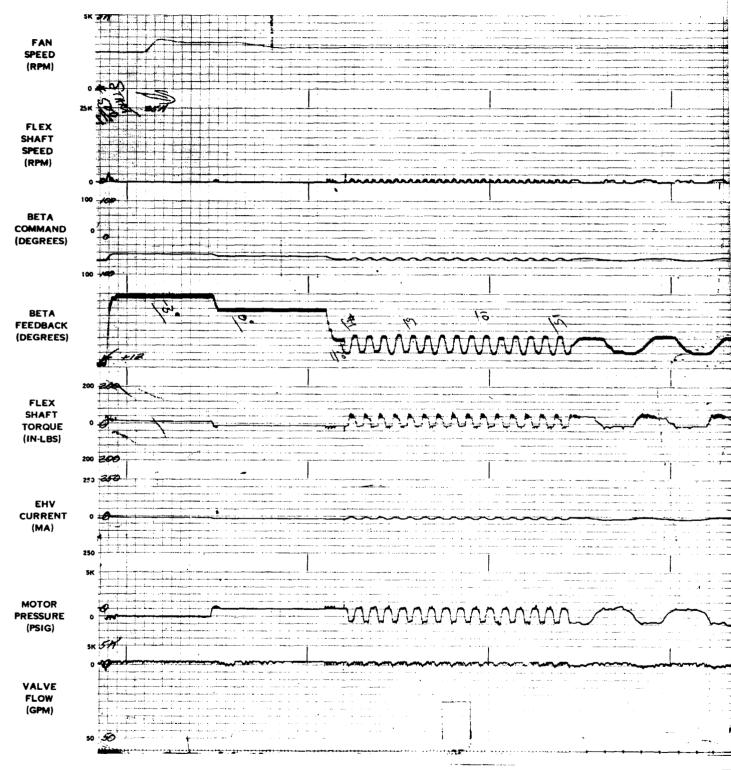








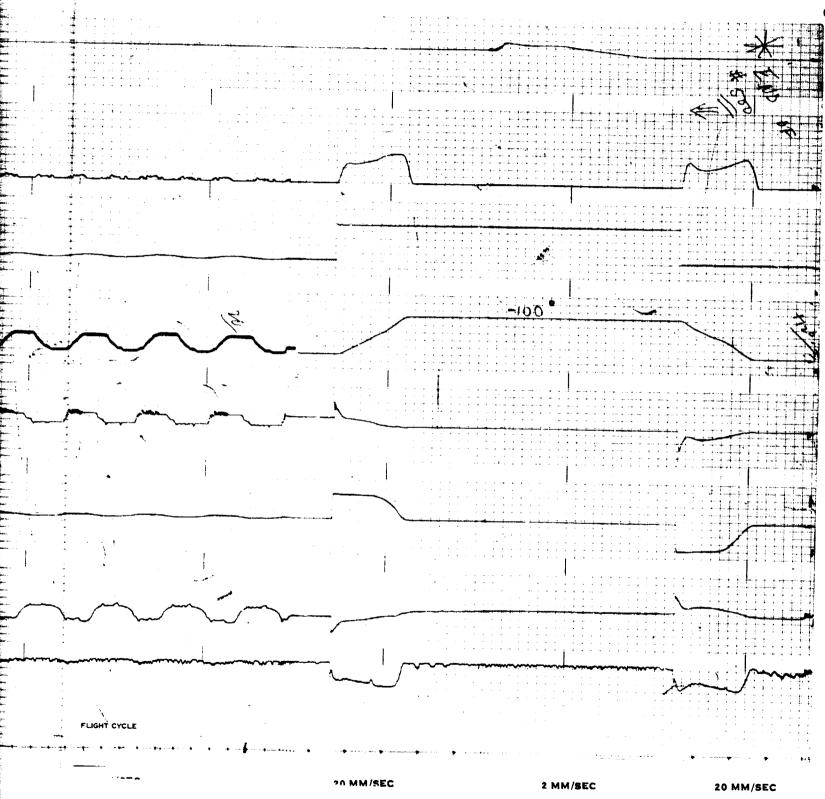




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APPENDIX D

LOG SHEETS

HSER 7002 OPERATORS . ENGINEERING LABORATORIES ORIGINAL PAGE IS 12 - 17 - 17 CO **10G OF 1EST** 1, 1, Z 202611 ; PART NO. 0.05=7 MAN OF TEST NO. 4/4 (CC) 71 L 1) 人过少分分子 0,00 SERIAL NO. 13milton Standard La Karrania -5K1 147 147 CA 27.7 1111 KKI 150 135 liii Linn, 0 15 in in its and 25 302560 011 0258 3015 75 30% CEDE 3000 35 392 30 i^{ω} 05 1375 J.E. 20% 20% 12.77 3.2% 1835 B 17:15 N CE 300 20 30 0 3:5 25.5 0 30 12.15 いので 2250 255 INFE OF TEST ↑ Silvio TIME W.P.L. NO. REMARKS. ON O

REFORT NO.

DATE ENGINEER **HSER 7002** CH. 10. 10.00 10 ENGINEERING LABORATORIES iciti. 1 E. 1. 2 10 PART NO. ひょうかがか 167.15 FULCITIONAL MAN OF TEST NO. 145/1971 よびがあ 121.14 せるでも 5.72 SERVE NO. Œ 10 1000 4.1.1 15 かい 7 11.00 73 15.35 47/10 WINDSOR LOCKS, COMMEDICUT . U.S.A.) 10 20 VY 015 2000 00 000 3:50 35 30 31 40 ļ 13.7 5.0 3/30 563 3:5 35 35 550 3.020 3 11/2 Tree T 13 3 17 4 55 O O 7.37 ンジン 2,22 53 3.1 220,2 ·124 TIME PEMARKS.

REPORT NO.

REPORT NO. HSER 7002 P LOG OF TEST ENGINEERING LABORATORIES Aprico PART NO. BCKT WINBALT MYKOUT PLAN OF TEST NO. SERIAL NO. Hamilton Standard VIBRATION Viet Hore 0 C \mathcal{C} O Ç Ø 4 O ~ OF POOR Q G-12.7 20 70 2 000 20 200 109-con-cosA RIM 3.50c المناكك. 1500 200 3000 1000 300 ± crass → W.P.I. NO. TYPE OF TEST -SF 1758 4/67 TIME PEMARKS MCE NO

OPERATORS PART NO. 7: 1: 10 / CE (D.C S.C. L. ENGINEERING LABORATORIES LOG OF TEST \$ 22 PT- 51 Brun 1.04.04.6) 1.1. Set alectora. 17 22. FLAN OF TEST NO. SERAL NO. Mamilton Standard C. S. AMERICAN STATE STA Veirs Veirs burs 30.01 5 1901 1.905 5.667 35 6 10 117 2.105 5:663 O 42.6 41.2 2 2.37 2.370 6.666 0 1.715 1.711 5.664 1.517 1.509 7.66 1.402 1.377 5.667 10 1.307 1.305 1.667 .813 .810 5.66¢ 15 1.04 1.09 5.63 533 :517: 618. 539: 102. 32. .033 5:59 153 1665 .3:42 5.666 131 131 565 LVOT CRUSKSTICH (Date 7.2) 13/15 1.87 21,0 <u>ج</u>ٰ 3 40 345 AME 12 11:5 25 ्र स العر 11 ंश 10 3, 20 <u>ن</u> 00ز ,es 7: 7 * .. * TIME NS 1758 4/67 REMARKS PAGE NO.

HSER 7002

FT ... **HSER 7002** OPERATORS DATE ENGINEERING LABORATORIES 1-11-20 LOG OF TEST chetyrial PART NO. PLAN OF TEST NO. 2.23 PT- 31 OF POOL とのと SERIAL NO. 4.35 1.35 1. 36.5 16.50 5.665 2.576 11514 5.065 14.4 1.683 5.665 2 red 2.003 5 365 2.63 2.45 5.665 1. 122.3.78 1.16.6 2170 1.157 5.005 7,850 1847 5.565 Die. 1.7.28 . 1.75 5.65 1.353 1.351 5.665 1.7.7. 25.00 1873 48 - Class Com 27.7.2 8.1.75 1777 Hamilton Standard 77 12.6.27 1527 4.25 - 4.6.6. 1.3. 1.3. 1.5.1 11.0 2 Š 10.00 11.70 15.5 157 27 (Z) 2 3 \. \. Si 0,0 7.1.4 MG NO. ¥ SizS W.P.I. NO. TIME MSF 1758 4/67 PAGE NO.

REPORT NO. ENGINEER NELS 6.1 **HSER 7002** OPERATORS H. . . . / Line, usen 1255 SEGULATION ON PULP - LUBE PACSS, YEADING ON G-ECO PSI GAUGE. ENGINEERING LABORATORIES PART NO. 7645 00-1 LOG OF TEST SERAL NO. 1222 17 - 31 - 1/2 / 1 Under To Washate Paess with Pross Pelier Valve in Hele L.M. 501 Hamilton Standard 19. 1- K 174 Nele :99 Little From Cheek 2.5 1.447 17.7. -7011-17 9 12.11.21 P.T. Lecon 70 .. 8,0 XY -9: 1 ŗ W.P.L. NO. MSF 1758 4767 . PE OF TEST TIME EMARKS. ONS

REPORT NO. 1350 **HSER 7002** 156 1320 ENGINEERING LABORATORIES List Rete wish James 2000 E. W. 10 LOG OF TEST 6-7 (63-AC) ANTEL SETTING 1: 15 AN OF REST NO. 222 PT- 31 PENA A. SERA NO. 222 PT- 31 PENA A. 00/ B ハリーソロ Hamilton Standard ORIGINAL PAGE B (C) 11/200 2000 2000 400 17 14.5 2000 H_{X} 1911Pel J. 1. 11. 12. 0 2 2 2 2 4378 4758 4.4 PAGE NO 129

Hamilton Standard Q HS 1738 6/67

ENGINEERING LABORATORIES LOG OF TEST

8 BIGNER P 11.1511.21.16. Ten'y T'eur OPERATORS 1 REPORT NO. DATE 11-11 23 **HSER 7002** Amic. (FORG 23) RAN OF TEST NO. 222FT 31 FOR A OPERATORS - SERVINO. -Sunt JUN S 751 h 103 dece decer 125 21.77 La- Crade Thin Iken S. C. C. 1 2 3 5.51. ¢. 1 3 11000 . 55 ; 9- 5007 57 - wast 10500 -6 INTER TOTAL LACKED į 1 1 1.1 ı ŧ. Actionede 75.7 1.160 1/160 2.100 1500 1100 TOWO CI-7
TOWN ON F. CM FALL C Q C C Time Trans Time 36.4. Units + Trist Mil TIME KMANS 130 PAGE NO.

DATE //-4.75 DOCUMER D 1: **HSER 7002** 7/1 07/ 81 ENGINEERING LABORATORIES THE OWEN TO SEE THE WASTER TO SEE TO LOG OF TEST Cucie Long ¢ 6.0 \$ 100 1 ١ 1 15 / D Wer ì 2500 2000 न्त्री क 17.50 17.0 AZT 1.C7 MAN NO

REPORT NO. 11.33 13.37 <u>.</u> 230 71 186 F - 1 - 1 - 1 V 12 . 1.7. **HSER 7002** - Lune DATE 11.21.75 Z II ---W 7 BACINETA D. OPPRATORS RAN ; ! 1.112 H. 2 ١ ŧ į 4 1 16 5 15 27 2744 **ENGINEERING LABORATORIES** * 0: 12 0: 4 0: 5 0: 18 115 00 112 27 **106 of 1EST** Cuta Lodo @ 120 601 124 22277-31 Ker A 100 101 24 72 CLUTCH WILLY 701 108 7 7 0 مر 400 171 エード 29 35 511 dre ! ? 7 Decepes 132 CI くさかと 6 ricken out oh 70 WIASKY TI. FLAN OF REST NO. Stapped 16.5 7.5.0 4 this way 3 7 2 SPEM NO. 7 # > 27.50 27.50 1900 2,50 2 2150 727 3 14/1 じんつ 1 Motor 007 しいじつし 42.75 001 100 100 00/ 730 910 00/ 0 となったった To or 4 2 2 ファマ 182 1000 120 2500 225 トンとの Mari Drive 200 Moin Drive 227 170 **lamilton Standard** 7 792 ナンジト 4 20 74. 2000 2,00 2500 2300 2500 25/10 ントって かいろうむ WHISTOR LOCKS, COMMETICAL . U.S.A. 7 5751 1013 36 109. C. 3.8. HOZ B G. E 1810 $ar{\mathcal{E}}'$ 18 JEST LIST 3011 C lë' H क्ष ह 기 は 0 0 -री ;î 4 15 :1 검 1 1,5 31 6.7 2 O C ¢ 0 C ٥ 2215 1:05 30.0 22.20 F. ... 33 14 72.15 Step 2250 4 525 100 とから 1011 5:21 T3.47. 1736 スンと TME. TYPE OF REST W.P.I. NO. 500 1103 173 L/S 2707 C10/ 132

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REPORT NO. 200 P. . . D. 5.4-112.97 9.70+ OPERATORS T. PALLET K. 6.45-112.9 T **HSER 7002** ENGINER TO CLILITY 2 SEVICE LOS VISIT NOVE WAST Perstr 7. 1000 13/cv [] No | CC NO 1 4.0 <u>-</u> 200 のな 4.0 ENGINEERING LABORATORIES HHY 1.30 600 73. PART NO. 7 (3 5 0 0 LOG OF TEST 10% 2.40 らいい KCV. H 132 930 ひゃんいっか TIPE OF TEST G. E. GC CCT ACTUSTICE FUNCTIONALTEST MANOFTEST NO. 222 PT-31 W.P.I. NO. 109-C03-432 B र ठ Parker 12 molas Noise Shoves 1170 09/ 1780 10/01 このとから 2000/75 1516 1216 19.5 lbs 2000185 į. 5 0 100 2000 150.00 000 001 > 11/1 フェスノーコ・モン 1/1/2/ 31:52 しいない 001 00/ 00 .amilton Standard 7 د ۱۹۰ 16110150 イル 1000 Yourer 401 154.1 2 (47) 人 /0% 2450 ۳^ر، کر \$18 \$1 MINDSOR LOCKS, CONNECTICUT . U.S.A. 109-CO2-402 B Tome Tome Time 17.10 K 덖 नु ० T 515 417 よび下 TME HSF 1758 4/67 EG NO PAGE NO 133

DATE 1/-36-75 الرائية 8.47 5.87 ENGINEER LIGHT ST **HSER 7002** W.Z. 7: OPERATORS 2300 PS1G ر خ <u>ت</u> J.; _ 1×1 000 to 2700 ENGINEERING LABORATORIES 103 (032) + 02 Bartunchon et 2 1 mm or rest no. 22 2 1 - 31 12 my A
sena no. 103 - 100 l mar no 100 l mar no. 100 ニメニ 101 103 10 69 16 LOG OF TEST 1-4-6 113 Klaxx 200 10 87 112 97 28 RAISED PHE STURE 6 12 1211 6.9 Servo Press 6.4.5 8 ن ن 23 4 1 2 102 1,01 92. 9 100 9 TA OCEN PS/G PS/ 6 PS/2 , BROAght <u>%</u> / 18 ļ 2000 18 100 KIARTS 2000 ST .amilton Standard 2000 1000 3070 186 100 2300 2000 ORIGINAL OF POOR PAGE CLANGE BLADEX IS CHANGE BLADE 100 001 100 101 5 001 Rig CVS KVM F35W 3670 172 2700 150 31/10 169 2700 0 17 31/00 WINDSOR LOCKS. CONNECTICUT . U.S.A. דוור קאישוני 10 1 LINABLE 3 3 15 11/1/ A SINS 2155 511 136 21.15 2305 531 to 34.1 134 2300 10 REMARKS. W P.I. NO. TIME HSF 1758 4/67 PAGE NO.

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30655 \$ 405 RPN ENGINEERING LABORATORIES MAN OF TEST NO. 2 2 2 8 T - 5/ 10 16 LOG OF TEST Kelland 100% 071 P700, 040 164 930 3/4/6 18 1240 960 103 101° . 96 .50/ Stopped to wspect Prop, linds ele 1200 .4// 1.30 d 30/5 6 1.0 118 ... 3/100/2 8 Ç 8 00 6/ 3400 1562 001 8:SK 01/2 185 3516 PS/2 Tamilton Standard 33757 3042 237 53% 7000 8 Vet State Ferenson 2st 001 110 00/ 3/04 220 1100 153/572.5 100 3409 223 100 00/ 3068 225 106 3572 736 100 3411 236 100 2703/273 LYN Flow 5070 202 2702 229 KIS 2703 WINDSOR LOCKS, COMMECTICUT . U.S.A. 0

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. text us as ð OPERATORS J. A D. V. C. J. 21.2 0 ・ハイ ø DATE 1/2-7 SHEFT SA M **HSER 7002** Ş 4 M , ENGINEER . イオア ा <u>ं</u> j Circe 4 K ENGINEERING LABORATORIES 7.7 2.67 701 5.17 Cir. 58 107 130 10 Proposed#1 EHIN Paul Zobrontewith #2 93. 136. 166. 166 0 75300 LOG OF TEST 9 No. 1090 110 . " Chall 5 410 1.7 N PART NO. 341 22 N 222 47-31 N. N. 93.1.35 le! 1.00/ Ų 163 S C a.12.15 0 0 Ö 10% Leco roes PLAN OF TEST NO. PSIG 18516 PSIG 100017.5 Survey Ord 12:0 356017.5 42 7.17 SERIAL NO. 0 1015 Thank CLC & MAIN CHE STUTION 00/2/00/ Brid Libe Lube EHV 3K.00 lamilton Standard ____U___ 11/1/1/ 3230 21.0 3000 is on Hewlette Nickard F05-005-140 Fre London Wal Test MACIN 100 100 176 102 100 158 100 0 1916 350 53 MAT-DOWN 4000 3017 198 97100 160 - 4:3 V 1104.XO 5.1977 53/3 3700 131.4 7317 2 1 to 11/2 3318 JUNI 20 to 10 to 11 17 4.5.4 (1124/1112 E 1001 110 1. <u>~</u> |%| 11/2 1 17 M ان إن 1 14 1; 15 D ON DAY 6 VMTS ★ 13000 W.P.I. NO. 1846 135 346 1000 1100 HSF 1758 4/67 TIME 17% 0 11.22 ? PEMARKS. 6.0 PAGE NO.

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ENGINEERING LABORATORIES LOG OF TEST

100×10 54 234 12 1/3 1/2 2.31 3/1/ OPERATORS ////// **HSER 7002** 1/425 3 j 1111 1.1 1 2 G 012/ ートに外つ 1180112 168 163 1610 96. 113. 16. 168. 400 113.1110 12. 94. 119. 112. 13. 1. PART NO. 2000 06.67 14. 137 3 186 (M. 10) 162 92. 000 5.0 Jener 108 ノイノ 16 ı ソノイン 7/11/1/ 2 6 9.11.11.P C.W. This Tiri 3800 18.0 100 30% 170 3010/120 2800 175 7130 17.0 3130 17.5 1120 17.0 SERIAL NO. 1. 1. 15 1 1000 1.24.70 100 00 100 Ú 0/ 100 01/ 100 Down \$ 4 30C8 115 5 3/65/3/0 7 07/1/1/1 Ü 36.68 179 1111 1067511 3700 166 4. 2. 2. 2. 3. B 3065191 17.11 Mer 17.4 ر از از Nest 202 1.27 当く 1 2150 2/2 11/2 ジジ 3/3 10 " 20 19. 7 5 611 4 5135 11/11 1116 12/2 11:5 REMAPKS W.P.L. NO. PAGE NO.

8 REPORT NO. ì 1 11: m 130 o **∴** I 12 SKET // **HSER 7002** 7 OPERATORS 1 DATE 11'-11/10/2 3 ENGINEER 11.13 17 15 J 10000 • " ú ENGINEERING LABORATORIES 60.11 イイン 17 .011 . 4 106. 167 12 LOG OF TEST 1 * 1 1 . . ナイベイ ー・インバン 1330 111: デオイナン 100 96. 1340 1130 1111.116" 3 5/11/1/2 PART NO. 5107540 1000 11 138 1 6 931 · /. 100 11 11111 22. 103 "VALE 4 Tust "were 1755 MAN OF TEST NO. 3120120 1014 Filing Jey 100 3130 17.0 Shi 3100 17.0 7:1 21.20 17.0 160 3000 170 SERIAL NO. 17.0 Jan Set familton Standard 11/1 Serie 100 3000 100 3780 100 30cc 17 15/1/2 16.13. EX C 100 100 8 2/2 X N The Make son feel han Ale VC012 3 5.1. 1114 1:01 11 209 1277 16:81.305 210 128 chinking 172 300 11/08/162 3 1118111 Trig イルノングラ 13/16 2007 1/10 3000 Cederal 3760 4.63.92.0 1/0/ CALANO 3768 WINDSOR LOCKS, CONNECTICUT . U.S.A. (0/0) とシー 1111 3 \il 1 13 انإ 0 1 1: 1 3 17 1000 STATE OF TEST CO. 3 زا 34 0 i VZ1S ₩ 73.8 7/1. 2 7. 17 W.P.I. NO. TIME 7.10 13 HSF 1758 4/67 Ċ 4/11 2356 1 REMARKS 17. Ç PAGE NO.

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0 12.37 02.24 **HSER 7002** SMEET A. 3 OPERATORS 710 12.6.5 d N \ \ \ ENGINEERING LABORATORIES 119.108. 1.501 اراي Netton 180 110 J. Kokeck LOG OF TEST FLAN OF TEST NO. 2-22-21-21 17.7 1. 1. 1111111 113.108. 95. 100.110. × × <u>ွှ</u> Cheretor Conservator PART NO ** (" " The said 11.7.1 111 76.115 730 2 2,5 "Mike-Milk . 50/ 1001 2/4/2 2 ./0/ ` 12.0 142 × 45.45 100 3ces 170 3130 17.0 3006 207 100 3000 170 FNITTON 2,02/15/ 1100 |3400/19.5 100 3/00 17.5 Si wast 3001 4/1 611/2 可然%/1/37 111 300 11-29-75 1 14 3063 219 100 3100 in a site It having hemply as 15111 100 Drune 16:41 00 3067 302 1704 arivino, 一人と ダジ Crest Carrie 181.41 30.05 405 300 1200,000 1930 11ch オルンジ 71.7 Correctel Twe On 0.0 11.5 1 WINDSOR LOCKS, CONNECTICUT . U.S.A. 13 1 * TIPE OF TEST VMTS ₩ W.P.I. NO. 45F 175B 4/67 PAGE NO. 139

SHEET STORY 11.71.47 65.53 05:10 **HSER 7002** Lite Mile 0.0 0.1 0.0 CPERATORS - waters Vectority 110 62 105 1150 106 108 10.7 113 [13] [11] [0.2] 103° 1103° 16° 104° 6.2 ENGINEERING LABORATORIES SERVINO SERVINO SERVINO DO DO PARTINO latal to. 22,10,c/e LOG OF TEST 1-374 CYEVE Helent Store 2 Tisk . 100 1132 COLORAL 1866 FACILITIES ASKILL Cohas Lited 1814 156 941 axide of 56x121 ./0/ 001 Comments 2151 Fre El Codlor 113,101 Sus cel 1.01 10.1 Coshillet 2011/5/11/4/11/21/21/2/2/ RAILED SOUPEY ALESSAG DURING Copy Ched 2 pms Flight cycle & 11 0018 001 507 0017 170 1111 11 202 156 100 5000 125 2 103 D. 16 100 3100 17 Flisht alde 2065 222 100 5100 17 206/220/100 3/00 17 3410217 100 3100 17 2707 170 100 3000 18 3067 204 100 2000 17 Hamilton Standard 2500 365 171 100 RYN 1.70.V 34 tel 2/3 1713 -6654 See 455 1111 ひっとしい 17 47 7 11 Grats ▼ // 140 NO TEST 750 HSF 1758 4/67 201 REMARKS PAGE NO

OPERATORS 5- EGG STATE OF 10.51 16 70 103 1665A To the 102 118 1111 112 02 0 3 70/09 **HSER 7002** DATE / - 2 5-75 0.0 112 0.1 0.2 0 112 0.2 0.2 24/11/24/10 with your tok-twelfecture 92' 120' 112' 112' 02 113° b.1 25/1/cy/c/e 125 115 111 01 113 111 0.1 Completed 27th Flight Hycly & Show 1254 Cycle 25-12 Fhiltwood Estrellzeth excle 26-1/2 Flisht frole 15 Butel 2 2 May cole ENGINEERING LABORATORIES LOG OF TEST 114. 115. CCC - 10 - B. CALLINIA CEST RAN OF TEST NO. 2 2 2 8T- 7 1 1 20m/2/2/23x2/7/1/01/1/02/1/ 5 yested इंडीक्टी 2 the Flight Exclise State 6 5 toxt 101.115 98. 98. 104~ 2/4. J. 2/3. 1/4 Flight Cyd Co. 18/6/6/ 2 \$ 3/ Flight 15/6/ . 301/ Flishtore 3/29 8 42 SMI 210 100 29m 12 CIL 3063 213 100 3025 16.5 3066 213 100 2900 16.5 3066 214 100 3000 16.5 5.2/10002 245 211 100 2535 3063 211 1106 297417 · ě Hamilton Standard man granger man American Constitution 1884. 3066|214 |100 (10 mo) Com//e/ed Com Meded WINDSOR LOCKS, CONNECTICUT . U.S.A. 1320 206 9 19 223 209 9 24 1000 W.P.I. NO. TIPE OF TEST HSF 1758 4:67 TAGE NO

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REPORT NO 7.7 1 1.6.7 12-5-13 OF 7. (6.3) 11. **HSER 7002** ENGRATER VIVE ST. • 108 3+ WRTED TYCKE #100 L.XI 1,07 C40/00/043 SHAKTOB Cycle 108 Pulche ENGINEERING LABORATORIES 7/10 22.4. KY STAKIOD WILL " 107 1 7 61. 000 101 50 17. ... 22 DI- 31 6 1. 13 **106 OF TEST** 105 101 5 tak 100 102 70 137 165 3 ETHRITED 1. STARTCO 187 16/15 811 1.6 125 7 <u>.</u> 4.7. ゲ PART NO. H 17. 7.6 ٠, 10.7 100 11: 2/ * V 0/ 100 Cheke "Los Ļ Ż: 6.4. 10 **†**: Cycle C./c.1E CUCLE FUNCTIONAL MAN OF TEST NO. 7 15 6 + 1.911 Cych 11.9hr Cycle رد مرد न्धः द je / ر در، 3555 8/ *S* ξ SERIAL NO. Trist 45.30 ئارىر amilton Standard いこうじ 0,00 مُددَدَ امتزدز 35.50 7 7 11/11/1 PSIC ナンコカナ ر. تر 1310 7.50 5 , , ,3 150 1 1 7 7 b 10:12/cteb De 2 2 60 70 Carrolera 161,016 ta. 13 ... 3063 400 261 ٦. 1. V. 1. 1. V. V 1.61 ?! 000 " 5 Cor11/ct. 0 CES-ALZB 3000 9 3000 •) かいのの 1 WINDSOR LOCKS, CONNECTICUT + U.S.A. 11.7 4.7 3 151 151 151 6.8. 8.37 W. 813 1:33 1017 (27 - 1 15 1 33 6 C. 1. 2 3/3/5 12013 119 12.5 the Oriest CNITS # 2000 2000 TIME W.P.I. NO. MSF 1758 4 67 PAGE NO 154

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OF POOR QUALITY

ENGINEER 17 1 1.00 REPORT NO CLEBERT JULIA JULIA TOTAL MESTE STEER AND STORY OPERATORS (**HSER 7002** PANTINO 1: 15 CC E.U.S. 1. 63 T. 1944 . 3144 DATE ENGINEERING LABORATORIES 9. 11. 2. 21. 2. 21. 2. 22 4.26 16.6.5 11 C.S. **LOG OF TEST** 4 100 Total Total Kind Read Substitution 16.65 16.61 1. Eng 3.65 11. 3 511 27 32 19 34 515 3700 1846 1/2 cap 10.5 17.5 Jamilton Standard 11 35'0 The Garage THEF COLIC #207 1 KO CKL1 207 100 11 11 12 CA 100, 114) 11.11 mire Corre 210 39 350c 11. Chil 211 1. C. 1. 212 14. CAN 213 2700 1815 97 3500 1. 535 20m 3166 6561 17:26 27.0 . 145 31763. 1 ~...... 11.7 1105 HOF 1758 4767 REMAPKS. PAGE NO

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8 MILS Pleg. Vib. Blade Horiz, Angle **HSER 7002** ٠ ١ J, 0 Q OPERATORS / JULIAL SHEET 32 ENGINEER Z Vert. MILS ENGINEERING LABORATORIES Temp. (K) Shroud 1057 #2 16.4 1071 LOG OF TEST PART NO. 76750 Lube: 12%0 0:1 116 11 Rev. Supply 122 #3 11.11 1111 137.7 1:75 12. 226. 122 Ò Clutch 222PT-31 1.81 720 7 PLAN OF TEST NO. PSIG 6/B 0ii SERIAL NO. . 1000 Light Wife The Supply The state of 3,10 d375 PSI 11.11 L W 11 1 W 1 Hamilton Standard 9 PSIG Lube 0i1 6 3 S. S.C.E. G.E. QCSEE ACTUATOR ENDURANCE TEST Lube Flow . S 5 5 TEG 103 6561 13. · 16.2. B Rig R.P.M. 7.0 OF FOOR 13.10E 3700 3700 237 3.16 7 (00) 300 ×108 35.65 PAGE IS Para 1.1 1.7 11.7 11 1.7 11.1 111 1.7 1.7 1:1 7 Ł 13 ine Total . ţ 11 <u>-</u>. 1 21 71 Z 7 ټ 0 ; Tine 01 $t_{\mathcal{G}}$ \mathcal{V} $\frac{1}{e^{\epsilon}}$ <u>"</u>] \geq 1 1 U_{ϵ} 7 51 6-7 ا.د . REMARKS **★ SINO** 0.5 TIME TYPE OF TEST 7 . 1. T • HSF 1758 4/67 WPL NO - } ¥ 02.03 PAGE NO

'Hamilton Standard MSF 1758 4/67

ENGINEERING LABORATORIES LOG OF TEST

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ENGINEER D. List With Con-Angle Blade OPERATORS L'ABLIETE 2 12 16 75 Horiz. r. Vert. MILS DATE #5 Shroud Temp. PART NO. 767500 ٠, Lube 0:1 #4 ğ Supply #3 EHV Clutch 222PT-31 7 Vidur Chies PLAN OF TEST NO. PSIG 6/8 0i1 SERIAL NO. PSIG PSICE EHV Supp Jy Lube Oil 11 62 PKETED 32.8 G.E. QCSEE ACTUATOR ENDURANCE TEST S £ 5673. . Kr 91. Fube Se Ac. A Rig R.P.M. 27160 301: 7,700 Para 11.7 11.7 1:1 1.1.7 100 - 601 Total 130 111 Time ر در در ا زار م اج 29 أيز زا Test Time 13 1 21 71 **C-**2 (> 07.7/ TYPE OF TEST W.P.I. NO. 0.77 TIME REMARKS 615 2 170

HSER 7002

PAGE NO.

REPORT NO. ENGINER D. L. H. H. 7,5 OPERATORS F. . LELETE **HSER 7002** ENGINEERING LABORATORIES Tirall 5. 2. 2. LOG OF TEST 金三とのなー」なって 144 757 1 2 1 CC LLIQUENCY DESCONSE TOST MAN OF TESTINO. C 3520 1,1,1,2, 3:10 4 5 - C 1 2 2 14.6.2 21.08 - 8.60 97 VYE 22 11.01. VE 11 2.1.C. V 0 31 2.51 05 17 16 17 17 17 ORIGINAL PAGE OF POOR QUALITY Title Time | Species more G-7 TIME W P.I. NO. REWARKS 1158 1758 4767 PAGE NO 171

REPORT NO. SMEET - CO Blace 103/ the farmer of n Horiz. Angle **HSER 7002** O Ú ١ OPERATORS ____ ENCINEER • Vert. 17 MILS DATE ENGINEERING LABORATORIES 112.7 13.2 113.9 Temp. 11.12 101.0 Shroud 1116 113 LOG OF TEST 1139 18.2 167 シング PART NO. 767500 10 Lube <u>:</u> 7 Rev. A 130.2 Supply 27.5 12.3 11.11 101 #3 -Env 102 101.8 10% 1.40 Clutch 222PT-31 **"** FLAN OF TEST NO. C 7. 71 Line Cycle 253 Ent Cyde 25% 71.1.1. 170 Cyde 252 **PS16** 2 . T. 2. 6/8 0:1 ě. SERIAL NO. PSIC 200 200 Supply 11/1/11/11 <u>.</u> どこし PSIG . Lube Oi 1 50 99 B 66 00 12 '--- 'Jamilton Standard 85 \<u>\</u> 1500 G.E. UCSEE ACTUATOR ENDURANCE TEST Lube Flow .6.11. .85 `. `. 18. 3716 XWX 3/2 スジス 2408 111. ムンと 077 20% MINDSOR LOCKS, CONNECTICUT . U.S.A. 17.7 1.7 Para 1:1 1.7 11.7 1 11, - ' 11 री Ξ, 1 ञ Total 51 =1 ارًا = ķ 1 ., 3 : 17,31 67 ķ 5 ,5.I Test Time ij = -1 **C-**2 KON 77.37 -÷ ; ; REMARKS 20% **★** 5125 4 TYPE OF TEST W.P.I. NO. TIME 49/1 95/1 45H 0 2 2 2 3 i 172

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ENGINEERING LABORATORIES LOG OF TEST

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ENGENEER 17 15 17 17 Elade | 1001 3 7 100% 0 % 100 1.5. Wm // 1.4 1, 1, 1, ຸກ 4 (0) C Ü Horiz. MILS • • OPERATORS 411 Vert. Temp. 1697 5/11 1073 Shroud 111.1 116.3 2:11 108.1 117.6 ~ [ンジン 112.5 1000 111 **#**2 13 10.000 107.2 スペン 1678 110.5 PART NO. 767500 111.3 113.0 ? 11.7 1:8:1 17.1 2/1 Lube 11.11 1001 Oi) Rev. A Supply 1465 135.5 1.5.5. 1:11.6 1750 105.0 1.10.7 7//1 1. 25 1430 1111 1:1:7 #3 ETIV 1136 5 100.7 103.2 Clutch 11. /S - / 222PT-31 101 0211 1010 1.8% 11.11 7 MAN STAN PLAN OF TEST NO. PS 1G 1 2/2 (3) 8/10/ 8172 81,2 5,5 27 / 5.3 31:0 SERIAL MO. 1 Supply PSIG PSICE 36.30 150 10 1 mg 0 3.750 11111 2 2 Jasen! 3.11 196.12 Lube Oil 60 25 50 66 66 16 6 4 ? G.E. UCSEE ACTUATOR ENDURANCE TEST . 85 75.55 Fibe Fige نن مر ۱.م ین بها ند 35 , C ٠<u>٠</u> 33. 3 55, .55, FAMILY OF Rig R.P.M. 2100 20.65 27.6 2770 3000 30/0 3716 3700 3408 80% 2700 2700 1 603- 402 15 Para 1.2 1. 11 -1:5 1,752. 81 Total :31 Time , 31 13 21 51 ij. Ý١ ... 23 Test 37 Lime is. -1 •.. ائد 15 = 21.15 4 5:25 21.70 7000 71.17 600 11:10 TIME TYPE OF TEST 2002 W P.I. NO.

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HSER 7002

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"amilton Standard ---PSF 1758 A. 63

Vib. Blase Horiz, Angle ENCHALE TO 1 00 100 + *31 **HSER 7002** 0 0 ZII S OPERATORS Vert. 2 ENGINEERING LABORATORIES Shroud Temp. 11.5 200 110 > 110.0 **£**2 111.3 LOG OF TEST PART NO. _767500 11.7 1110 63 171 7111 111.7 16.78 1111 011 7111 Rev. A Supply 11:11 1111.5 1.1. 1:75 1.15.0 148.1 1.18.2 17.7.8 1.76.8 2 Clutch 222PT-31 600 101.2 RAN OF TEST NO. 8/12 30 PS 16 11. 2. 2. 3,00 (33/18) 28.3.5 SERVAL NO. 250 75. 7 K 10 Cycle 250 -280 3 Supply Stancy-16 PSICA 10 Cycle 3 7363 5000 100 12K3 1×45 1363 6361 12/12/10 PSIG dug ub 40. 64 -£ 5.4 Lube Oil 66 00 60 દ Z E 5 Ę G.E. UCSEE ACTUATOR ENDURANCE TEST 1.5 X Flow , 50 85 30 120 R. P.K. 2665 2200 3.708 \$ 7/1. 8 3065 300 30/1 1100 3.705 3.105 3408 2000 MINDSOR SOCKS, COMMECTICUT . U.S.A. (C3 412 B Para. 1:1 1.7 1.7 7 ·;` 1 4.7 2 \$1 -131 1,70 18 Test | Total الثر श 和 18. 18. 31 12 Time 3 3 31 10 ? 2 10.11 4 5195 1:12 1.22 02 ST 174 0200 ON Tak 11.AE RVARCS. PAGE NO

ام Wils Deg. Vib. Blase Horiz, Angle ENGINEER D. 1-15-1/11/1 1700 100 SHEET 1/2 C 100 **HSER 7002** (+) OPERATORS S.1/4 .17 12 11. O 1 0 Ć ı DATE 13 15 MILS Vert. Vib. ENGINEERING LABORATORIES 1699 767 1691 165.4 116.3 1116.7 6621 111.9 Temp. 126.4 Shroud Ġ #5 011 109 111 LOG OF TEST 1095 16% 2777 16/2 1631 1075 50 121 811 6711 , do いつと PART NO. 767500 Lube 0i1 168 Rev. A Supply 244 7.66.5 1373 100 147.3 61111 1.16.3 011.1 1.41.5 0111 197 111 ## ** *Y::* 11.0.5 95.6 6:221 8% 11.7 Clutch 97.5 666 ÷. مد 222PT-31 2 # 3 300 PLAN OF TEST NO. 8/9/2 238 133. 18. 18/500 1374/8 18/16: 18/8/ 8/15-1 PSIG 6/8 0i1 Š SINT CYCI CARS 360/8 Š 11 0 100 000 0.5/5 05/20 05/30 10/30 000 37.50 PSICE EHV Supply TOTO COMPAS 5+46.1 Spel 1 10 chite 15 START CO ָ בּיְ 0 一 66 1 June 66 FNO Just (6) 194 856.7 PSIG 99 -11 5 Lube Oil 65 19 66 66 66 66 ORIGI li 'Hamilton Standard 90 1°2 200 1,33 156 G.E. CCSEE ACTUATOR ENDURANCE TEST , 55. PAGE A 15 20 12 .85 \% '% *ا*ر ا Lube Flow ندن. دیمر 3 Sing A Rig R.P.M. 2700 3068 31/08 3708 3068 2700 2700 1700 8908 3708 4700 3700 2700 WINDSOR LOCKS, CONNECTICUT . U.S.A. インドー Para 1:1 1:1 \ \ ! 1.7 *'* 1 11 11 11 Line 2 N. 281 35.24 Total 15 1 3 53 35 22, 3 , , 3 70 \tilde{S} Test Time 91 انځ 13 \\\ = 11 Ĭ 2-5 3 いへいり 35 1000 4 27.5 **★ 51:35** 2000 REMARKS: 100 200 5/2 0770 6200 2000 TIME TYPE OF TEST W.P.L. NO. 110 HSF 1758 4/07 8 PAGE NO 175

MILS Deo. Vib. Blace Horiz. Angle MET 21 05 PATHO ZOFESO . 11. OPERATORS / COLL . 1 **HSER 7002** 0 ENGINEER DATE Temp. Wert. Shroud | Vib. ٠ . r 1 ` ENGINEERING LABORATORIES 17.3 17.7 10 1110 1:12 à **10G OF TEST** Lube **5**# 1.11 P.CV. A Supply #3 112 117 Clutch 222PT-31 .3 : 11 ~ / // 71: PLAN OF TEST NO. 1 4.6 117 11.41. St 1 Suce 1 318 PSIG . . 6/B 0i1 1: Jak Jak 1 . SERIAL NO. Lube En. 1000 PSIG PSICE 37.80 1000 11111111 21/10 Hamilton Standard 34.34 61 " コンシ 3.E. COSEE ACTUATOR ENDURANCE TEST Flow 5150 > R. P. 9. 1111 4); Fire Para 1.7 , 1: 111 1:1 Time Time :_ 2 2 17 . 1 ,.! • 6-7 VN:3 ₩ 1231 90 EEST ٠,٠, TIME -20 1738 4787 REMARKS. 2 2 2 176 á 0. Ď ζ^ Ì 1 PAGE NO

#5 TILS MILS Ce. Shroud Vib. Vib. Blade Temp. Vert. Horiz. Angle **HSER 7002** ENGINEER D. DATE 12 1/2 S ENGINEERING LASORATORIES ٤, LOG OF TEST MT NO 763500 Lute Pev. A Supply 1120 Clutch 222PT-31 PLAN OF TEST NO. 366 Lube ERV 6/8 0il Supply 0il 3.6. 1 17. . SEMAL NO. 17. 11.1.1.11 PSIG PSICE 11.4 . 11.1. : had any *** *** 11/2 10 11/1 . . . \$ 3 · 8 12 11 Hamilton Standard S.E. COSEE ACTUATOR ENDURANCE TEST W.F.I NO. Lube Flow 9561 1.5. Tire Para R.P.M. 8 140 1 -1 21 7 -1 170 ? 1 2-5 10% tO 801 SEAL SE 23/4 65/ . 1.-PÁĞL NO

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Hamilton Standard

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ENGINEERING LABORATORIES LOG OF TEST

SHEFT REPORT NO. Blade Horiz. Angle 0/11 · ~/* HSER 7002 11 11. OPENATORS /// //// Ü Vib. X á Ġ Ų ENCINEER Vert. × 1 #5 Shroud 13/01 Temp. Jul. 13 1 11 1 Ç ٥ PART NO. 767500 Lube Oil ø 130 5 CX #4 1, 3 11 Rev. A Supply 12:21 #3 ERIV à 40 11110 1110 Clutch 222PT~31 11110 1134 17 # PLAN OF TEST NO. PS:6 100 33 1. 3.3.4 1326 1111 1330 376 0110 11. 1/1 SERIAL NO. 1. PSICE Supply 1.7.6 11/11 27.0 111/1. 0.7.0 37.76 £i⊀ 3750 11sh 1111 0.220 111/2 inh. 1111 11/1/2 111/ 0 11/1 PSIG 11011 ÷ 11.11. Jun! Lube Oil 27 34413 12 0 14.0 11 G.E. GCSEE ACTUATOR ENDURANCE TEST Lube Flow 1.1.1 15/1 3118 177 1/2 1110011 Rig R.P.M. بر 118 يرويل 160 6.76. Fara 1 1.1 1 1. 11 Total Tine 1777 77.72 17 Take 13/32 11/2 11 Test Time 11111 6. **★** SLIND 12.50 110 REMARKS. TYPE OF TEST W.P.I. NO. 1110 TIME 1.1.1 PACE NO. 178

Vib. Blade Noriz. Angle Deg. ĩ 153 **HSER 7002 >** 11 Ŋ • MILS J Z 4 く DATE / D. OPERATORS ENGINEER Yert. MILS . T ` `, ENGINEERING LABORATORIES #5 Shroud Temp. 9 J 1 65 1 7 LOG OF TEST PART NO. 767500 lube Oil 9 4 ø ù 7.0% ₽# 1 18 5 (1 Pev. A Supply 11 1.77 ý 田 #3 1 10% 1 . Clutch 222PT-31 1101 6 <u>[</u># 1 PLAN OF TEST NO. 37.6. 6/B 0i1 PSIG 6/2/3 11. 377 1.24 10 120 1311 350 4.6 1 10.31 7 1:2 7 1,10 SERAL NO. Supply PSIG PSIC 11.17 0.7. 21.11 3,00 27.7 37.0 11.1. 11111 13.75 1111 ハシバ 11211 4411 11/11 111/1 ::/ 100 1111 11/11/2/11/11 11/11 Lab along Lube 011 \$1.00m 11.0.7 Start. 1 11 31 1.00 ゾルル h Lj 0-41 111/11 2 5% 140 1 1 1 0 Ü Hamilton Standard G.E. UCSEE ACTUATOR ENDURANCE TEST Lube Flow 1.2 - 16.7 - 01.313 19 111 Rig. R.P.M. 11 311 1/2 108 1110 WINDSOR LOCKS, CONNECTICUT . U.S.A. Para 1.77 1 1.11 1.12 3511 117 Total 37.11 176 (5) 17.6 Тіте 17/1 1/32 Test Time 1 3 6-7 **▼ STAU** 13.11 27.50 ۲ TYPE OF TEST TIME W.P.I. NO. GEMARKS. HSF 1758 4/67 सु 1 PAGE NO. ORIGINAL PAGE IS OF POOR QUALITY 179

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Hamilton Standard HSF 1758 &/67

LOG OF TEST

19 SHEET // 6/ MILS Deg. Vib. Blade REPORT NO. Shroud Vib. Vib. Blade Temp. Vert. Horiz. Angle ONTRATORS 17.22.17 1 <u>, 2</u> ٠. را * 11/2 **HSER 7002** 1 ١ ્યું 1 ٠ ن • MILS DATE " ENGINEERING LABORATORIES This 10 74: 10 ĺ, 1. 1 17 #3 #4 LHV Lube Supply Gil PART NO. 767500 10 ę ام (مر 1. N. Rev. A 1 G_{2}^{2} Supply 4 132 12×1 (X) 17 Clutch 222PT-33 ナナ • _# 1 1 ; ! 11 7 PLAN OF TEST NO. PSIG 77 小, 1. 1.44 in Ja 6/8 0i1 1/2 1/4 79 346 11 5 نور ان 676 ~ ~ VO. 41 SERIAL MO. 18.00 16 11 34.0 ? // ~ Sink will soll Supply PSIG PSIC 37.10 11.115 3111 11.849 eyen 11.115 9 all 11/10 11.11 ĒΉ 11/ 11111 001 17.0 1:438-111. 11/1 Lube Oil inni Soul 1.8.0 1 12 4111 1000 1 110 11 11.11 1000 G.E. CCSEE ACTUATOR ENDURANCE TEST Lube Flow 0 177 162 348 110 101 X 101 Rig R.P.M. IX. 1100 110 1:1 1 1. 1 1 Ting Total 15 19 700 100% 出 1 Ž. 1 Test 17:51 お谷 1111 沙公 11.11 2-5 UNITS # Ş TYPE OF REST V.P.I. NO. 3 TIME REMAPKS 121 7 17 PAGE NO

Yamilton Standard HSF 1758 4/67

LOG OF TEST

DATE SHEET ALE OF Vib. Blade Horiz. Angle REPORT NO. ・ヘメ 5 7.x 1000 **HSER 7002** MILS × Vert. ENGINEERING LABORATORIES Tenp. #5 Shroud 0 i N 1 PART NO. 767500 Lube ₽# : Supply #3 EHV 13:27 110 0/2 Clutch | 222PT-31 0711 113.0 # 15 PLA:4 OF TEST NO. PSIG 7 6/B 0i1 21. 12 シング , SFRIAL NO. 10 PSIC EHV Supaly 1.20 27.5 111/ 1243 Cycsi 270 with rieu 2%,0 11/10 Nick 17/10 ンメイス 7011 PSIG Lube Oil 357 7 4 / 2. Mis 2417 420 110 dri 32% 63 1. 620 G.E. QCSEE ACTUATOR ENDURANCE TEST Lube Flow 110 201 Rig R. P.M. 8.7% 2110 Para 1. 1. 1 _Time_ 17.17 Total 13 15 1/67 沙河 1 Time **Test** 120.2 2) 13' **6-7** \ \ == ▼ Szizo TIME TYPE OF TEST W.P.I. NO. PEMARKS. ON US 189E NO.

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Vib. Blade Horiz. Angle ENGINEER 11 " 11 1 " ... DATE 12 - 15 - 7 5 MILS OPERATORS ' 11.00.1... Vib. Vert. MILS 1 ENGINEERING LABORATORIES 15.5 523 Temp. Shroud ,,,, **¥**2 1, LOG OF TEST PART NO. 767500 (); ; ; Lube Oil #4 Ġ Rev. A Supply バガ ٠, ر-#3 EHA : 7. 20. 0 % Clutch 111.2 1.163 222PT-31 **-**5.2. <u>ن</u> بر 4 326 3 7.17 1.1.W PLAN OF TEST NO. 2.17 11.11 PS1G C. KLIE . rur 1 NA (11) Find Correct 'we' 17.0 6/8 0i1 11:1 1131) 4 ... SEMAL NO. PSIC EHV Supply 14. rate Vere . N. 3.150 °, 20 Sicer 244 Cione. 0 1 7 ב ב 7 1-1 C 21.12 2115 271 SV 17011 250 : PSIG Lube Oil 600 7 07 65 Hamilton Standard • 27.2 40.3. 2000 G.E. QCSEE ACTUATOR ENDURANCE TEST . 5 5.Th Lube Flow Rig R.P.M. 3.69 2700 < 7co 1/67 10.27 11.23 WINDSOR LOCKS, CONNECTICUT • U.S.A. Para 1:1 1:1 Ņ. Con 2 -Total 14 Line 1 : × 7 1. 1. 11 7 : Test Time ž 21 6-7 `. 11/11 **★ 2175** TIME ~ 10, FID 0 TYPE OF TEST W.P.I. NO. . -Ų. 2 0 3 de 182 182

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REPORT NO. 8 Deg. Blade Angle 'n **HSER 7002** SHEET Ho iz. OPERATORS ENGINEER Vert. MILS ENGINEERING LABORATORIES #5 Shroud Temp. 176 LOG OF TEST PART NO. 767500 Lube Oil Rev. A Supply #3 EHV Clutch ۱, 222PT-31 F 7:17 e. 313 115 a ii 1. 2. . ÷ 3. 3 × PLAN OF TEST NO. SERIAL NO. PSIG 13 6/8 0i1 1111 11.1 : 1111 : : PSIC EHV Supply 541 . 140.0 (S) (S) 0.1 Hamilton Standard 000 0041 34.50 Ç . . . $\langle \cdot \rangle$ PSIG Lube Oil 29 . 8258 1.7675 G.E. GCSEE ACTUATOR ENDURANCE TEST Lube Flow Rig.R.P.M. いご 3777 Para. 11/1 1.7 1.7 1 Total 7.1 ١ Time :-; Test Time . 1 2 بنج **C-**2 Vals ★ REMARKS: TIME TYPE OF 165T W.P.I. NO. HSF 1755 4/6% PAGE NO. .. đ ORIGINAL PAUL

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SHEET OF Deg. Ulade Horiz. Angle OPERIORS Vib. MILS ENGINEER / Vert. MILS ENGINEERING LABORATORIES Temp. Shroud , , , LOG OF TEST Lube Oil . PART NO. 767500 Rev. A Supply アーチャーつド EE EE Clutch 222PT-31 Ţ * 7 大 1 PLAN OF TEST NO. PSIG 1 TORGUE AT MANDING INTO 1. 6. 1. 1. 1. 65 05.5 (475. 10. 11 SERVAL NO. Supply PSIG PSIC 1.01 5. 1 170 ; , ,,,, Hamilton Standard Lube Oil 25.03 G.E. QCSEE ACTUATOR ENDURANCE TEST Lube Flow Rig R.P.M. 371. Para `. ... Time Total : : / Time Test ÷ ' ₩ SINO ING NO. TIME W.P.I. NO.

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8 ENGINEER 27 / 1 / 1/1/ AL Deg. Blade . 10 **HSER 7002** 111-See . Horiz. RILS Vib. OPERATORS / Vert. MILS ENGINEERING LABORATORIES Temp. Shroud 11.11 LOG OF TEST #4 Lube Oil PAIT NO. 767500 - (• Rev. A Supply #3 EHV 1.7.1 222PT-31 Clutch 2 21 <u>*</u> 1.11 1000 711. 113 127 11114 111 ? 211. 7/// 111 PLAN OF TEST NO. PSIG 6/8 0i1 127V. 14.03 11 11 SERAL NO. _ PSICE EHV Supply 21.1. 3.7.5 ~~~ 35/25 1111 17.11. ٠,٨٠, ことに 1:1 1777 1.00 11.11 6.4 / PSIG 3.9 いつ 1 G.E. UCSEE ACTUATOR ENDURANCE TEST 17:3 Hamilton Standard Lube Flow SC 213 Rig R.P.M. ルニ: Para 11:17 77.677 Time Total -' } Test Time **C-**2 **★** 2175 TYPE OF TEST TIME MSF 1758 4/67 REMARKS. W.P.I. NO. PAUL NO ORIGINAL PAGE OF POOR QUALITY 185

Hamilton Standard --12/1 123 1/L

LOG OF TEST

ENGRATE C. L. L. Blade 13: رم دة - H.O OPERATORS L. SPERT Horiz. • DATE / 2. -Wils Vib. • • ENGINEERING LABORATORIES Shroud Temp. 5/2 1: 19 PART NO. 767500 e e e 57 1/2 1 Rev. A Supply 13 EPV 1.0 120 1501 Jutch 222PT-31 = 1.23 Star Sycar 124 25 7 10 Cycle 124 35 24 Cycle 124 C.K.L.C # 426 1.21 6 2 1.27 Cy01 C 8432 1416 A 1127 C/210 E/12 1. 1e 7435 シバル 1 1st spale PLAN OF TEST NO. PSIG 1. Jet 1 12/5 950 SERIAL NO. Supply (1) PSIC. . Mark So F. W. STORE 32.1 NEC 1700 3 1100 そろ 盖 3500 PSIG Lube 0il 66 39 66 G.E. GCSEE ACTUATOR ENDURANCE TEST . 하 왕 ام. ند. . 85 . لاگ Rig R.P.M. 2002 3/1/28 0% WINDSOR LOCKS, COMMECTICUT . U.S.A. Para 1.7 1.7 ; Time Total 01 13 31 . Test Time ı 12,12 6-7 **4** 22.25 1. W. TIME TIPE OF TEST W L NO • 2 0

REPORT NO. **HSER 7002** cycle # 429 tel. 17:30 (~ Y REMARKS.

Hamilton Standard HS7 1758 4/67

see 55 00 DATE 12-15-75 ENGINEERING LABORATORIES **106 OF TEST**

Deg. Horiz, I fingle ENGINEER D. 17:17 5 1.1.1 ورا 110 CHELATORS S. HELLT 4 3 'n MILS 7 • • Vert. • • . Temp. #5 Shroud 5 76 73 76 18 PART NO. __762500 Lube 0i1 15 ^ \ 7. * 20 20 Supply EIM 127 200 'n, 129 26 £, ~ Clutch 222PT-31 20 'n 95 1.63 7 6 NB Cy 112 41/8 350 LND Cycle "156 NO CY 16 " 411 AL CYCIC #138 Surrey Course Enn Cycle "13 The art IND CHOP "130 and cycle " 43. no cylic. #11 Icha in cyle 13 NO CHIEFUN tering 500 FAD Cycle # 131 TEKA JAhr E 10 Cy. 16 #153 140 cycle *433 HKT CYCTONIA THE COURT WITH " KI Cydre # 119 PLAN OF TEST NO. Supply 6/8 PSIG JAKT CY SERAL NO. 3500 18 START STEET PSICE 3500 3500 PSIG 00 Lube Oil 66 66 6.E. QCSEE ACTUATOR ENDURANCE TEST . 85 Lube Flow , 85 , 85 \$ \ . 53 Rig. R. P. M. 3068 3002 306 8 170C 31/08 Para. 1.7 11 4.7 11 Time 1.25 25. Total 51 12 1 ? 71/18 10 (1) **Test** Time ? 5 6-7 7 2000 ₹ 5125 17.00 1:03 15.51 0)/./ TIME TYPE OF TEST dit 1951 W.P. NO. 0101 0000 original.

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Deg. Angle Blade 001 10 - 7 U Wib. Horiz. • • MILS Vib. Vert. • Shroud Temp. 15 75 77 3 ノク 76 PART NO. 767500 35 60) (40) 6. 8 Ì. 2 Supply 13 13 133 3 .0 : Clutch 7 ن ۱ <u>د</u> **=** 96 Conductory 3415 Jul. 8 3 Con Cycle Butt 4. Cyc C 4. 184.73 143 Cre 1249 10.2 CY 10.958 10.2 CY 10.958 10.9 CY 10.958 5 1: 4756 2. Cy. A. 8450 the Cylinds " Cy 1. " W. "x1 . 4 15 0 1153 E514.31 19 C. 1 10 219 PACT CONTENSO PSIG 35:00 ch 13:25 1.1.1 1. 102 9<u>5</u> * C4 Supp Jy PSIG PSICE 35.0 3560 50,41 3500 Lube Oil 06 66 66 6 66 ري ح .85 age Se 185 <u>ئې</u> ,85 Rig R. P.M. 2700 2700 3.108 30.65 3068 Para. イン . ! 1.7 1.1 - Ime 6 135 Total Ċ 13 1 7 ; Time Test 27 Ç,I 4 5.0 7 26 0. 2115 4 225 1. 10 W.P.I. NO. ンング TIME 2115

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12. 27.00 Brade Z. Angle 111111 · ,• ENGINEER MILS Horiz. Υib. ` ; Vert. vib. _ ENGINEERING LABORATORIES Shroud Temp. 122 イイ 12 1/2 **#**2 LOG OF TEST PART NO 76.7550 S S Lube 0i1 11 1/2 **7**# ٠ بزر. Rev. A Supply ,,, 101 ?) ?) ' #3 EHV 32 Clutch 222PT-31 11 601 401 100 7 500 Cyle 462 Fire Cycle " 1:6 1947 Cycle 8465 Fino cyall "167 540 3411 mye3 tant byseaner No capte 165 mxt cyte ext 110 Cycl 6"170 TWO CYPIETES FAU Cydie 1/68 Ful Cy. 1e"/67 HARTCHING WILL our cytue and Ins Cycle Hel 4 END Cycle#111 Transportar MAN OF TEST NO. Supply (0)1 PSIG 1 4 SERIAL MO. PSIG PSICE 3500 3500 Hamilton Standard 3500 35,00 60 66 Lube Oil 10 66 chines lyex in SonsexN . 337 (U) (X) مدر G.E. GCSEE ACTUATOR ENDURANCE TEST Lube Flow 3 Rig R.P.M. 2018 2700 3068 31/28 Para 1.1 1.1 1.7 1.1 Totai Jime 31 31 ` ' ' **%** 1:1 ; Test Time 1/ 2 7,5 20 1230 27 6-7 7 133.45 UPETS # 11:50 21.15 2116 TYPE OF TEST W.P.I NO. TIME HSF 1758 4/67

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Hamilton Standard

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LOG OF TEST ENGINEERING LABORATORIES

BOSER L'A Linial Vib. Blade Horiz, Engle REPORT NO. OFFILTORS L **HSER 7002** KILS MICS DATE 1 Temp. Vert Shroud **‡**2 PART 1-0 _762500 Lute 0:1 Sev. A Cupply #3 FEE Clutch 7. . . . 222PT-31 = 7.56 13.77 100 14 William. 1821 8 1.1 PLAN OF TEST NO. PSIG 9 (3) Sypply 6/8 11.11 SERAL HO. 111 21:1 i PSIG PSICE 316. 1 N 111. F Juli コンプラ 1.71.11 10.00 7777 IUNK • 1770 **** 74/ Lube Oi J 101 M.P.I. NO. 1.E. OCCEE ACTUATOR ENDIPMICE TEST 77.4 1/4 Lube Flo¥ Rig R. P. S. 01.11 000 Para. Time Time ĸ,t 6-7 ر ا ا 4 2 5 PAGE NO. 191

13965 Blade Angle Deg. 60 Horiz. WILS Vib. OPERATORS ENCINEER WILS Vib. Vert. DATE 4 ENGINEERING LABORATORIES #5 Shroud Temp. **106 OF TEST** PART NO 767500 Lube 0i1 Rev. A Supply #3 EH Clutch 222PT-31 = 100, 205 4.0 CHOS # 500 1.1. PLAN OF TEST NO. 12.81 NO CHEST 11/1/2010 11 71 ; 4-7-7-PSIG 711.777 1000 THE FACT (... 17.5 でん ノンド 6/8 011 ارس (دیر Hamilton Standard SERIAL NO. PSICE EHV Supply 37.70 175.4 11 70 PSIG 1 Lube 0i1 66 000 G.E. UCSEE ACTUATOR ENDURANCE TEST · F 15.7 1355. Lube Flow 4 377 11/1/ Rig R.P.M. 1.40 Para 1:1 1:1 1.7 11 Total Time ; 1 13 ġ, 14. 1 , S _1 Test ime ١. ्रा ì . `, 3 A SINO 1100 シャンナ 0 0 5 3 3 1 192 192 ٠, TIME 19/1 SZ 1 52 V.P.L.NO. FALLES : ~ PAGE NO

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ENGINEER ELL COLLA W. 45 1/46 1/2 HICK IN L. **HSER 7002** ENGINEERING LABORATORIES 1150 LOG OF TEST 117.5 ٠ ئ 107.7 1150 "amilton Standard 7.07 تعودت O CAR 220 True Fran TOM Com Coc Trais miss; 15 77.5 77.5 77.6 32.25 32.36 11.68 6 50.1 25.64 11.1 Pec 15th 26.2 3220 35. " TIME PIG NO TYPE OF TEST REMARKS. HSF 1758 4767 PAGE NO 193

DATE 14 30 75
ENCINER 7 (E 15 KILLA)
OPERATORS (L.) HUMILLES 0-REPORT NO. **HSER 7002** r£1 ENGINEERING LABORATORIES LOG OF TEST 222 PT. 31 REV A PART NO. HSER 7002 US CENT ALICIE MAN OF TEST NO. 17.6 18.4 55.4 77.2 37.6 9.3.6 5.55 8.7 SERIAL MO. Wamilton Standard 124.2 .. ¥¥ - 49. 1:59-~ CU. 7 ~ ... 151 CO3 AC2B IVOT -.87 1.07-2.0 -1.77 108 721 1.35 2 9.. W.P.I. NO. 23.644 #11. ٠. ا シル・ - 75 -15 11.-- 10.3 C 1, -3 1-3 usa 40 34. 4 4 275 TIME HSF 1758 4/67

REPORT NO. Er X 100 ENGINEER 12 LC 1 ... 114 ... 14 ... OFFRATORS A.C. 1. ... 5 480 **HSER 7002** Liter. VAI 93 . 31 -1 120 777 177 120 7 15.0 120 176 . 7. -# W. A. M ~ ქ 4 ن 4 1 17 'n 3 LOG OF TEST ENGINEERING LABORATORIES 7. M. W. L ٦ 4 ~ 4 <u>_</u> 7 SIV C1.11 Ve 1500 57.43 11.7.6 1. 11. 11 <u>.</u>5. 1063 25 % 236 1111 0.17 1277222 PAM NO. 11.4 1010 Dut Trick Section 1.23 ٠ : : : : : PLAN OF TEST NO. SERIAL NO. 0.11 19.0 11.0 E 4100 3000 300 3.7.0 Secret 200 0 2 2 J.H. Psic Cir 00 11. 40 66 00 0 45.3 2365 15:3 The horse \$720 · \$508 XC 11 012 Pairen Acuracy 3700 1,000 () Y 1000 3.700 30 6 5 270 4 -710 011 11 110 711 113 7.7. 11.11 11.11 11.11 7 13 ORIGINAL QUALITY 3 UMIS ★ TYPE OF TEST W.P.I. NO. HSF 1758 4/67 <u>-</u> PAGE NO. 195

106 OF TEST

ENGINEERING LABORATORIES

in 12 or REPORT NO. ENGINEER D. L. 11. 11. 500 777.77 **HSER 7002** 2 יישיני ו 16. C 146. -18 +2+ 12.0 OPERATORS E 3 ~ V. 2.4 **c/** PART NO. // 3 1. C 1: "1 100.5 11. 111 116.9 1.00.1 111.7 22211.31 Surery Coll W.J. 13:0 1004 4 196. 4 £ 18.7 1.301 ...) PLAN OF TEST NO. 3400 16C 18.0 Secure 016 3/2/16 SERIAL NO. "Hamilton Standard Hamilton Standard A. 37.10 17.5°C 1:16 3,11 3.33 BLADE AKELE 5 1.90 011 1.5.1 31ch | 95ch 99 NW. 26 435.3. 22 20cs | 8598 29 99 No. 3. 52 1.1. 31.00 NSGB 92 Avere Terrien Accumber AC 3-13 1.1:15 \$ 56.15 C 5Cb FICH Ticcour 4. 5.41 3.10E 7.4. 30.60 44. 6. 31.65 4 5 4 3408 Tiera 1111 Trat True 100 1,7 50.4 25 35 3 111 では \\ - \| 400 ▲ SIRO REMARKS 31 12. 1 TIME TYPE OF TEST W.P.I. NO. HSF 1758 4767 9 9 196 PAGE NO

ENGINEER D.L. 15115144 TSIG PSIG PSIG " 1 TAV LEILE WALL VIB. VIB. VIB. COLL COLL TEON. YEAR MELL ANCE REPORT NO. OPERATORS F. SAUNDIFLS 31. 25 0:2--100 0 - 2/5 **HSER 7002** -3.0 150 r 2.0 67.0 O ENGINEERING LABORATORIES FLAN OF TEST NO. 222 [1-31 1. 1. A. SENAL NO. 743 500 4.41 2710 - 8496 22 3400 16.7 1125 12.6 7.44 18.3 -1 28.4 .2 LOG OF TEST 1010 98.1 1024 4.7 2 3 6 8 8 8 72 375 0 18 5 100,0 89.5 88.6 TO TOUCH OF WHITE PHINE ON PRINTER 3450 18.5 3726 18.5 2750 185 Wamilton Standard TAGA RIM FLUW SIL SUCIES 37.50 3750 3750 3750 5750 3/10 3/50 ICT (C) 103. BY ACCUSEDY 66 215 39 2665 85618 22 1 68 8:5.3 99 31.15 スノン 300 3000 3025 ようし 35.6) C S 3068 \mathfrak{S} ORIGINAL OF FOCK - PAGE IS QUALITY TIME VriTS ★ TYPE CH TEST HSF 1738 4767 ¥.P.I. NO. REMARKS. PAGE NO.

LOG OF TEST 1-lamilton Standard order of the source of t HSF 1758 4/67

ENGINEER D. J. L. L. L. L. L. OPERATORS E. Sayuple REPORT NO. 1555 1662 ARICHARD **HSER 7002** 77.11 Vin. Viti. Stant 411. -100 22 Q DATE / 7 ENGINEERING LABORATORIES FART NO. 222 553 D 17.7 4112 4.7.6 13/1/ 138.5 10.00 1. MAN OF TEST NO. 322 77.31 Lurica Sucrey 133 6 1631 FIRV 1CS C 101.7 PEN TARE 18.5 5.17 15.5 Psie 011. 18.5 SERAL NO. Y CC' Y 7516 127 Pic INCINER 7516 LVBr St - E'465 /461E lostrical Accordacy 016 99 55 51353 2700 1800 52 66 23.4. . S. Scu | 99 22 3923 21.00 1.550 15 15.66 79 17.06 - 4 spul Tie LUSE 120 . 1 . cm 15:17 2/40 7.70E 1068 7 6.80 30.65 P. C. S. 7.77 1, 52. Time from 3, 1 71 <u>ر</u> ن REGARD TASE OF REST ON O. UNITS # TIME ن -١.

N. C. KI **HSER 7002** i, OPERATORS 2. LOG OF TEST ENGINEERING LABORATORIES PART NO. PLAN OF TEST NO. TO SERIAL NO. OF POOR PA. 1973 7...... 133 The President of Flow 129 135 134 133 134 27 33 20 34.25 61.75 201 10 HIL-L- 1808 G 0/2 95 (1) 23 Š W.P.I. NO. HSF 1758 4.67 PAGE NO 199

HSER 7002 DATE 12 - 5 - 7/ OPERATORS TO A STATE OF ENGINEERING LABORATORIES 021 **106 OF TEST** 772Dr - 74 PART NO. 100 50.11 ALAN OF TEST NO. SERAL NO. Hamilton Standard tyo roi exc 16/45 5.667 100 Lung 50 105 1.817 LP3 7.812 -832 -18,7137.7 2.30 J.M. 45 4019 208 15 1.113 118 -m.5 115.5 2.143 2.110 15 1.4.82 1.460 53 - AIX 1000 35 TB88 +404 + 743 +758 2.564 Tuck 8 18.85. 37 1.1.36 1.450 127 CH 1 C: xx 1.148 1.12 T. ... TSM 72 Pt I · · 2 \$ 0 4 2275 TIME TYPE OF TEST HSF 1758 4/67 S N N PAGE NO

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HSER 7002 ENGINEERING LABORATORIES LOG OF TEST PLAN OF TEST NO. Hamilton Standard SEBAL NO. F. C. J. C. S. C. 137 12 200 230 2002 CANET 10 70.8 7.083 15/10/20 553 433 233 231 14.4 4 12 4 26 12.30 TIPE OF ITST W P.I. NO. HSF 1758 4/67 PAGE NO 201

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ENGINEERING LABORATORIES LOG OF TEST

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ENCHATER (" () L. LICTORY OPERATORS Blade Angle 1.0 O **HSER 7002** 7 DATE 1. VEET .1 2 ~ Horiz. MILS Vib. 3 4 7 Vert. Vib. 4 #5 Shroud Temp. 25.5 . 5.5.3 65.3 3 59.7 10. PART NO. 767500 Lube 107.6 Ož. 1 · · · ن د : 176 115 Rev. A Supply . #3 121 % 1150 1. 1. 1.27 1 222PT-38 Clutch 1.001 1111 11/2 11 10 2.00 11.11 1.1.1 のケイ /(C:1 11:11 1000 in The 7.5. 1 יין פי Ath to 1, ... • PLAN JF TEST NO. PS 1G かから 6/8 013 ٠٠٠٠) SERIAL NO. ٧/ 41 5 ين ر 6 EHV Supply PSIC2 17:0 , 3CC D. A 720 8 3,00 verten depun 8 3 A. C. 1. 1. 1. 1. 1. 1. PSIG 76 -DUN Lube Oil X N X / **SC** S S ň 77 8×4 30 17 X. 4. G.E. CCSEE ACTUATOR ENDURANCE TEST Lube Flow 21/00 10 86 11/3 X 4-7 5135.3 1586.13 2 SC 15 ر د . Rig. R. P. M. 3 111 アンジ 62,12 go 1700 D)cr 100 ž 7/10 ~1 () Para 1.1 <u>ن</u> : ١ ري `` ب : ? ر 12 Time Total 100 Ş ١ `, 1 ~) :--1 Time **Fest** 54.1 ۶*۲* ۲ **C-**5 -4 21 · · · :XE Tra Co TEST W.P.I NO. 5:10 Ç . . · _ 0/4 15 07.03 PAGE L OF T PAGE NO. 20:

REPORT NO. 8 Angle Blade 177 Deg. **HSER 7002** SHEET ... Horiz. 11. MILS OPERATORS Z ENGINEER Vib. Vert. MILS OF POOR QUAL TY ENGINEERING LABORATORIES モルド Temp. Shroud : 18 #5 LOG OF TEST 0.7.7 PART NO. 767500 Lube 0i1 ₽# Rev. Supply #3 EHV 3 222PT-3B Clutch -V 2 1. S. S. S. 10/10 Lun 12 Games 1 1111.1 119 1... J. / Crewits 200 11.00.1.00 21/11/11 LIM COME ST. 1.117 1,132.13 150A J. France 1. 1. 1 PLAN OF TEST NO. PSIG CIXI 6/8 0:1 ď SERIAL NO. EHV Supply PSIC 240 1 Ţ 'Hamilton Standard 3/100 40 CHES PSIG **ዕ** Lube Oil 3 4 QCSEE ACTUATOR ENDURANCE TEST 2 Lube Flow 7757 LIFP L Rig R. P.M. 170 XIK Derit 371.5 110 311 2017 1117 Para. = Total Time الم <u>-</u>' ات S S Test Time 21 G.E. 2 6-7 121 7. 4. 5. UNITS ₩ 1 ...1 REMARKS TIME TYPE OF TEST W.P.I. NO. HSF 1758 4/67 - ; <u>;</u>-PAGE NO. 203

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HSER 7002

APPENDIX E

TEST CHRONOLOGY

ORIGINAL PAGE I.
OF POOR QUALITY

TEST CHRONOLOGY

11-4-75	Started installation of actuator on rig.
11-5-75 & 11-6-75	Continued installation of actuator on rig.
11-7-75	Checked torque required to change pitch at wave generator. Appears high.
11-8-75	Removed actuator from disc. Retorqued blade retaining nuts to approximately equal twisting torques for each blade. Reinstalled actuator. Torque at wave generator now == 150 inch pounds. Continued assembly.
11-10-75	Completed installation. Ran LVDT calibrations and lubrication flow check. Found flex shaft leaks.
11-11-75	Sealed up leaks in flex shaft. Started running travel limit switch tests. After several stops, actuator would not move. Partial disassembly revealed no back output shaft fractured.
11-12-75 thru 11-19-75	Removed actuator from rig for inspection, and reinstalled on rig with repaired (welded) no back output shaft. Torque at wave generator to change pitch 140 inch pounds and at manual input 45 inch pounds.
11-20-75	Attempts to run stopped by oil leaks at rig drive flange and water leaks at rig clutch. Removed actuator and disc from rig. Reworked oil drain holes in rig drive flange, and revised drive flange seal and clutch

cooling water plumbing.

TEST CHRONOLOGY (continued)

11-21-75	Reinstalled disc and actuator. Oil leaks o.k., but clutch cooling water marginal. Could not get fan speed above 2750 rpm with blade angle as circuit breakers trip.
11-22-75	Exploring rig power problem.
11-24-75 thru 11-26-75	Fabricated shroud to enclose blades. Check run shows power problem solved.
11-28-75	Ran performance tests and 15 flight cycles with reduced pitch change rate.
11-29-75	Ran 31 flight cycles (total 46) and max pitch change rate tests.
12-1-75	Started disassembly for inspection. Found no-back output shaft fractured.
12-2-75 thru 12-5-75	Continued inspection and reassembly with new output shaft. Hydraulic motor bevel gear mesh pattern high. Reshimmed closer into mesh. Trunnion roller pattern on cam is high on one side of track, low on other. Torque to change pitch at manual input \$\iff 40\$ inch pounds.
12-6-75	Ran 14 flight cycles (total 60). Inspected no-back hardware, o.k.
12-8-75	Ran 71 flight cycles (total 131).
12-9-75 212	Ran 24 flight cycles (total 155). Inspected no-back hardware, o.k Ran 33 flight cycles (total 188).

TEST CHRONOLOGY (continued)

12-10-75	Ran 52 flight cycles (total 240). Torque to change pitch at manual input now 35-40 inch pounds.
12-15-75	Ran part of frequency response testing and 60 flight cycles (total 300).
12-16-75	Inspected no-back hardware, o.k. Reworked no-back output shaft to reduce stress concentrations and shot peened it in web area.
12-17-75	Reassembled and ran 70 flight cycles (total 370).
12-18-75	Ran 115 flight cycles (total 485).
12-19-75	Ran 20 flight cycles (total 505). Disassembled no-back for inspection. Output shaft fractured in one web.
12-29-75	Reassembled and ran rotating frequency response test.
12-30-75	Ran positioning accuracy and minimum blade angle change tests.
12-31-75	Re-ran travel limit switch, performance, positioning accuracy, and static frequency response tests.
1-2-76 thru 1-7-76	Complete disassembly and inspection including magnaflux and zyglo. Hydraulic motor bevel gear mesh showed pitting and scoring. One hydraulic motor damaged during disassembly. No-back output shaft was fractured. All other magnaflux and zyglo o.k.

TEST CHRONOLOGY (continued)

1-8-76 thru 1-12-76	Reassembled beta regulator with one new motor, new bevel gear set, and reworked feedback shaft to provide positive bevel gear mesh lubrication.
1-13-76	Reassembled differential gear train. Beta regulator at inspection for installation dimension check.
1-14-76	Ran lubrication flow versus pressure test on beta regulator.
1-15-76	Beta regulator to shipping. Reassembled no-back with new spring and new output shaft.
1-16-76	Fitting spring to drum.
1-19-76	Started actuator installation in rig. Rear housing fractured during assembly.
1-20-76 thru 1-30-76	Repair of rear housing. Ran static deflection tests on snubber.
2-2-76 thru	Reassembly on rig. Actuator has new no-back spring and output shaft, new flex shaft, and snubber. New flex shaft has slight leak.
2-5-76	Ran an LVDT calibration and maximum pitch change rate testing. Inspected no-back hardware, o.k.

TEST CHRONOLOGY) (continued)

2-6-76	Reassembled actuator and attempted to run flight cycles.
	Had trouble with controller and pumps which supply pitch change
	fluid. Ran 3 flight cycles.
2-7-76	Ran 12 flight cycles (total 15). Had trouble with pitch change fluid pumps.
2-9-76	Installed an accumulator in the pitch change fluid supply system.
2-10-76	Ran 45 flight cycles (total 60).
2-11-76	Attempted to run static frequency response test. No response at + 4 ma input to servo valve.
2-12-76	Disassembled regulator. Found one hydraulic motor (new one) has heavy wear on housing at drive gear face. Dimensional checks show nothing. Started rework necessary to obtain new motor.
2-13-76	Reassembled regulator and ran travel limit switch tests.
2-14-76	Installed new motor in regulator. Actuator responds to \pm 4 ma inputs to servo valve. Ran blade angle accuracy and performance tests.
2-16-76	Ran static frequency response test. Started disassembly.
2-17-76	Hardware inspection revealed that flex shaft had been overtorqued. All other hardware o.k.

APPENDIX F REFERENCES

HSER 7002

REFERENCE LIST

HSPC 74A14 QCSEE Variable Pitch Fan System Proposal

SP 08A74 QCSEE Variable Pitch Fan Pitch Change System

NASA CR-134852 Hamilton Standard Cam/Harmonic

Drive Variable Pitch Fan Actuation

System Detail Design Report

NASA CR-134873 QCSEE Ball Spline Pitch Change

Mechanism Design Report